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FINITE ELEMENT SOLUTION OF THE NONLINEAR
COUPLED NEUTRONIC-ENERGY EQUATIONS
FOR A FAST REACTOR FUEL CELL

Roy Edward Kasdorf

NAVAL POSTGRADUATE SCHOOL

Monterey, California



THESIS

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COUPLED NEUTRONIC-ENERGY EQUATIONS
FOR A FAST REACTOR FUEL CELL

by

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December 1976

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T176641

REPORT DOCUMENTATION PAGE		READ INSTRUCTIONS BEFORE COMPLETING FORM
1. REPORT NUMBER	2. GOVT ACCESSION NO.	3. RECIPIENT'S CATALOG NUMBER
4. TITLE (and Subtitle) Finite Element Solution of the Non-linear Coupled Neutronic-Energy Equations for a Fast Reactor Fuel Cell		5. TYPE OF REPORT & PERIOD COVERED Master's & Mechanical Engineer's Thesis; December 1976
7. AUTHOR(s) Roy Edward Kasdorf		6. PERFORMING ORG. REPORT NUMBER
9. PERFORMING ORGANIZATION NAME AND ADDRESS Naval Postgraduate School Monterey, California 93940		8. CONTRACT OR GRANT NUMBER(s)
11. CONTROLLING OFFICE NAME AND ADDRESS Naval Postgraduate School Monterey, California 93940		10. PROGRAM ELEMENT, PROJECT, TASK AREA & WORK UNIT NUMBERS
14. MONITORING AGENCY NAME & ADDRESS (if different from Controlling Office) Naval Postgraduate School Monterey, California 93940		12. REPORT DATE December 1976
		13. NUMBER OF PAGES 155
		15. SECURITY CLASS. (of this report) Unclassified
		15a. DECLASSIFICATION/DOWNGRADING SCHEDULE
16. DISTRIBUTION STATEMENT (of this Report) Approved for public release; distribution unlimited.		
17. DISTRIBUTION STATEMENT (of the abstract entered in Block 20, if different from Report)		
18. SUPPLEMENTARY NOTES		
19. KEY WORDS (Continue on reverse side if necessary and identify by block number)		
20. ABSTRACT (Continue on reverse side if necessary and identify by block number) A transient overpower (TOP) accident in a Liquid Metal Fast Breeder Reactor (LMFBR) is considered. The analysis is formulated to model the dynamic response of the reactor fuel subassembly during the initial period of the postulated overpower transient. An equivalent cylindrical cell is used to model the fuel subassembly. The governing neutronic and heat transport equations for each region (fuel, clad, and		

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Finite Element Solution of the Nonlinear
Coupled Neutronic-Energy Equations
for a Fast Reactor Fuel Cell

by

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Submitted in partial fulfillment of the
requirements for the degrees of

MASTER OF SCIENCE IN MECHANICAL ENGINEERING

and

MECHANICAL ENGINEER

from the
NAVAL POSTGRADUATE SCHOOL
December 1976

ABSTRACT

A transient overpower (TOP) accident in a Liquid Metal Fast Breeder Reactor (LMFBR) is considered. The analysis is formulated to model the dynamic response of the reactor fuel subassembly during the initial period of the postulated overpower transient. An equivalent cylindrical cell is used to model the fuel subassembly. The governing neutronic and heat transport equations for each region (fuel, clad, and coolant) of the equivalent cylindrical cell are developed. Nuclear Doppler broadening feedback is included in the dynamic model making the coupled equations non-linear. The resulting non-linear partial differential field equations are transformed into a system of ordinary differential equations by the finite element method. An isoparametric, quadratic, rectangular element is used for the discretization of the spatial domain. When using the finite element method, large system matrices may result. To facilitate solution of these large systems, an optimum compacting scheme is utilized. The implicit Gear's method is used for the solution of the system of ordinary differential equations. The results for a sample problem are presented.

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LIST OF SYMBOLS AND NOTATION

A. NOTATION

$\langle \quad \rangle$	Row vector
$\{ \quad \}$	Column vector
$[\quad]$	Square matrix or indicates a reference
$[\quad]^{-1}$	Inverse of a square matrix
∇	Del operator
$\frac{\partial}{\partial x}$	Partial derivative with respect to x
Δx	Change in x
\int_v	Volume integral
\int_x	Integration with respect to x
$\det [x]$	Determinant of x

B. SYMBOLS

b	Nuclear Doppler constant
C	Concentration of delayed neutron precursors
C_p	Specific heat [cal/gm °C]
$D(\underline{r})$	Neutron diffusion coefficient [cm]
e	Nuclear energy released per fission [cal/fission]
h	Heat transfer coefficient [cal/cm ² sec °C]
J	Jacobian matrix
$J(\underline{r}, t)$	Neutron current [$\frac{\text{neutrons}}{\text{cm}^2 \text{ sec}}$]
k_∞	Infinite multiplication factor
K_D	Doppler constant
$k(\underline{r})$	Thermal conductivity [cal/cm sec °C]
LMFBR	Liquid Metal Fast Breeder Reactor

LOCA	Loss of Coolant Accident	
N	Shape function	
n	Number of delay neutron groups	
$n(\underline{r},t)$	Neutron density	$[\frac{\text{neutrons}}{\text{cm}^3}]$
N,x	Derivative of N with respect to x	
$\dot{q}(\underline{r},t)$	Nuclear generation	$[\text{cal}/\text{cm}^3\text{sec}]$
R	Residual	
\underline{r}	Spatial coordinate	$[\text{cm}]$
r,z	Global coordinates	
$S(\underline{r},t)$	Neutron production	$[\frac{\text{neutrons}}{\text{cm}^3\text{sec}}]$
T	Temperature	$[^\circ\text{C}]$
t	Time	$[\text{sec}]$
TOP	Transient Overpower	
v	Neutron velocity	$[\text{cm}/\text{sec}]$
V_{co}	Velocity of coolant flow	$[\text{cm}/\text{sec}]$
W	Weighting function	
β	Fraction of fission neutrons which appear as delayed neutrons	
$\phi(\underline{r},t)$	Neutron flux	$[\frac{\text{neutrons}}{\text{cm}^2\text{sec}}]$
λ	Decay constant of the delayed neutron precursors	$[1/\text{sec}]$
η,ξ	Local coordinates	
ν	Average number of neutrons released per fission	
ρ	Reactivity	
$\rho(\underline{r})$	Density	$[\text{gm}/\text{cm}^2]$
Σ	Neutron cross section	$[\text{cm}^{-1}]'$

C. SUBSCRIPTS

a	Absorption
c	Clad
co	Coolant
CR	Critical
D	Delayed, Doppler
f	Fission
F	Fuel
gap	Fuel-clad interface
i,j	Group, equation
P	Prompt
surf	Clad-coolant interface

D. SUPERSSCRIPTS

e	Element
e*	Adjacent element
o	At time zero
.	Derivative with respect to time

ACKNOWLEDGEMENTS

The author wishes to express his appreciation to Dr. Dong Nguyen and to Dr. David Salinas, Professors of Mechanical Engineering, for their advice and guidance throughout the course of this work.

The author is obligated to Dr. Richard Franke, Professor of Mathematics, for his assistance in implementing the integration procedure and for his assistance as second reader.

The author wishes to thank Dr. Gilles Cantin, Professor of Mechanical Engineering, for his invaluable discussions concerning the finite element method.

Finally, the author wishes to thank his wife, Gail, for her encouragement and understanding throughout the course of this study.

I. INTRODUCTION

As the world's fossil fuel resources are depleted, more emphasis is being placed on the breeder reactor as a potential means of solving the coming energy crisis. While the development of new energy sources is being pushed, equal effort is being given to the maintenance of an environmentally clean world. To this end, the safety of breeder reactors is receiving a considerable amount of attention before assuming that the breeder reactor is the answer to the energy problem.

The Liquid Metal Fast Breeder Reactor (LMFBR) appears to be one of the most promising breeder reactors. Most engineers will concede there is little probability of a nuclear explosion occurring in the operation of a nuclear reactor. Of major concern to engineers is the loss of coolant accident (LOCA) and the transient overpower accident (TOP). The present analysis is concerned with a TOP accident in a LMFBR. The analysis is formulated to model the dynamic response of the reactor fuel subassembly during the initial period of the postulated overpower transient. The primary consideration is given to the early response of this fuel subassembly to various conditions of disturbances. The phenomenon which occurs after core disassembly (i.e., clad melting) is not the concern of this analysis. Only the time prior to clad melting is being considered.

No consideration is given here as to how the overpower transient occurs or to why the safety features of the reactor did not operate properly. It is postulated that the accident has occurred. In this analysis, the TOP accident is created by either a step increase in reactivity, a ramp increase in reactivity, or a combination of both.

An inherent safety feature of most reactors, nuclear Doppler broadening feedback, is included in the dynamic model of the fuel subassembly. The Doppler feedback acts to reduce the effect of the excursion. Consideration of this feedback creates a non-linear system model which is described by a non-linear, initial-boundary-value problem.

The conventional method of solution uses the standard point kinetics formulation. Recent studies have pointed out a non-negligible error in this model [1], particularly with asymmetric disturbances [2], or space-dependent feedback [3]. In Ref. [4], a somewhat novel approach of using the finite element method (FEM) for the space-time dependent solution of the reactor dynamics problem was demonstrated. The FEM is effective in handling these asymmetric disturbances and space-dependent feedbacks. Therefore, the finite element method was used so that the spatial effects on the postulated problem may be studied further.

The purpose of this work was to demonstrate further the applicability of the FEM to the non-linear reactor dynamics problem as well as to investigate the dynamic response of the reactor fuel subassembly. The analysis required a novel

approach to handle the gap conductances present at the interfaces of the equivalent cell model of the fuel subassembly; this will be clarified in the analysis.

II. DESCRIPTION OF PROBLEM

A. PHYSICAL SYSTEM

The typical Liquid Metal Fast Breeder Reactor (LMFBR) core consists of many hexagonal modules, each containing several hundred fuel pins. For this analysis, an equivalent cylindrical cell is used to model the fuel subassembly; see Figure 1. The use of equivalent cells as models for larger systems has been common practice in nuclear analysis (i.e., the well known Wigner-Sietz method). In using an equivalent cell, the actual shape of the reactor core is not important, and the analysis is applicable to any reactor which has the same equivalent cell.

The equivalent cell considered in this analysis, Figure 1, is fueled with enriched uranium dioxide, has a stainless steel cladding, and has liquid sodium for a coolant. The dimensions used are

$$a = 0.254 \text{ cm},$$

$$b = 0.292 \text{ cm},$$

$$c = 0.365 \text{ cm},$$

and $H = 33.0 \text{ cm}.$

The gap between the fuel and cladding is very small and, in fact, may be nonexistent as in bonded fuels. The dimension of this gap has been assumed negligible. The height, H , of the fuel rod is shorter than many proposed systems (Fast Flux Testing Facility and Clinch River Breeder Reactor). However,

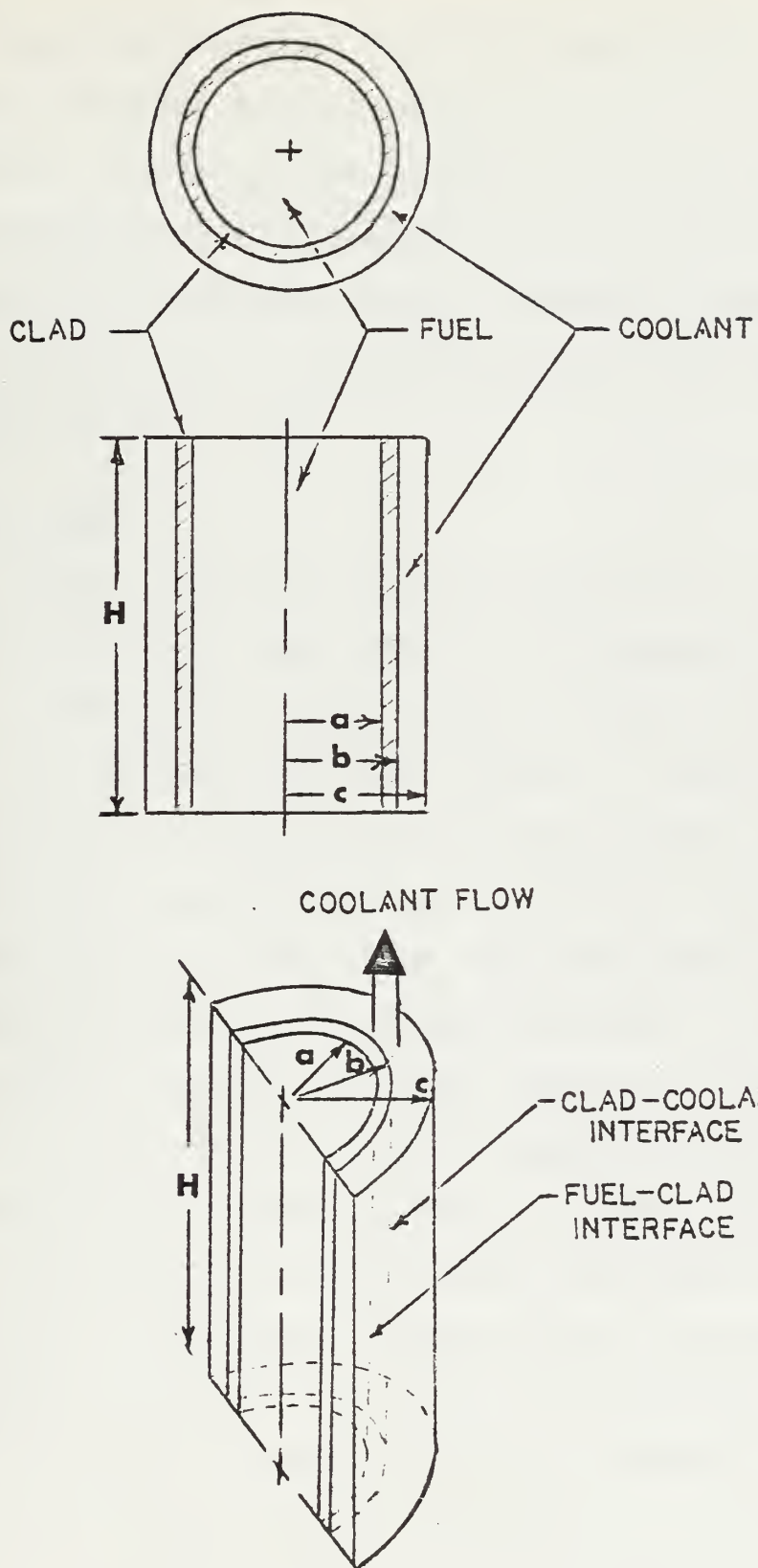


Figure 1. Equivalent Cylindrical Cell

to facilitate the numerical solution a smaller rod was used. The dynamic behavior prior to fuel pin failure for this system should be similar to the behavior of larger systems.

The treatment of this problem in three dimensions would be prohibitive in computer usage. Therefore, azimuthal symmetry is assumed, and the problem becomes a two-dimensional cylindrical (r,z) problem.

B. SYSTEM MODEL

The analysis considers the monoenergetic neutron diffusion approximation to model the transient neutron transport problem. A simple conduction-convection heat transfer model is used for the energy transport problem. The temperatures in the model are directly coupled to the neutron flux through the nuclear heat generation within the fuel. The neutron population is in turn coupled to the temperature through any of a number of reactivity feedback mechanisms. The nuclear Doppler effect is perhaps the most important of these mechanisms since it provides a negative temperature coefficient which increases the inherent stability of the reactor. Prior to core disassembly and fuel melting, the nuclear Doppler effect is the most dominant feedback and is, therefore, the only feedback mechanism considered in this analysis.

For irradiated, mixed-oxide fuels, a phenomenon of fuel restructuring has been commonly observed. This restructuring, essentially a change of phase of the fuel material, presents a unique heat transfer problem particularly during transient conditions. The problem has not been fully characterized and

is beyond the scope of this analysis. The fuel was, therefore, assumed to be a homogeneous mixture of enriched uranium dioxide.

At the fuel-cladding interface, there exists a gap which produces a thermal resistance. This thermal resistance is one of the most significant deterrents to the energy transfer to the coolant. The interface may be in physical contact or an actual gap may exist. The prediction of the thermal resistance is extremely complicated and must take into consideration many parameters: initial dimensions, type of bond, fill gas composition, fuel restructuring, fuel swelling, prior fuel life cycle, to mention a few. Reference [5] documents a computer program which attempts to predict the gap conductance, h_{gap} . This treats the thermal resistance at the interface in the same manner as a convection heat transfer coefficient when considering convection heat transfer. Since it is not the objective of this analysis to predict the gap coefficient, a representative set of values for gap coefficient, as given in Ref [6], is used in this analysis. These values are assumed to remain static during the transient. The gap conductance profile actually varies with time and will have an effect upon the transient, as noted in Ref. [7]. The prediction of this variance was not considered important for this analysis; therefore, the static assumption was made.

An average convection heat transfer coefficient was used to determine the heat transfer from the cladding to the coolant. The value used was determined from an empirical formula given in Ref. [5] and repeated in Appendix C.

In this work, consideration is given to step and ramp increases in reactivity, although any reactivity transient may easily be considered. The step and ramp increases in reactivity probably represent the most realistic physical reactivity inputs in a reactor. Once the reactivity has been inserted, the transient overpower excursion begins. Unless the Doppler feedback can override the inserted reactivity, the excursion will continue until there is physical core disassembly.

C. NUMERICAL SOLUTION

The system of equations which models the proposed problem is a non-linear, initial-boundary-value problem. The conventional method of solution of the reactor dynamics is the point kinetics formulation. It was pointed out in Refs. [1], [2], [3], and [4], that there is a non-negligible error in this model, particularly under conditions of asymmetric disturbances or space-dependent feedback. Reference [4] demonstrates the somewhat novel approach of using the finite element method (FEM) to solve the space-time dependent reactor dynamics problem. As shown in Ref. [4], the FEM is quite effective in handling localized perturbations and space-dependent feedback. In this work, only uniform disturbances were considered; however, the feedback model was space-dependent. Therefore, the finite element method is used to solve the non-linear, coupled, space-time dependent neutronic and heat transport field equations. The solution technique results in a large computer storage requirement; therefore, an optimum compact storage scheme, Ref. [8], is utilized for storage of the discretized matrices.

Once the domain has been discretized by the FEM, the solution of the resulting ordinary differential equations was to be accomplished by the implicit Gear's method, Ref. [9].

III. MODEL DEVELOPMENT

A. NEUTRONIC ANALYSIS

In this section the governing field equations for the neutron population (flux) for each of the three regions (fuel, clad, and coolant) of the domain will be formulated. The monoenergetic, diffusion theory will be used.

Consider an arbitrary volume of material within a reactor. Applying the condition of conservation to the monoenergetic neutrons leads to the neutron equation of continuity [10]

$$\frac{\partial n(\underline{r},t)}{\partial t} = S(\underline{r},t) - \Sigma_a(\underline{r})\phi(\underline{r},t) - \text{div } J(\underline{r},t) \quad (1)$$

where

\underline{r} - spatial point

t - time

$n(\underline{r},t)$ - neutron density

$S(\underline{r},t)$ - neutron production

$\Sigma_a(\underline{r})$ - neutron absorption cross section

$\phi(\underline{r},t)$ - neutron flux

$J(\underline{r},t)$ - neutron current

The left-hand side of equation (1) represents the time rate of change of the neutron density which is related to flux, ϕ , by

$$n(\underline{r},t) = \frac{1}{V} \phi(\underline{r},t) \quad (2)$$

where

v - neutron velocity

On the right-hand side of equation (1), the first term is neutron production, the second term is a neutron loss through absorption, and the third term is a neutron loss through leakage from the control volume.

Using equation (2) and applying Fick's Law to the equation of continuity results in the classical neutron diffusion equation

$$\nabla \cdot (D(\underline{r}) \nabla \phi(\underline{r}, t)) - \Sigma_a(\underline{r}) \phi(\underline{r}, t) + S(\underline{r}, t) = \frac{1}{v} \frac{\partial \phi(\underline{r}, t)}{\partial t} \quad (3)$$

where $D(\underline{r})$ - neutron diffusion coefficient

The vector notation used here is intended to include only two dimensions (r, z) since azimuthal symmetry has been assumed.

Equation (3) is applicable to each of the three regions of the equivalent cylindrical cell. The subscripts F (fuel), c (cladding), and co (coolant), will be used to denote these regions.

1. Fuel Region

In applying equation (3) to the fuel region the material properties of the fuel must be used. Within the fuel the neutron source term is due to the nuclear fission process. During fission, neutrons are released as both prompt neutrons and delayed neutrons so that

$$S(\underline{r}, t) = S_p(\underline{r}, t) + S_D(\underline{r}, t) \quad (4)$$

where $S_p(\underline{r}, t)$ - prompt neutron source
 $S_D(\underline{r}, t)$ - delayed neutron source

The neutron sources are commonly represented as [10]

$$S_p = k_\infty \Sigma_{aF} \phi_F (1 - \beta) \quad (5)$$

and

$$S_D = \sum_{i=1}^n C_i \lambda_i \quad (6)$$

where

k_∞ - infinite multiplication factor

β - fraction of fission neutrons which appear as delayed neutrons

n - number of delayed neutron groups

C_i - concentration of delayed neutron precursors in the i th group

λ_i - decay constant of the delayed neutrons

The space and time variables, \underline{r} and t , will be dropped except where needed for clarification.

The concentration of delayed neutron precursors, C_i , is given by the following first order partial differential equation [10]

$$\frac{\partial C_i}{\partial t} = \beta_i k_\infty \Sigma_{aF} \phi_F - \lambda_i C_i \quad (7)$$

where

β_i - fraction of delayed neutrons which appear as delayed neutrons in the i th group



The solution of equation (7) is

$$C_i = \beta_i \int_0^t e^{-\lambda_i(t-t')} k_\infty(r, t') \Sigma_{aF} \phi_F dt' + C_i^0 e^{-\lambda_i t} \quad (8)$$

C_i^0 - concentration of delayed neutron precursors of the ith group at time zero

Reference [10] develops an expression for the initial concentration of delayed neutrons

$$C_i^0 = \beta_i k_\infty^0 \Sigma_{aF} \phi^0 / \lambda_i \quad (9)$$

where

k_∞^0 - initial finite multiplication factor

ϕ^0 - initial neutron flux

Combining equations (3), (4), (5), (6), (8), and (9), yields the governing equation for the fuel region

$$\begin{aligned} \nabla \cdot (D_F \nabla \phi_F) + \Sigma_{aF} \phi_F [k_\infty(1-\beta) - 1] \\ + \sum_{i=1}^n \lambda_i [\beta_i \int_0^t e^{-\lambda_i(t-t')} k_\infty(\underline{r}, t') \phi_F \Sigma_{aF} dt' \\ + \beta_i k_\infty^0 \Sigma_{aF} \phi_F / \lambda_i e^{-\lambda_i t}] = \frac{1}{v} \frac{\partial \phi_F}{\partial t} \end{aligned} \quad (10)$$

2. Cladding Region

The cladding separates the fuel and coolant and contains the fuel and the fission by-products. Equation (3) governs the neutron flux in the clad. Since the clad contains

no fissile material, the neutron source term is zero. The field equation is, then,

$$\nabla \cdot (D_c \nabla \phi_c) - \Sigma_{ac} \phi_c = \frac{1}{v} \frac{\partial \phi_c}{\partial t} \quad (11)$$

3. Coolant Region

The annular region around the cladding of the equivalent cell contains the coolant. As in the clad, there is no fissile material in the coolant and, therefore, no neutron source term. The equation governing the neutron flux in the coolant is

$$\nabla \cdot (D_{co} \nabla \phi_{co}) - \Sigma_{aco} \phi_{co} = \frac{1}{v} \frac{\partial \phi_{co}}{\partial t} \quad (12)$$

Equations (10), (11), and (12) are the one-velocity, diffusion approximation used to model the neutron transport problem.

4. Infinite Multiplication Factor

The infinite multiplication factor, k_∞ , may be expressed as the infinite multiplication factor at time zero (start of transient), k_∞^0 , plus the postulated reactivity insertion (such as a step or a ramp), ρ , minus the change in the Doppler reactivity feedback, $\Delta \rho_D$. Other feedback mechanisms are normally not as significant as the Doppler broadening feedback prior to fuel melting and have been neglected in this analysis. Therefore,

$$k_\infty = k_\infty^0 + \rho - \Delta \rho_D \quad (13)$$

For a fast reactor, k_{∞}° may be approximated as

$$k_{\infty}^{\circ} = \nu \frac{\Sigma_{fF}}{\Sigma_{aF}} \quad (14)$$

where

ν - average number of neutrons released
per fission

Σ_{fF} - fission cross section of the fuel

Σ_{aF} - absorption cross section of the fuel

The nuclear Doppler effect is a very important safety feature in a nuclear reactor. Nuclei in an atom are in continual motion due to their own thermal energy. As a result of this motion, even when monoenergetic neutrons interact with the atom, there appears to be a spread in the energy of the neutron - the Doppler effect. It can be shown that the cross section of a resonance becomes less in magnitude and wider as the motion of the nuclei increases [10]. As the temperature increases, the motion increases, and the shape of a resonance cross section broadens. This broadening increases the average cross section, thus, providing a negative temperature coefficient. This effect is shown in Figure 2. It is this nuclear Doppler broadening effect which provides one of the few inherent, reliable, negative reactivity feedbacks which slows an overpower transient and possibly stops a mild overpower transient.

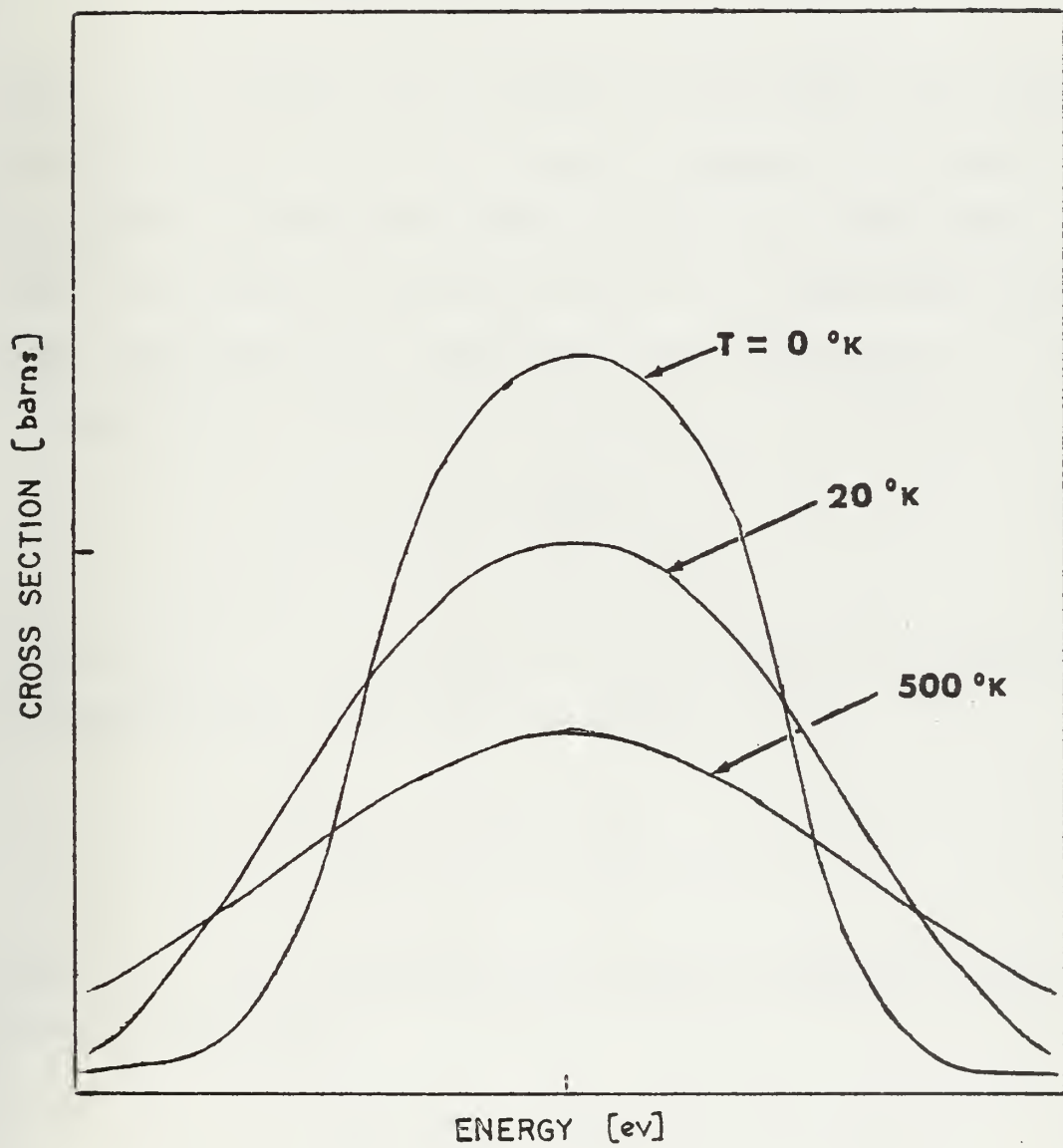


Figure 2. Doppler Broadening of a Resonance Peak

The Doppler reactivity change with respect to fuel temperature changes should be written as [6]

$$\frac{d\rho_D}{dT} = aT^{-3/2} + bT^{-1} + cT^{m-1} \quad (15)$$

where a , b , and c are parameters determined from experimental work and m is an integer. However, as noted in Ref. [6], a substantial amount of work has shown that $T \frac{d\rho_D}{dT}$ is very nearly constant over the temperature range under consideration. Therefore, the coefficients a and c have been set equal to zero, and b is defined as

$$b = K_D = T \frac{d\rho_D}{dT} \quad (16)$$

The constant, K_D , is commonly called the Doppler constant. Solving equation (16) for ρ_D yields

$$\rho_D = b \ln T_F + K \quad (17)$$

where K is an integration which may be obtained from initial conditions.

$$K = \rho_D^0 - b \ln T_F^0 \quad (18)$$

where

ρ_D^0 - Doppler effect at time zero

T_F^0 - fuel temperature at time zero

Substituting for K in equation (17) will give

$$\rho_D - \rho_D^0 = \Delta\rho_D = b \ln(T_F/T_F^0) \quad (19)$$

The infinite multiplication factor now becomes

$$k_\infty = k_\infty^0 + \rho - b \ln(T_F/T_F^0) \quad (20)$$

The effect of delayed neutrons is small compared to the prompt neutron effect; therefore, it may be assumed that the Doppler effect on delayed neutrons is insignificant to the overall problem. The k_∞ of equation (8) is, then, assumed to be k_∞^0 . With this assumption and equation (20), the neutron diffusion equation in the fuel, equation (10), may be rewritten as

$$\begin{aligned} \nabla \cdot (D_F \nabla \phi_F) + \Sigma_{aF} \phi_F [k_\infty^0(1-\beta) - 1 + \rho(1-\beta) - b(1-\beta) \ln(T_F/T_F^0)] \\ + \sum_{i=1}^n \lambda_i \{ \beta_i \Sigma_{aF} \int_0^t e^{-\lambda_i(t-t')} [k_\infty^0 + \rho] \phi_F dt' + C_i^0 e^{-\lambda_i t} \} = \frac{1}{v} \frac{\partial \phi_F}{\partial t} \end{aligned} \quad (21)$$

To facilitate the present analysis, the number of delayed neutron groups is taken as one averaged group. So that, λ_i and β_i become $\bar{\lambda}$ and $\bar{\beta}$, respectively. This approximation should have little effect on the problem under consideration.

5. Boundary Conditions

The neutron diffusion problem involves solution of the partial differential equations (11), (12), and (21), with the following boundary, interface, and initial conditions:

- 1) $\frac{\partial \phi_F}{\partial r} (0, z, t) = 0$
- 2) $\phi_F (a, z, t) = \phi_c (a, z, t)$
- 3) $D_F \frac{\partial \phi_F}{\partial r} (a, z, t) = D_c \frac{\partial \phi_c}{\partial r} (a, z, t)$
- 4) $\phi_c (b, z, t) = \phi_{c0} (b, z, t)$
- 5) $D_c \frac{\partial \phi_c}{\partial r} (b, z, t) = D_{c0} \frac{\partial \phi_{c0}}{\partial r} (b, z, t)$
- 6) $\frac{\partial \phi_{c0}}{\partial r} (c, z, t) = 0$
- 7) $\phi_F (r, \pm \frac{H}{2}, t) = \phi_c (r, \pm \frac{H}{2}, t) = \phi_{c0} (r, \pm \frac{H}{2}, t) = 0$
- 8) $\phi_F (\underline{r}, 0) = \phi_F^\circ (\underline{r})$
- 9) $\phi_c (\underline{r}, 0) = \phi_c^\circ (\underline{r})$
- 10) $\phi_{c0} (\underline{r}, 0) = \phi_{c0}^\circ (\underline{r})$

Boundary condition 1) results from the assumed azimuthal symmetry. Interface conditions 2), 3), 4), and 5) are continuity conditions of the flux. Boundary condition 6) results from the use of an equivalent cell and basically indicates there is an equal number of neutrons transferred in and out of the cell at the outer boundary. This should be valid unless the cell is located near the outer edge of the reactor. Boundary condition 7) is an assumption that the flux is zero at the axial boundaries of the cell. Initial conditions 8), 9), and 10) are the assumed initial distributions of the neutron flux.

B. HEAT TRANSFER ANALYSIS

In this section, the principle of conservation of energy will be used to formulate the governing field equations for the heat transport in each of the three regions. A simple heat conduction model with convection heat transfer to the coolant is used to model the heat transport problem. A gap conductance model is used to describe the heat transport across the gap at the fuel-clad interface.

1. Fuel Region

Conservation of energy within the fuel region yields the unsteady heat conduction equation with a generation term

$$\nabla \cdot (k_F(\underline{r}) \nabla T_F(\underline{r}, t)) + \dot{q}(\underline{r}, t) = \rho_F(\underline{r}) c_{pF}(\underline{r}) \frac{\partial T_F}{\partial t}(\underline{r}, t) \quad (22)$$

where

- $k_F(\underline{r})$ - thermal conductivity of the fuel
- $T_F(\underline{r}, t)$ - fuel temperature
- $\dot{q}(\underline{r}, t)$ - nuclear energy generation per unit volume
- $\rho_F(\underline{r})$ - fuel density
- $c_{pF}(\underline{r})$ - fuel specific heat

As in the neutronic analysis, the vector notation is intended to include only two dimensions, (r, z) . The \underline{r} and t will be dropped except where needed for clarification.

The nuclear generation term may be expressed as

$$\dot{q} = e \Sigma_{fF} \phi_F \quad (23)$$

where

e - nuclear energy released per fission

As can be seen, it is through the nuclear generation term that the temperature is directly coupled to the neutron flux. This coupling and the temperature dependent Doppler reactivity feedback combine to make the coupled problem non-linear.

Substituting equation (23) into equation (22) yields the governing thermal equation for the fuel

$$\nabla \cdot (k_F \nabla T_F) + e \Sigma_{fF} \phi_F = \rho_F C_{pF} \frac{\partial T_F}{\partial t} \quad (24)$$

2. Cladding Region

Conservation of energy within the clad will yield the heat conduction equation. With nuclear generation, a relatively small amount (~5%) of the energy will be released in the cladding and the coolant. However, in this analysis it is assumed that the total energy release is in the fuel region. This should not create any significant error. Using this assumption, the unsteady heat conduction equation for the cladding becomes

$$\nabla \cdot (k_c \nabla T_c) = \rho_c C_{pc} \frac{\partial T_c}{\partial t} \quad (25)$$

3. Coolant Region

Once again, conservation of energy will lead to the heat conduction equation plus an additional term which

takes into consideration the coolant flow. The governing equation is

$$\nabla \cdot (k_{co} \nabla T_{co}) - V_{co} \frac{\partial}{\partial z} (\rho_{co} C_{pco} T_{co}) = \rho_{co} C_{pco} \frac{\partial T_{co}}{\partial t} \quad (26)$$

where V_{co} - coolant flow velocity

Equations (24), (25), and (26) are the governing equations used to model the energy transport problem.

4. Interface Conditions

The interface between the fuel and the cladding may be an actual gap with a finite distance, or the surfaces may be in intermittent contact on a microscopic scale. To model the heat transfer across this interface, a gap heat transfer coefficient is introduced. The gap coefficient must take into consideration many items (e.g., radiation heat transfer across the gap, heat transfer by solid-to-solid contact, heat conduction across a gas filled gap). The prediction of this gap coefficient is extremely complicated and beyond the scope of this work. In Ref. [6], a set of values for H_{gap} is given and the axial variation of H_{gap} in this analysis is approximately the same. A cosine curve has been fitted to the sample data to determine the gap coefficient, see Figure 3. The heat flux across the fuel-clad interface is, then,

$$q = H_{gap} (T_F - T_C) \quad (27)$$

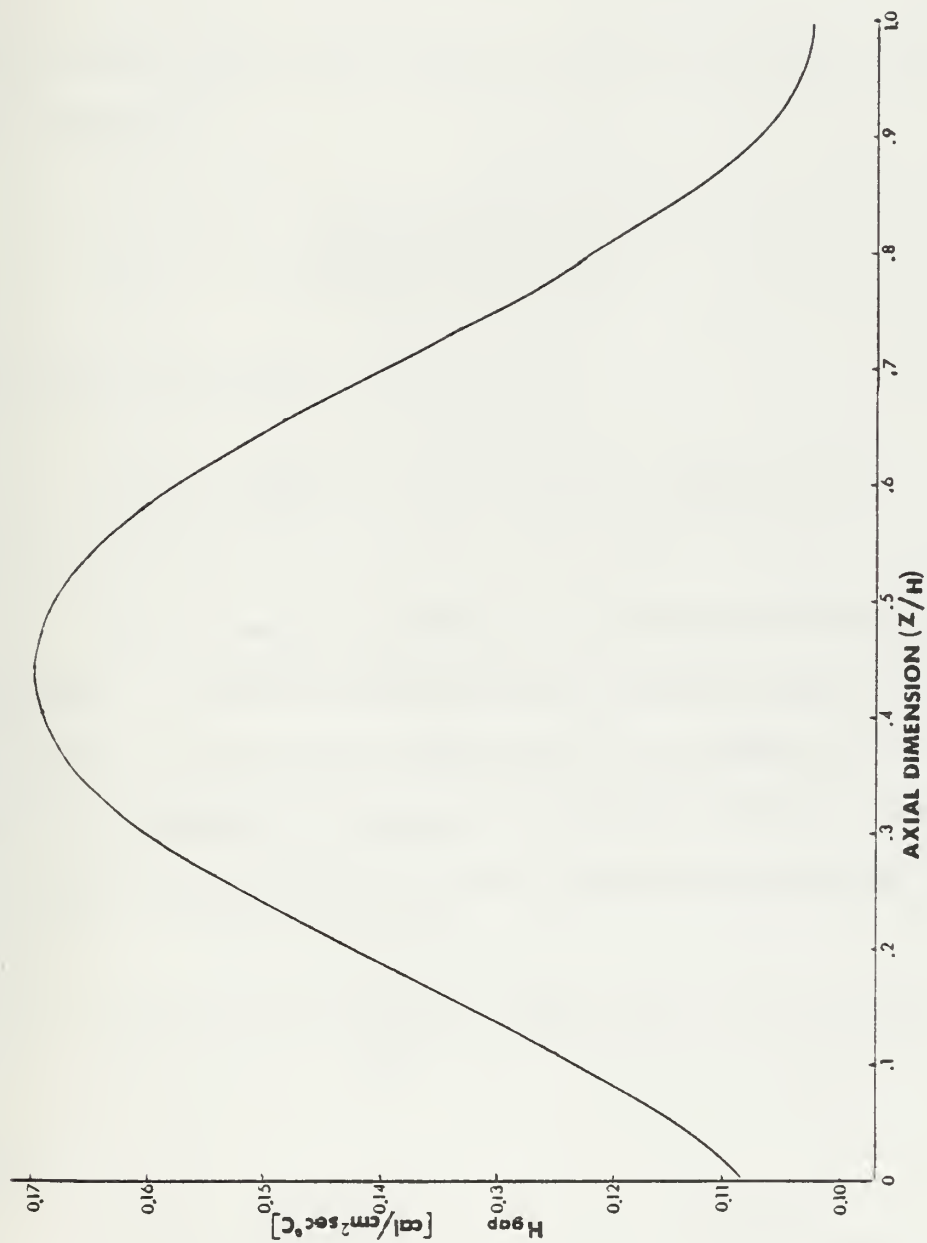


Figure 3. Gap Heat Transfer Coefficient

The heat conducted out of the fuel is governed by Fourier's equation and is equal to the heat transferred across the gap

$$q = -k_F \frac{\partial T_F}{\partial r} \quad (28)$$

Equating equations (27) and (28) gives the fuel-clad interface condition

$$T_F(a,z,t) + \frac{k_F(a,z)}{H_{\text{gap}}(z)} \frac{\partial T_F}{\partial r}(a,z,t) = T_C(a,z,t) \quad (29)$$

and from continuity

$$k_F(a,z) \frac{\partial T_F}{\partial r}(a,z,t) = k_C(a,z) \frac{\partial T_C}{\partial r}(a,z,t) \quad (29a)$$

As is common practice in convection heat transfer analysis, a coolant surface heat transfer coefficient, h_{surf} , is used to account for the thermal resistance at the clad-coolant interface. Similar to the fuel-clad interface analysis, the clad-coolant interface conditions may be determined

$$T_C(b,z,t) + \frac{k_C(b,z)}{h_{\text{surf}}} \frac{\partial T_C}{\partial r}(b,z,t) = T_{C0}(b,z,t) \quad (30)$$

and

$$k_C(b,z) \frac{\partial T_C}{\partial r}(b,z,t) = k_{C0}(b,z) \frac{\partial T_{C0}}{\partial r}(b,z,t) \quad (30a)$$

5. Boundary Conditions

The boundary and initial conditions for the heat transport problem are:

- 1) $\frac{\partial T_F}{\partial r} (0, z, t) = 0$
- 2) $T_{co}(r, -\frac{H}{2}, t) = T_{PLENUM}$
- 3) $\frac{\partial T_{co}}{\partial z} (r, \frac{H}{2}, t) = 0$
- 4) $\frac{\partial T_{co}}{\partial r} (c, z, t) = 0$
- 5) $\frac{\partial T_F}{\partial z} (r, \pm \frac{H}{2}, t) = \frac{\partial T_c}{\partial z} (r, \pm \frac{H}{2}, t) = 0$
- 6) $T_F(\underline{r}, 0) = T_F^{\circ}(\underline{r})$
- 7) $T_c(\underline{r}, 0) = T_c^{\circ}(\underline{r})$
- 8) $T_{co}(\underline{r}, 0) = T_{co}^{\circ}(\underline{r})$

Boundary condition 1) results from the assumed azimuthal symmetry. The coolant has been assumed to enter the flow channel at a constant temperature, T_{PLENUM} . This results in condition 2). Boundary conditions 3) and 5) result from an assumption that no heat is transferred axially from the fuel rod. Boundary condition 4) is the result of the use of the equivalent cell. Conditions 6), 7), and 8) are the assumed initial conditions.

Solution of the nonlinear coupled neutronic and energy transport problem involves the solution of the partial differential equations (11), (12), (21), (24), (25), and (26), with the appropriate boundary, interface, and initial conditions.

Several works, Refs. [4], [11], [12], have demonstrated the feasibility and success of the finite element method in solving nuclear reactor dynamics problems. The FEM

is used to reduce the partial differential equations developed in this analysis to a system of ordinary differential equations. Integration of these ordinary differential equations (ODE) yields the solution.

IV. FINITE ELEMENT FORMULATION

In this section, the basic theory underlying the finite element method is formulated. Selection of the finite elements and the shape functions for the elements are given. Some simple transformations which facilitate the integration necessary in the FEM are also presented.

A. BASIC THEORY

To obtain a numerical solution, the governing partial differential field equations are transformed into a system of ODE in finite dimensional vector space. This may be accomplished in several manners such as the finite-difference method, the variational method, or the weighted residual method. In this work, the Galerkin method (a weighted residual method) is utilized for the discretization of the spatial domain. The Galerkin procedure will be applied to each of the governing field equations. Any of these equations may be considered to be in the following form

$$\frac{\partial \psi}{\partial t}(\underline{r}, t) - \mathcal{L} \psi(\underline{r}, t) = f(\underline{r}, t) \quad (31)$$

where ψ represents the unknown function, e.g., in equation (21), ψ represents ϕ_F , \mathcal{L} represents the operator for each individual equation, and f is a forcing function. In the finite element method the solution is approximated as

$$\psi(\underline{r}, t) \approx \tilde{\psi}(\underline{r}, t) = \sum_{i=1}^N N_i(\underline{r}) \psi_i(t) = \langle N_i \rangle \{ \psi_i \} \quad (32)$$

where

N - the number of degrees of freedom

N_i - the element shape function

ψ_i - unknown coordinate function

$< >$ - matrix notation for a row vector

$\{ \}$ - matrix notation for a column vector

$<N_i> = <N_1 \ N_2 \ \dots \ N_i \ \dots \ N_N>$

$$\{N_i\} = \begin{Bmatrix} N_1 \\ N_2 \\ \vdots \\ N_i \\ \vdots \\ N_N \end{Bmatrix}$$

The residual, $R(\underline{r}, t)$, is a measure of the error in this finite element approximation. The residual may be considered as

$$R = \frac{\partial \psi}{\partial t} - \mathcal{L} \psi - f \quad (33)$$

The best solution for $\tilde{\psi}$ is one which "minimizes" this residual. Various "minimums" are obtained by the weighted residual method by setting

$$\int_V w_i(\underline{r}) R \, dV = 0 \quad i=1,2,\dots,N \quad (34)$$

$w_i(\underline{r})$ - weighting functions

With the Galerkin method, the weighting functions are the shape functions defining the approximation of equation (32) (i.e., $w_i = N_i$). A noteworthy attribute of the Galerkin method is the opportunity of using an integration-by-parts

of the terms involving the second order spatial derivatives. A lower order finite element may be used than would have been possible otherwise. Once the weighting functions have been chosen, the problem becomes

$$\int_V N_i \left\{ \frac{\partial \psi}{\partial t} - \mathcal{L} \psi - f \right\} dV = 0 \quad i=1,2,\dots,N \quad (35)$$

The integration involved in equation (35) is carried out on the element level, taking advantage of the use of a "local" coordinate system. Once the integration is accomplished, the results are merged into a system using "global" coordinates. On the element level

$$\tilde{\psi}^e = \langle N_j \rangle^e \{\psi_j\}^e \quad j=1,2,\dots,N \quad (36)$$

where the superscript e indicates the element level. Substituting $\tilde{\psi}^e$ into equation (35) and noting $\{\psi_j\}^e$ is not a function of the spatial domain yields

$$\int_V \langle N_i \rangle^e \{N_j\}^e dV^e \{\dot{\psi}_j\}^e - \int_V [\langle N_i \rangle^e \mathcal{L} \{N_j\}^e - \langle N_i \rangle^e f^e] dV^e \{\psi_j\}^e = 0 \quad (37)$$

where $i, j = 1, 2, \dots, N^e$
 N^e - number of degrees of freedom for an element

The operator \mathcal{L} will vary depending upon which governing equation is under consideration.

B. SHAPE FUNCTIONS

The shape functions, N_i , are chosen to satisfy certain completeness and convergence criteria [13] and will depend upon the finite-element used for the spatial discretization.

Many previous works, Refs. [4], [11], and [12], utilized linear triangular shaped elements to discretize the spatial domain. This element was the first element considered. However, because the width of the cladding is very thin, elements in the cladding region would have extremely large aspect ratios (ratio of base to height) unless an extremely large number of elements in the axial direction were used. A large number of elements becomes numerically untractable. Previous experience with triangular elements had shown that large aspect ratios yield inaccurate results. Zlamal, Ref. [14], showed the error, e , when using triangular elements, is proportional to the square of the longest side, h , and inversely proportional to the sine of the smallest angle, γ

$$e \propto h^2 / \sin \gamma$$

A triangular element with a large aspect ratio necessarily must have a small related angle which adversely affects the error in the FEM. Hopefully to alleviate the problem, an isoparametric, quadratic, rectangular element was selected. The aspect ratio would still be large, but experience, Ref. [15], with the use of rectangular elements indicated that a large aspect ratio is not always a detrimental factor.

The shape functions for this element are well documented, Ref. [13]. Utilizing a "local" coordinate system (See Figure 4), the shape functions may be written as

Corner nodes $i = 1, 3, 5, 7$

$$N_i = \frac{1}{4}(1+\xi_0)(1+\eta_0)(\xi_0+\eta_0-1) \quad (38)$$

Midside nodes $N_i = \frac{1}{2}(1-\xi^2)(1+\eta_0) \quad i=2,6 \quad (38a)$

$$N_i = \frac{1}{2}(1+\xi_0)(1-\eta^2) \quad i=4,8 \quad (38b)$$

where

$$\xi_0 = \xi \xi_i$$

$$\eta_0 = \eta \eta_i$$

These normalized shape functions are shown in Figure 5.

The local and global coordinates are related by the following

$$r = \langle N_i \rangle^e \{r_i\}^e \quad (39)$$

and $z = \langle N_i \rangle^e \{z_i\}^e \quad (39a)$

C. COORDINATE TRANSFORMATIONS

When using a local coordinate system, some simple transformations facilitate the integrations required by equation (37). In cylindrical coordinates with azimuthal symmetry

$$dV = 2\pi r dr dz \quad (40)$$

The derivative terms may be transformed by the following

$$\begin{Bmatrix} \frac{\partial N_i}{\partial r} \\ \frac{\partial N_i}{\partial z} \end{Bmatrix} = [J]^{-1} \begin{Bmatrix} \frac{\partial N_i}{\partial \xi} \\ \frac{\partial N_i}{\partial \eta} \end{Bmatrix} \quad (41)$$

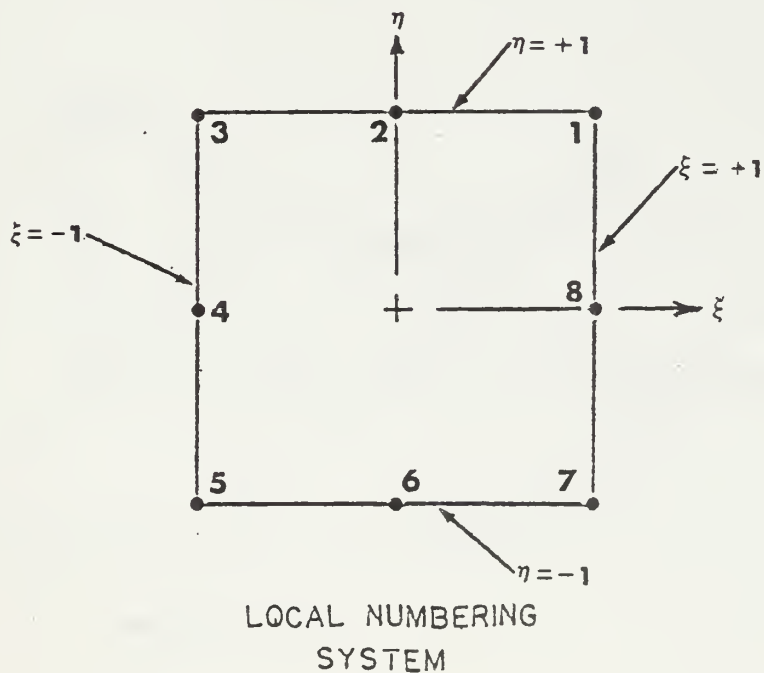
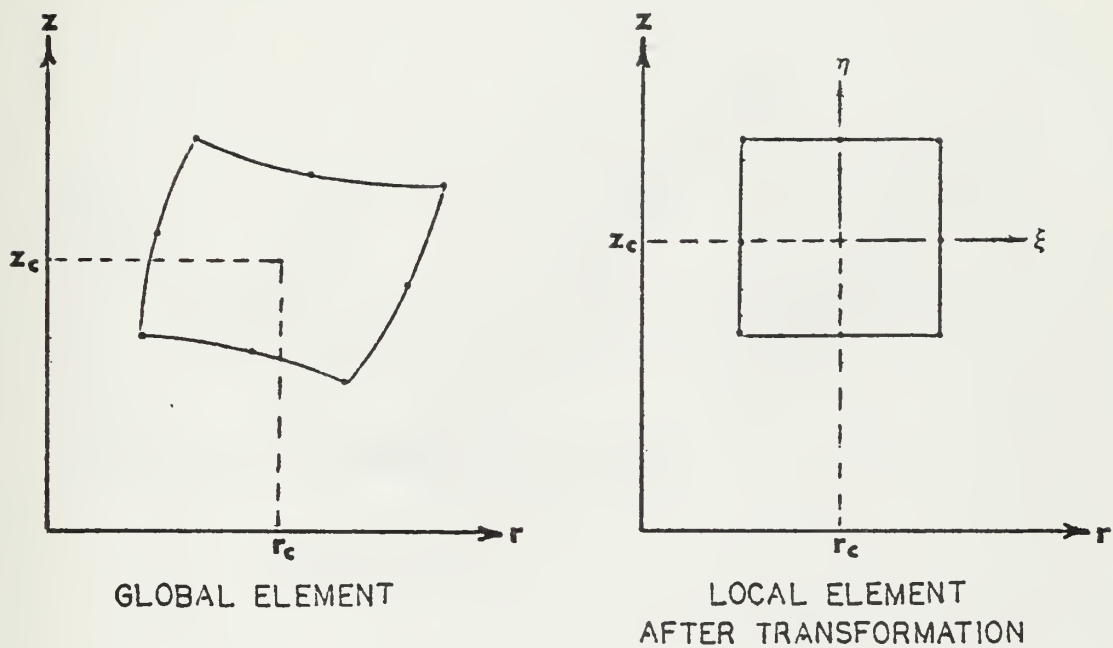
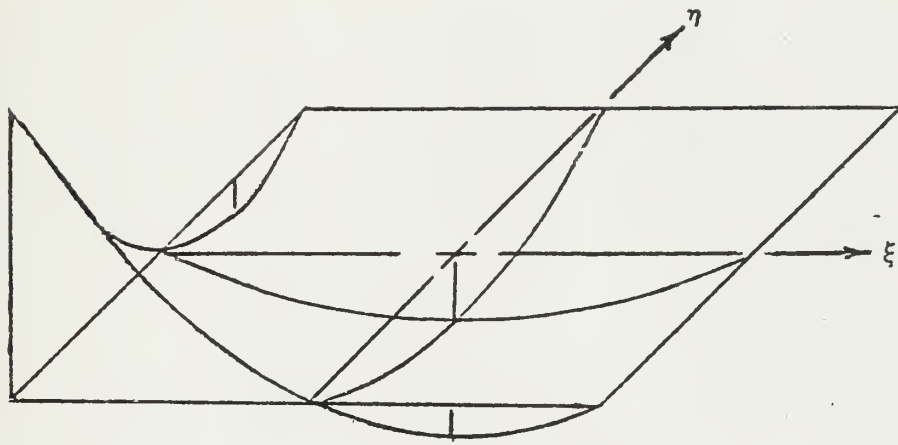
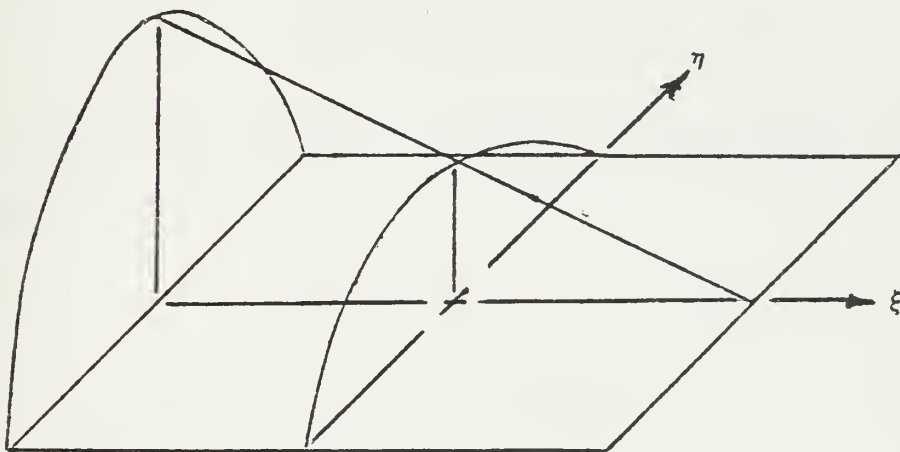


Figure 4. Element Transformation



CORNER NODE



MIDSIDE NODE

Figure 5. Normalized Shape Functions

where $[J]^{-1}$ is the inverse of the 2x2 Jacobian matrix defined in Appendix A. As shown in Appendix A, this inverse can be easily shown to be (for this problem)

$$[J]^{-1} = \begin{bmatrix} J_{11}^* & J_{12}^* \\ J_{21}^* & J_{22}^* \end{bmatrix} \quad (42)$$

where

$$\begin{aligned} J_{11}^* &= 2/\Delta r, & J_{12}^* &= 0 \\ J_{21}^* &= 0, & J_{22}^* &= 2/\Delta Z \\ \Delta r &- \text{radial length of the element} \\ \Delta Z &- \text{axial length of the element} \end{aligned}$$

Elements of area transform as

$$drdz = \det[J] d\xi d\eta \quad (43)$$

For this particular problem, $\det[J]$ may be shown to be (Appendix A)

$$\det[J] = \frac{A^e}{4} \quad (44)$$

where A^e - area of the element

Elements of axial length become

$$dz = \frac{L^e}{2} \quad (45)$$

where L^e - axial length of the element

Utilization of these transformations makes the integrations required in equation (37) amenable to integration by numerical Gaussian quadrature.

V. APPLICATION OF FEM TO GOVERNING FIELD EQUATIONS

In this chapter, equation (37) is applied to the governing field equations previously derived. The element matrices for each of the operators are developed so that the discretization of the spatial domain may be accomplished. The integrations required by the application of equation (37) are performed numerically using Gaussian quadrature.

A. GAUSSIAN QUADRATURE

Prior to the application of equation (37), it is appropriate to discuss briefly the procedure used for the numerical integration. The product Gaussian quadrature formula is [16]

$$I_A = \int_{-1}^1 \int_{-1}^1 g(\xi, \eta) d\eta d\xi = \sum_{i=1}^m \sum_{j=1}^m W_i W_j g(\xi_i, \eta_j) \quad (46)$$

where

I_A	- area integration
$g(\xi, \eta)$	- any function of ξ and η
$W_{i,j}$	- weight associated with location i or j
m	- number of Gauss sampling points in one-dimension

The values of the weights associated with each Gauss point are given in Ref. [16]. Equation (42) may be simplified somewhat by combining the summations and weights

$$I_A = \sum_{k=1}^{m^2} W_k g(\xi_k, \eta_k) \quad (47)$$

where

$$k = i \times j$$

$$W_k = W_i \times W_j$$

For line integrations, the Gaussian quadrature formula involves only one summation

$$I_L = \int_{-1}^1 f(\eta) d\eta = \sum_{i=1}^m W_i f(\eta_i) \tag{48}$$

B. NEUTRONIC FIELD EQUATIONS

The discretization of the spatial domain by the finite element method is accomplished by applying equation (37) to the governing field equations, using

$$\phi_k = \langle N_j \rangle^e \{ \psi_{kj} \}^e \quad \begin{matrix} j=1,2,\dots,8 \\ k=F,c,co \end{matrix} \tag{49}$$

1. Fuel Region

The governing equation for the fuel region, equation (20) after applying equation (37) becomes

$$\begin{aligned} & 2\pi \int_r \int_z \frac{N_i}{v} \frac{\partial \phi_F}{\partial t} r dr dz - 2\pi \int_r \int_z N_i \left\{ \frac{1}{r} \frac{\partial}{\partial r} (r D_F \frac{\partial \phi_F}{\partial r}) \right. \\ & \quad + \frac{\partial}{\partial z} (D_F \frac{\partial \phi_F}{\partial z}) + \Sigma_{aF} \phi_F (1-\beta) [k_{\infty}^o + \rho - \beta \ln(T_F/T_F^o)] \\ & \quad \left. + \bar{\lambda} \bar{\beta} \Sigma_{aF} \int_0^t e^{-\lambda_i(t-t')} (k_{\infty}^o + \rho) \phi_F dt' + \bar{\lambda} C^o e^{-\lambda t} \right\} r dr dz = 0 \end{aligned} \tag{50}$$

The second order terms in equation (50) may be reduced to a first order term by application of Green's Theorem or equivalently integration-by-parts (See Appendix B). Dividing through by 2π and reducing the second order terms yields

$$\begin{aligned} & \int_z r N_i D_F \frac{\partial \phi_F}{\partial r} dz + \int_r r N_i D_F \frac{\partial \phi_F}{\partial z} dr + \int_r \int_z \frac{N_i}{v} \frac{\partial \phi_F}{\partial t} r dr dz \\ & + \int_r \int_z \left\{ D_F \left[\frac{\partial N_i}{\partial r} \frac{\partial \phi_F}{\partial r} + \frac{\partial N_i}{\partial z} \frac{\partial \phi_F}{\partial z} \right] - N_i \Sigma_{aF} \phi_F (1-\beta) [k_{\infty}^{\circ} + \rho - b \ln(T_F/F_{F0})] \right. \\ & \left. - N_i \bar{\lambda} \bar{\beta} \Sigma_{aF} \int_0^t e^{-\lambda(t-t')} (k_{\infty}^{\circ} + \rho) \phi_F dt' - N_i \bar{\lambda} C^{\circ} e^{-\bar{\lambda} t} \right\} r dr dz = 0 \quad (51) \end{aligned}$$

From continuity and boundary conditions the line integrals are zero. Now using the approximate functions of equation (49) and noting that $\{\psi_F\}^e$ is not a function of space, equation (51) may be written as

$$\begin{aligned} & \frac{1}{v} \iint_{r,z} \{N_i\}^e \langle N_j \rangle^e r dr dz \{\psi_F\}^e + \iint_{r,z} D_F [\{N_{i,r}\}^e \langle N_{j,r} \rangle^e + \{N_{i,z}\}^e \langle N_{j,z} \rangle^e] r dr dz \{\psi_F\}^e \\ & - \iint_{r,z} \Sigma_{aF} (1-\beta) [k_{\infty}^{\circ} + \rho - b \ln(T_F/T_F^{\circ})] \{N_i\}^e \langle N_j \rangle^e r dr dz \{\psi_F\}^e - \bar{\lambda} \bar{\beta} \Sigma_{aF} \int_0^t e^{-\bar{\lambda}(t-t')} (k_{\infty}^{\circ} + \rho) dt' \\ & \iint_{r,z} \{N_i\} \langle N_j \rangle r dr dz \{\psi_F\}^e - \bar{\lambda} C^{\circ} e^{-\bar{\lambda} t} \iint_{r,z} \{N_i\} r dr dz = 0 \quad (52) \end{aligned}$$

The integrations may not be easily carried out with a shift to local coordinates and with use of the previously derived transformations. Rearranging equation (52) and assuming the properties are constant for each time step gives

$$\begin{aligned}
 D_F \int_{-1}^1 \int_{-1}^1 & [\{N_{i,\xi}^{J_{11}}\}^* \langle J_{11}^* N_{j,\xi} \rangle + \{N_{i,\eta}^{J_{22}}\}^* \langle J_{22}^* N_{j,\eta} \rangle] r_{\det}[J] d\xi d\eta \{\psi_F\}^e \\
 - \Sigma_{aF} (1-\beta) (k_{\infty}^o + \rho) & \int_{-1}^1 \int_{-1}^1 \{N_i\} \langle N_j \rangle r_{\det}[J] d\xi d\eta \{\psi_F\}^e \\
 - \Sigma_{aF} (1-\beta) b & \int_{-1}^1 \int_{-1}^1 \ln \frac{T_F}{T_o} \{N_i\} \langle N_j \rangle r_{\det}[J] d\xi d\eta \{\psi_F\}^e \\
 - \bar{\lambda} \bar{\beta} \Sigma_{aF} f(t) & \int_{-1}^1 \int_{-1}^1 \{N_i\} \langle N_j \rangle r_{\det}[J] d\xi d\eta \{\psi_F\}^e - \bar{\lambda} C^o e^{-\bar{\lambda} t} \int_{-1}^1 \int_{-1}^1 \{N_i\} r_{\det}[J] d\xi d\eta \\
 + \frac{1}{V} & \int_{-1}^1 \int_{-1}^1 \{N_i\} \langle N_j \rangle r_{\det}[J] d\xi d\eta \{\dot{\psi}_F\}^e = 0
 \end{aligned} \tag{53}$$

Since the element chosen has eight degrees of freedom (nodal points), the discretized matrices which result from the integration of equation (53) will be 8x8 matrices and the forcing function will be an 8x1 vector at the element level. Defining the matrices as

$$\begin{aligned}
 & \int_{-1}^1 \int_{-1}^1 [\{N_{i,\xi}^{J_{11}}\}^* \langle J_{11}^* N_{j,\xi} \rangle + \{N_{i,\eta}^{J_{22}}\}^* \langle J_{22}^* N_{j,\eta} \rangle] r_{\det}[J] d\xi d\eta \\
 = [H]_{2_{ij}} &_{8 \times 8} = \frac{A^e}{4} \sum_{k=1}^{m^2} [(N_{i,\xi})_k (N_{j,\xi})_k J_{11}^{*2} \\
 & + (N_{i,\eta})_k (N_{j,\eta})_k J_{22}^{*2}] r_k W_k
 \end{aligned} \tag{54}$$

where $W_k = W_i W_j$ and $r_k = \sum_{i=1}^8 r_i (N_i)_k$

$$\int_{-1}^1 \int_{-1}^1 \{N_i\} \langle N_j \rangle \text{rdet}[J] d\xi d\eta = [H3_{ij}]_{8 \times 8} = \frac{A e^{m^2}}{4} \sum_{k=1}^8 (N_i)_k (N_j)_k r_k W_k \quad (55)$$

$$\int_{-1}^1 \int_{-1}^1 \{N_i\} \text{rdet}[J] d\xi d\eta = \{F_i\}_{8 \times 1} = \frac{A e^{m^2}}{4} \sum_{k=1}^8 (N_i)_k r_k W_k \quad (56)$$

$$\begin{aligned} \int_{-1}^1 \int_{-1}^1 \ln(T_F/T_D) \{N_i\} \langle N_j \rangle \text{rdet}[J] d\xi d\eta &= [H4_{ij}]_{8 \times 8} \\ &= \frac{A e^{m^2}}{4} \sum_{k=1}^8 \ln(T_F/T_{F^0})_k (N_i)_k (N_j)_k r_k W_k \end{aligned} \quad (57)$$

To carry out the summation of equation (57), the temperature T_F must be known; however, the temperature is exactly what is being sought. To alleviate this problem, a linearization is used.

In the solution technique, the temperature is predicted for the next time step. It is this temperature which is used for the determination of matrix $H4$.

Equation (53) simplifies to

$$\begin{aligned} \{D_F[H12_{ij}] - [\sum_{aF} (1-\beta)(k_\infty^0 + \rho) + \bar{\lambda} \bar{\beta} \sum_{aF} f(t)] [H3_{ij}] + \sum_{aF} (1-\beta)b [H4_{ij}]\} \{\psi_F\}^e \\ - \bar{\lambda} C^0 c^{-\bar{\lambda} t} \{F_i\}^e + \frac{1}{V} [H3_{ij}] \{\dot{\psi}_F\} = 0 \end{aligned} \quad (58)$$

The function $f(t)$ is evaluated by summing the values at each time step using the trapezoid rule for numerical integration.

$$f(t) = e^{-\bar{\lambda}t} \int_0^t e^{-\bar{\lambda}t'} [k_{\infty}^o + \rho] dt' \quad (59)$$

Defining

$$I_i[g(t)] = \frac{1}{2} h_i \{g(t_i) + g(t_{i-1})\} \quad (60)$$

$$g(t) = e^{\bar{\lambda}t} (k_{\infty}^o + \rho)$$

h_i - time step taken

$$g_0 = 0$$

The function may be expressed as

$$f(t) = e^{-\bar{\lambda}t} \sum_{i=1}^S I_i [g(t)] \quad (61)$$

S - number of time steps

2. Clad Region

Following the same procedure as with the fuel equation, the discretized form of equation (11) becomes

$$\{D_c[H12_{ij}] + \Sigma_{ac}[H3_{ij}]\}\{\psi_c\}^e + \frac{1}{V}[H3_{ij}]\{\dot{\psi}_c\}^e = 0 \quad (62)$$

3. Coolant Region

The governing equation for the coolant region, equation (12), may be discretized into the form

$$\{D_{co}[H12_{ij}] + \Sigma_{aco}[H3_{ij}]\}\{\psi_{co}\}^e + \frac{1}{V}[H3_{ij}]\{\dot{\psi}_{co}\}^e = 0 \quad (63)$$

Once the governing equations have been discretized at the element level, they are combined into a system of equations at the global level. On the global level the governing equation for neutron transport takes the form of

$$[H]_{n \times n} \{\psi\}_{n \times 1} + [P]_{n \times n} \{\dot{\psi}\}_{n \times 1} + \{F\}_{n \times 1} = 0 \quad (64)$$

where $[H]$, $[P]$, and $[F]$ represent the system matrices and n is the number of nodal points used in the discretization. There are, then, n simultaneous ordinary differential equations used to describe the neutron transport problem.

C. HEAT TRANSPORT FIELD EQUATIONS

The spatial domain for the heat transport problem is discretized in the same manner as the domain for the neutronics problem. The same element matrices previously defined are valid. Let

$$T_k = \langle N_j \rangle^e \{\tau_{kj}\}^e \quad \begin{array}{l} j=1,2,\dots,8 \\ k=F,c,co \end{array} \quad (65)$$

1. Fuel Region

The governing field equation for the fuel, equation (24), is discretized by applying equation (37). Using an integration by part to lower the order of the second order terms allows equation (24) to be written as

$$\int_r \left[N_i k_F \frac{\partial T_F}{\partial z} \right]_0^z r dr dz + \int_z \left[N_i k_F \frac{\partial T_F}{\partial r} \right]_0^a dz - \int_r \int_z \left\{ k_F \left[\frac{\partial N_i}{\partial r} \frac{\partial T_F}{\partial r} + \frac{\partial N_i}{\partial z} \frac{\partial T_F}{\partial z} \right] - e \Sigma_{fF} N_i \phi_F + N_i \rho_F C_{pF} \frac{\partial T_F}{\partial t} \right\} r dr dz = 0 \quad (66)$$

From continuity considerations the line integrals are zero except along the boundaries of a region. It is assumed that no heat is transferred from the cell in the axial direction (boundary condition 5); therefore, the first line integral of equation (66) is zero. In the neutron diffusion problem there was continuity of flux at the interfaces so that the line integrals were zero; however, the heat transfer at the interfaces is affected by the gap and film conductances. The fuel-clad interface condition, equation (29), may be rewritten as

$$-k_F \frac{\partial T_F}{\partial r} = H_{\text{gap}}(T_F - T_C) = -k_C \frac{\partial T_C}{\partial r} \quad (67)$$

Substituting equation (67) into (66), dividing by minus one and utilizing equation (65) yields

$$\begin{aligned} & \int_z [r H_{\text{gap}} \{N_i\}^e \langle N_j \rangle^e]_0^a dz \{\tau_F\}^e - \int_z [r H_{\text{gap}} \{N_i\}^e \langle N_j \rangle^e]_0^a dz \{\tau_C\}^{e*} \\ & + \int_r \int_z k_F [\{N_{i,r}\}^e \langle N_{j,r} \rangle^e + \{N_{i,z}\}^e \langle N_{j,z} \rangle^e] r dr dz \{\tau_F\}^e \\ & - \int_r \int_z e \Sigma_{fF} \{N_i\}^e \langle N_j \rangle^e r dr dz \{\psi_F\}^e \\ & + \int_r \int_z \rho_F C_{pF} \{N_i\}^e \langle N_j \rangle^e r dr dz \{\dot{\tau}_F\}^e = 0 \end{aligned} \quad (68)$$

e^* - element across the interface

Transforming to local coordinates and integrating by Gaussian quadrature yields the same element matrices as given for the neutron flux, equations (54) and (55), except for the line integrals between regions. It should be noted that the line integrals exist only on the interfaces, along which there is a discontinuity of temperature. For the fuel equation the interface corresponds to the local coordinate $\xi=1$. Define

$$\int_{-1}^1 \{N_i\} \{N_j\} d\eta = [K1]_{ij} 8 \times 8 = \frac{L^e}{2} \sum_{k=1}^{m^2} (N_i)_k (N_j)_k w_k \quad (69)$$

where the N 's are evaluated at $\xi=1$. Many of the terms of $K1$ will be zero since only the nodes on the $\xi=1$ boundary will have shape functions which are non-zero. It is through $K1$ that the temperatures for each region are coupled together. Equation (68) may now be written as

$$r_a H_{gap} \{ [K1] \{ \tau_F \}^e - [K1] \{ \tau_C \}^{e*} \} + k_F [H12] \{ \tau_F \}^e - e \Sigma_{fF} [H3] \{ \psi_F \}^e + \rho_F C_{pF} [H3] \{ \dot{\tau}_F \} = 0 \quad (70)$$

In obtaining this equation, it was assumed that material properties for each nodal point were constant at each time step. Perhaps a better assumption would have been to assume an average value for the properties of each element. The difference should not be significant, and the assumed constant nodal properties were numerically more tractable.

2. Clad Region

Applying equation (37) to the governing field equation for the clad, equation (25) gives the discretized form of the equation. Assuming no heat transfer in the axial direction on the boundaries (Boundary condition 5), equation (25) becomes

$$\begin{aligned}
 - \int_z [N_i r k_c \frac{\partial T_c}{\partial r}]_a^b dz + \int_r \int_z k_c [\frac{\partial N_i}{\partial r} \frac{\partial T_c}{\partial r} + \frac{\partial N_i}{\partial z} \frac{\partial T_c}{\partial z}] r dr dz \\
 + \int_r \int_z \rho_c C_{pc} N_i \frac{\partial T_c}{\partial t} r dr dz = 0
 \end{aligned} \quad (71)$$

In the cladding, there are two interfaces along which the line integral of equation (71) is not zero, along the fuel-clad interface and along the clad-coolant interface. For the fuel-clad interface, equation (61) applies. For the clad-coolant interface, the interface condition, equation (30), may be rewritten as

$$k_c \frac{\partial T_c}{\partial r} = h_{surf} (T_c - T_{co}) = -k_{co} \frac{\partial T_{co}}{\partial r} \quad (72)$$

Along the fuel-clad interface, the local coordinate corresponds to $\xi = -1$. Define the new element matrix

$$\int_{-1}^1 \int_{-1}^1 \{N_i\}^e \langle N_j \rangle^e d\eta = [K2_{ij}]_{8 \times 8} = \frac{L}{2} \sum_{k=1}^m (N_i)_k (N_j)_k w_k$$

where the N 's are evaluated at $\xi = -1$. As with $K1$, $K2$ will have many zero values because the shape functions are evaluated at $\xi = -1$.

Along the clad-coolant interface, the local coordinate corresponds to $\xi=1$ and the K1 matrix is appropriate.

Substituting equations (67) and (72), equation (71) becomes

$$\begin{aligned} & \int_z [r h_{\text{surf}} N_i (T_c - T_{co})]_b dz - \int_z [r h_{\text{gap}} N_i (T_F - T_c)]_a dz \\ & + \int_r \int_z k_c \left[\frac{\partial N_i}{\partial r} \frac{\partial T_c}{\partial r} + \frac{\partial N_i}{\partial z} \frac{\partial T_c}{\partial z} \right] r dr dz \\ & + \int_r \int_z \rho_c C_{pc} N_i \frac{\partial T_c}{\partial t} r dr dz = 0 \end{aligned} \quad (73)$$

The governing equation for the clad region may now be written as

$$\begin{aligned} & r_a h_{\text{gap}} \{ [K2] \{ \tau_c \}^e - [K2] \{ \tau_F \}^{e*} \} + r_b h_{\text{surf}} \{ [K1] \{ \tau_c \}^e - [K1] \{ \tau_{co} \}^{e*} \} \\ & + k_c [H12] \{ \tau_F \}^e + \rho_c C_{pc} [H3] \{ \dot{\tau}_c \} = 0 \end{aligned} \quad (74)$$

The line integrals of equation (73) affect only nodes which are on one of the boundaries; therefore, the nodal inputs into K1 and K2 are zero unless the node is on one of the boundaries.

3. Coolant Region

The field equation governing the coolant may be discretized in the same manner as above. After applying the Galerkin method and performing an integration by parts

on the second order terms, equation (25)

becomes

$$\begin{aligned}
 & - \int_z [r N_i k_{co} \frac{\partial T}{\partial r}]_b^c dz + \int_r \int_z k_{co} \left[\frac{\partial N_i}{\partial r} \frac{\partial T_{co}}{\partial r} + \frac{\partial N_i}{\partial z} \frac{\partial T_{co}}{\partial z} \right] r dr dz \\
 & + \int_r \int_z V_{co} \rho_{co} C_{pco} N_i \frac{\partial T_{co}}{\partial z} r dr dz \\
 & + \int_r \int_z \rho_{co} C_{pco} N_i \frac{\partial T_{co}}{\partial t} r dr dz = 0 \quad (75)
 \end{aligned}$$

The line integral, when evaluated at c, is zero (boundary condition 4). When evaluated at b, or correspondingly at $\xi = -1$, equation (72) is valid and K2 matrix is appropriate. All the terms of equation (75) have been defined except the flow term. Define

$$\begin{aligned}
 & \int_{-1}^1 \int_{-1}^1 \{N_i\}^e \langle N_{i,n} \rangle^e r \det[J] d\xi d\eta = [H5_{ij}]_{8 \times 8} \\
 & = \frac{A^e m^2}{4} \sum_{k=1}^2 (N_i)_k (N_{i,n})_k r_k w_k \quad (76)
 \end{aligned}$$

Transforming to local coordinates and integrating reduces equation (75) to

$$\begin{aligned}
 & r_b h_{surf} \{ [K2] \{ \tau_{co} \}^e - [K2] \{ \tau_c \}^{e*} \} + k_{co} [H12] \{ \tau_{co} \}^e \\
 & + V_{co} \rho_{co} C_{pco} [H5] \{ \tau_{co} \}^e + \rho_{co} C_{pco} [H3] \{ \dot{\tau}_{co} \}^e = 0 \quad (77)
 \end{aligned}$$

Now that the governing equations have been defined for each region on the element level, equations (70), (74), and (77), they may be assembled into a system equation on the global level. The equation will be in the general form of

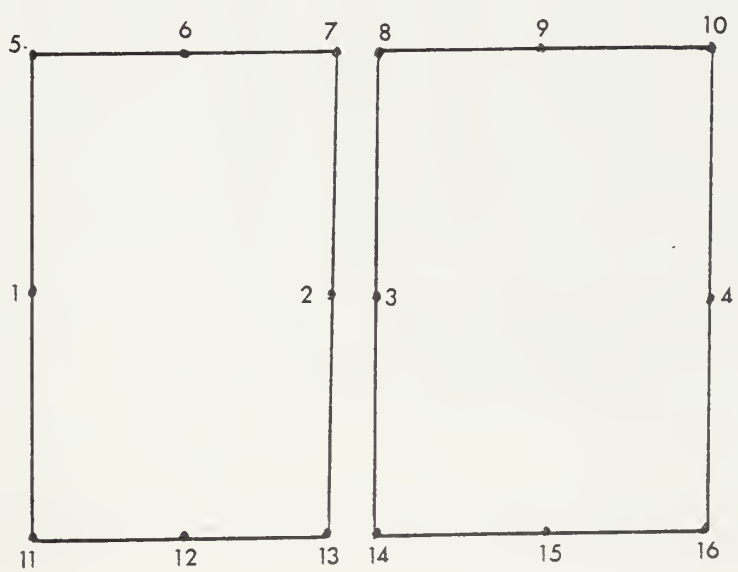
$$[K]_{n \times n} \{\tau\}_{n \times 1} + [M]_{n \times n} \{\psi\}_{n \times 1} + [G]_{n \times n} \{\dot{\tau}\}_{n \times 1} = 0 \quad (78)$$

D. DISCRETIZATION OF THE SPATIAL DOMAIN

Prior to the numerical solution of the governing equations, equations (64) and (78), the spatial domain must be divided into a number of elements. For this work the domain was subdivided as shown in figure 5.

Since there is a discontinuity of temperatures at the interfaces, as described by equations (29) and (30), a novel application of the FEM method was necessary. The common practice for handling these "flux" type boundary conditions is to define a constant reference temperature, T_{∞} , as when working with a convection heat transfer problem [17], or to define a known function, as when working with a fracture mechanics problem [18]. In either case the reference condition was known. The novel application here lies in the use of a different field equation to describe the reference temperature, e.g., the clad equation (74) is the reference condition for heat transfer from the fuel across the gap interface.

The discontinuity of temperature at the interface necessitated another novel application of the FEM. Since there is a temperature drop along each interface, a single node there is not adequate. In the discretization of the domain, two nodes were used for each interface point (for example, points 62 and 63 in figure 6). This allows the temperature drop due to the gap and film conductances to be taken into consideration. Since two nodes are used, the governing equations for each region are not directly coupled together. The coupling of the regions is accomplished by the "flux" boundary or interface conditions since it is assumed that any heat flux leaving a region enters the adjacent region. Consider a typical set of elements on an interface



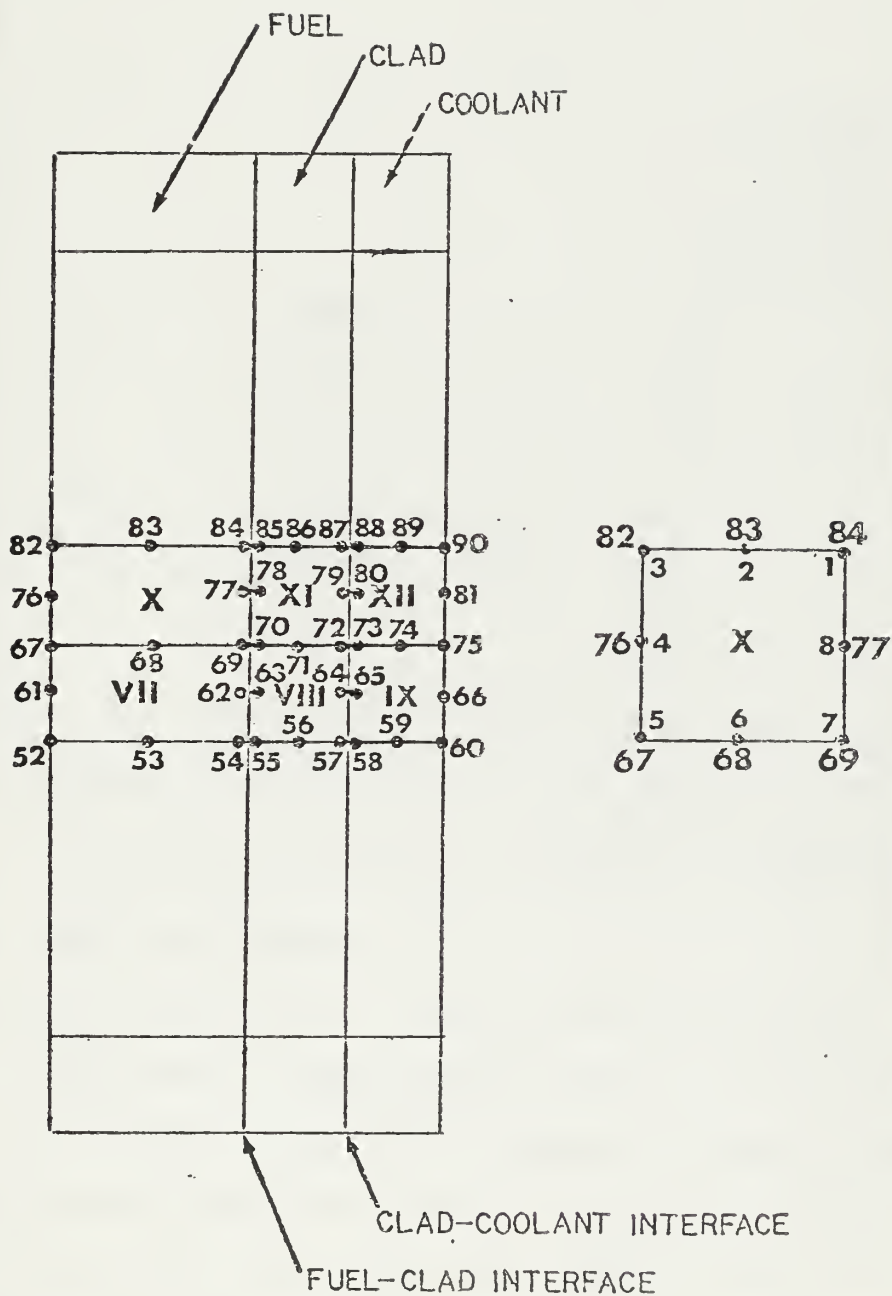


Figure 6. Finite Element Discretization

The coupling terms K_1 and K_2 may be combined into a system K matrix which shows the coupling. The K matrix for the simple set shown is

$$\begin{array}{c}
 \begin{array}{cccccccccccccccc}
 & 1 & 2 & 3 & 4 & 5 & 6 & 7 & 8 & \cdots & 13 & 14 & 15 & 16 \\
 \begin{array}{c} 1 \\ 2 \\ 3 \\ 4 \\ 5 \\ 6 \\ 7 \\ 8 \\ \vdots \\ \vdots \\ 13 \\ 14 \\ 15 \\ 16 \end{array} & \left[\begin{array}{cccccccccccccccc}
 & & & & & & & & & & & & & \\
 & \mathbf{a} & -\mathbf{a} & & & & & \mathbf{c} & -\mathbf{c} & & & \mathbf{c} & -\mathbf{c} & \\
 & -\mathbf{a} & \mathbf{a} & & & & & -\mathbf{c} & \mathbf{c} & & & -\mathbf{c} & \mathbf{c} & \\
 & & & & & & & & & & & & & \\
 & & & & & & & & & & & & & \\
 & & & & & & & & & & & & & \\
 & \mathbf{c} & -\mathbf{c} & & & & & \mathbf{b} & -\mathbf{b} & & & \mathbf{d} & -\mathbf{d} & \\
 & -\mathbf{c} & \mathbf{c} & & & & & -\mathbf{b} & \mathbf{b} & & & -\mathbf{d} & \mathbf{d} & \\
 & \vdots & & & & & & & & & & & & \\
 & \vdots & & & & & & & & & & & & \\
 & \mathbf{c} & -\mathbf{c} & & & & & \mathbf{d} & -\mathbf{d} & & & \mathbf{b} & -\mathbf{b} & \\
 & -\mathbf{c} & \mathbf{c} & & & & & -\mathbf{d} & \mathbf{d} & & & -\mathbf{b} & \mathbf{b} & \\
 & & & & & & & & & & & & & \\
 & & & & & & & & & & & & &
 \end{array} \right]
 \end{array}
 \end{array}
 \quad
 \begin{array}{l}
 \mathbf{a} = 16/15 \\
 \mathbf{b} = 4/15 \\
 \mathbf{c} = 2/15 \\
 \mathbf{d} = 1/15
 \end{array}$$

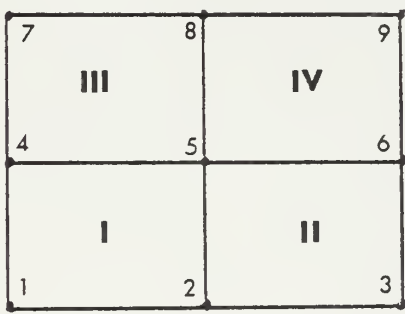
As can be seen, the nodes on the interface are coupled to the adjacent element interface nodes. For example, node 2 in element I is coupled to nodes 3, 8, and 14 in element II.

E. OPTIMUM COMPACTING SCHEME

The system matrices (H , P , etc.) are $n \times n$ matrices, where n is the number of nodal points used in the discretization of the domain. In terms of computer storage, these matrices may become excessively large if they are stored as $n \times n$. There are several techniques available to reduce this storage requirement. The most common method is the banded storage scheme, whereby only the banded portion of the matrices are stored. With judicious numbering of the nodes,

considerable savings may be realized. However, it is not the optimum storage scheme [8].

Since the shape functions, N_i , for the k^{th} nodal equation are nonzero over only the element containing k , the system matrices are not only banded but sparse as well. The sparseness is due to the non-consecutive numbering of the nodes surrounding the k^{th} node. The optimum compact storage (OCS) scheme compacts the matrices by storing only the non-zero elements of the matrices. The implementation of the OCS scheme requires two additional integer arrays, say JA and NAME. The NAME array identifies the nodal points which contribute to each nodal equation. The JA array acts as a pointer to indicate where the nodal equation starts in NAME. Consider the following simple 2x2 system with nodes as indicated



The NAME array starts with node 1 and identifies the nodes which contribute to node 1 (i.e., 5, 4, and 2). The NAME array would, then, give the nodes contributing to node 2 and so forth, so that

$$\begin{aligned}
 \text{and} \quad \langle \text{NAME} \rangle &= \begin{matrix} & \overset{1}{2} \overset{2}{3} \overset{3}{4} & \cdot & \overset{4}{5} \overset{5}{6} \overset{6}{7} \overset{7}{8} \overset{8}{9} \overset{9}{10} & \cdot & \overset{10}{11} \overset{11}{12} \overset{12}{13} \overset{13}{14} & \cdot & \dots & \cdot & \text{JC} \\
 & \langle 1542 : 254163 : 3652 : \dots : 9856 \rangle \end{matrix} \\
 \langle \text{JA} \rangle &= \langle \overset{1}{1} \overset{2}{5} \overset{3}{11} \dots \rangle
 \end{aligned}$$

The algorithm to assemble the element matrices into a compact storage vector is straightforward and represents a significant savings in computer storage [8]. The system matrices are stored as a vector rather than a two-dimensional array. For example, the value which would be stored in position (1,5) of the $n \times n$ array is stored in position 2 of the system vector.

VI. NUMERICAL SOLUTION

This section contains a brief description of possible solution techniques in addition to the solution technique chosen. Computer subroutines necessary to implement the technique are also described.

A. SELECTION OF METHOD

The numerical solution of the system of implicit ordinary differential equations, equations (64) and (78), may be accomplished by any of a number of different techniques such as Houbolt's method, Crank-Nicolson's method, Gear's method, or implicit Gear's method. It was not the objective of this analysis to determine which of the numerical solution schemes is the most efficient. Each method has its advantages and disadvantages. The Crank-Nicolson method is a single-step, implicit equation solver and, therefore, does not require storage of previous time solutions. When analyzing neutronic problems, the system of equations which arises is commonly very stiff (i.e., a rapid change in flux over a short period of time). The Crank-Nicolson method has, in a past work [8], demonstrated difficulty in tracking these stiff systems. Gear's method was specifically developed for stiff systems and can handle the problem very well. However, Gear's method is a multi-step, predictor-corrector method requiring storage of previous time solutions. In addition to this disadvantage, Gear's method requires the

transformation of the developed implicit O.D.E.'s into an explicit system of O.D.E.'s. After this transformation is done, the system matrices are no longer sparse or banded, thus eliminating the use of the optimum compacting scheme. In an effort to overcome these difficulties, Gear's method was modified, Ref. [9], to treat the implicit system of equations as well as to allow use of the optimum compacting scheme. A previous work, Ref. [8], has shown that the implicit Gear's method is particularly attractive in solving the type problem developed in this analysis. Therefore, the implicit Gear's method is used for the solution of the system of O.D.E.'s arising in this analysis.

No attempt will be made here to give the mathematics involved in developing the implicit Gear's method. Reference [9] may be consulted if details are desired. A listing of the computer program developed will be given in the Computer Program section. In order to utilize the implicit Gear's method, several user supplied subroutines must be developed: 1) DIFFUN, 2) JACMAT, and 3) NUITSL.

B. USER SUPPLIED SUBROUTINES TO IMPLEMENT THE IMPLICIT GEAR'S METHOD

1. DIFFUN

Subroutine DIFFUN evaluates equations (64) and (78) for a given time and for given values of ψ , $\dot{\psi}$, τ and $\dot{\tau}$. Since at each nodal point, i , there is a solution for the flux and for the temperature, the solution was set equal to DYI and DYII, respectively. In addition to having flux and

temperature at each nodal point, there are also three different regions in the domain which have different governing equations. An integer array, ITYPE, was developed to indicate for each nodal point whether it was: 0) a fuel node not in an interface element, 1) a fuel node in an interface element, 2) a cladding node or, 3) a coolant node. Using ITYPE, the computer program is directed to a different section depending upon the type of node being considered. After all the nodes have been considered, boundary conditions are established by changing DYI and DYII for the appropriate boundary nodes. Since in this analysis there is continuity of flux at the interfaces, special considerations must be given to these nodes. At the fuel-clad interface, the value of DYI for the clad node was set to the value of the flux at that node minus the value of the flux at the adjacent node (i.e., $DYI_i = \psi_i - \psi_{i-1}$). Similarly, at the clad-coolant interface, the value of DYI for the coolant node was set to the value of the flux at that node minus the value of the flux at the adjacent node. During the solution of the problem, DYI is driven toward zero, which in the limit forces ψ_i to equal ψ_{i-1} . This is the desired continuity result.

2. JACMAT

Subroutine JACMAT, evaluates the Jacobian matrix (for Gear's method) at the given time and for the current values of the dependent variables. The Jacobian for an equation of the type,

$$F(y, \dot{y}, t) = 0 \quad (79)$$

may be represented as [19],

$$J = \left[\frac{\partial F}{\partial y} - \frac{\alpha_0}{\beta_0 h} \frac{\partial F}{\partial \dot{y}} \right] \quad (80)$$

where α_0 and β_0 are coefficients from Gear's method and h is the time step. Using the notation of DIFFUN, let DYI and DYII represent equations (64) and (78), respectively. The Jacobian matrix may, then, be written as (J is called PW in JACMAT.)

$$PW = \left[\frac{\partial DYI}{\partial \psi} - \frac{\alpha_0}{\beta_0 h} \frac{\partial DYI}{\partial \dot{\psi}}, \frac{\partial DYII}{\partial \tau} - \frac{\alpha_0}{\beta_0 h} \frac{\partial DYII}{\partial \dot{\tau}} \right] \quad (81)$$

It is the form of equation (81) which is programmed in JACMAT. As in DIFFUN, the integer array ITYPE is used to indicate the appropriate section of the program to be utilized. The problem boundary conditions must also be accounted for in JACMAT. In DIFFUN, the value of DYI or DYII was set to zero for constant boundary conditions (i.e., zero). This cannot be done in JACMAT since a division by zero would occur. For a constant boundary condition at the i th node, the value of PW is set to one for the diagonal term and zero for all other terms of the i th equation.

3. NUITSL

Subroutine NUITSL solves the system of equations for the quasi-Newton iterates. In this analysis the system is solved using a successive over-relaxation (SOR) method. In this work, the optimum amount of over-relaxation was not determined. Since no effort was made to find the optimum,

it was felt a small over-relaxation would be best. The over-relaxation factor of 0.02 was used. For small values of this factor, the SOR method approaches the Gauss-Siedel iteration technique.

VII. PROCEDURE

In this section, the method utilized to obtain a solution is described. The input data necessary to run the developed computer program will be documented.

Prior to initiating a transient overpower excursion, the steady-state conditions for the fuel cell must be known. Since the system of equations which were developed are not specifically designed to obtain a steady-state solution, the initial steady-state conditions must be part of the input data. The initial temperature distribution was obtained from the steady-state conditions given in Ref. [7].

The axial temperature distribution for the fuel center-line, fuel surface, clad, and coolant have been determined for several different fuel life cycles [7]. For this analysis, the beginning of life cycle for channel 10 was used. Although this distribution is somewhat artificial, it should be adequate for this analysis. It is the trends of the results which are considered important. The distribution within the fuel radially is taken to vary as the square of the radial distance; then

$$T_F(r,z,0) = T_F(0,z,0)\left(1 - \frac{r^2}{a^2}\right) + T_F(a,z,0)\left(\frac{r^2}{a^2}\right)$$

Within the cladding and the coolant, the initial radial temperature distribution is assumed to be constant.

The initial flux distribution is assumed to be radially constant, a flat flux assumption. In the axial direction the flux is assumed to vary as the shape of the sine function. The maximum flux, the flux at the axial center, is an input parameter. For this analysis, the maximum initial flux was taken to be 10^{14} neutron/cm²sec.

To obtain a steady state flux distribution, the value of fission cross section for the fuel is varied. A trial-and-error method is used until a critical fission cross section, Σ_{ff}^{cr} , which gives a steady flux is obtained.

Once the steady-state conditions have been determined, the excess reactivity may be inserted. This starts the transient overpower excursion.

A. INPUT DATA

The first data card contains: the order of Gauss quadrature, the number of radial elements in the fuel, the number of axial elements, number of nodal points in the radial direction, and the height of the fuel rod. The next cards, one for each radial nodal point, contain the nodal radial distances. The next cards contain the fuel centerline, fuel surface, clad and coolant temperatures. There is one card for each axial node. The next card contains the maximum flux. The next four cards contain the physical parameters listed in Table I. The last input cards contain the time, end time, estimated initial time step, minimum time step, and maximum time step. A sample data deck is shown in figure 7.

TABLE I. Physical Parameters

Fuel Diffusion Coefficient (DCF)	0.93 [cm]
Doppler constant (B)	0.006
Energy release per fission (E)	7.652×10^{-12} [cal/fissions]
Fraction of delayed neutrons (BETA)	0.0064
Fraction of delayed neutrons for the ith group (BETAI)	0.0064
Decay constant for ith delayed neutron group (DCLAMI)	0.0784
Initial flux for delayed neutrons	1×10^{10} [neutron/cm ² sec]
Average number of neutrons released per fission (ANU)	2.44
Neutron velocity (VEL)	4.8×10^8 [cm/sec]
Fuel absorption cross section (SIGAF)	0.088 [cm ⁻¹]
Critical fuel fission cross section (SIGFF)	0.0586875 [cm ⁻¹]
Blanket absorption cross section (SIGAB)	0.0 [cm ⁻¹]
Blanket fission cross section (SIGFB)	0.0 [cm ⁻¹]
Step reactivity input (RHOA)	Variable
Ramp reactivity input (RHOB)	Variable
Fuel density (DENF)	10.9 [gm/cm ³]
Clad diffusion coefficient (DCC)	1.1 [cm]
Clad specific heat (CPC)	0.12 [cal/gm °C]
Clad density (DENC)	8.0 [gm/cm ³]
Clad thermal conductivity (TKC)	0.0526 [cal/cm sec °C]
Clad absorption cross section (SIGAC)	0.0015 [cm ⁻¹]
Coolant diffusion coefficient (DCCO)	1.55 [cm]
Coolant absorption cross section (SIGACO)	0.00004 [cm ⁻¹]
Coolant flow velocity (VCO)	396.0 [cm/sec]
Surface heat transfer coefficient(HSURF)	0.7 [cal/cm ² sec °C]

VIII. RESULTS

When using the finite element method, one of the first considerations must be given to the convergence of the method. To determine convergence, the results for a given point are compared for different finite element discretizations. As shown in figure 8, the results are comparable but no definite claim of convergence can be made. However, for this work, it was felt that these results were adequate. It was not the object of this analysis to arrive at the "final" result; it was the trends and methods that were of interest. Since the 66-element mesh appears to give a fair approximation of the results, the 66-element mesh was used as the discretized domain.

The next item of consideration was the determination of a neutronic steady-state condition. This proved to be a very time consuming task. The fission cross section for the fuel was varied by a trial-and-error method in an attempt to find the critical cross section which would give a steady state. As may be seen in figure 9, a change in cross section of less than one percent significantly affected the state of the problem. It was felt that the critical value was between the values of 0.05875 and 0.058625. Time did not permit investigation for critical value; therefore, it was assumed that the value for the critical fission cross section was half way between the values (i.e., $\Sigma_f^{cr} = 0.0586875$).

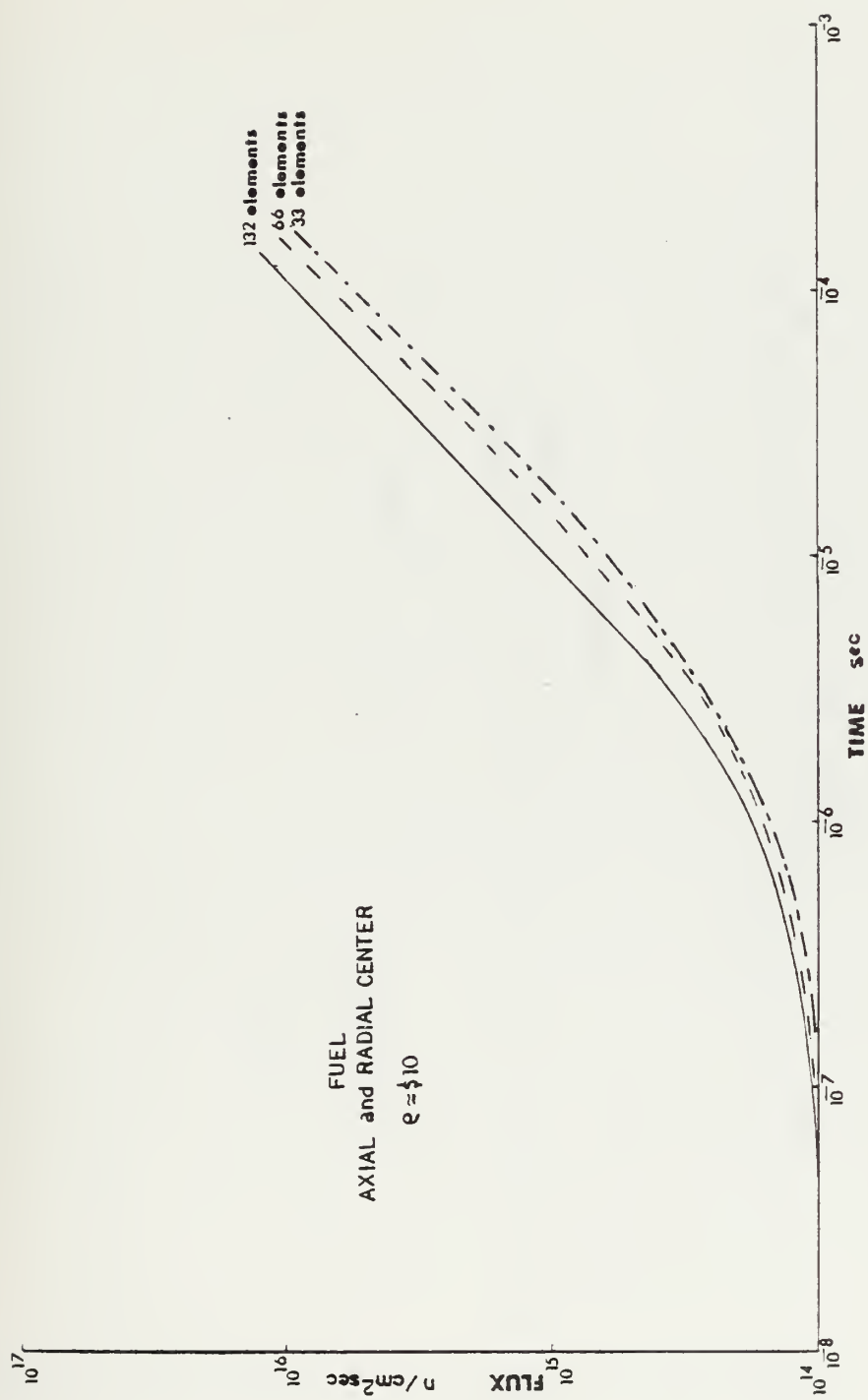


Figure 8. Convergence of Finite Element Method

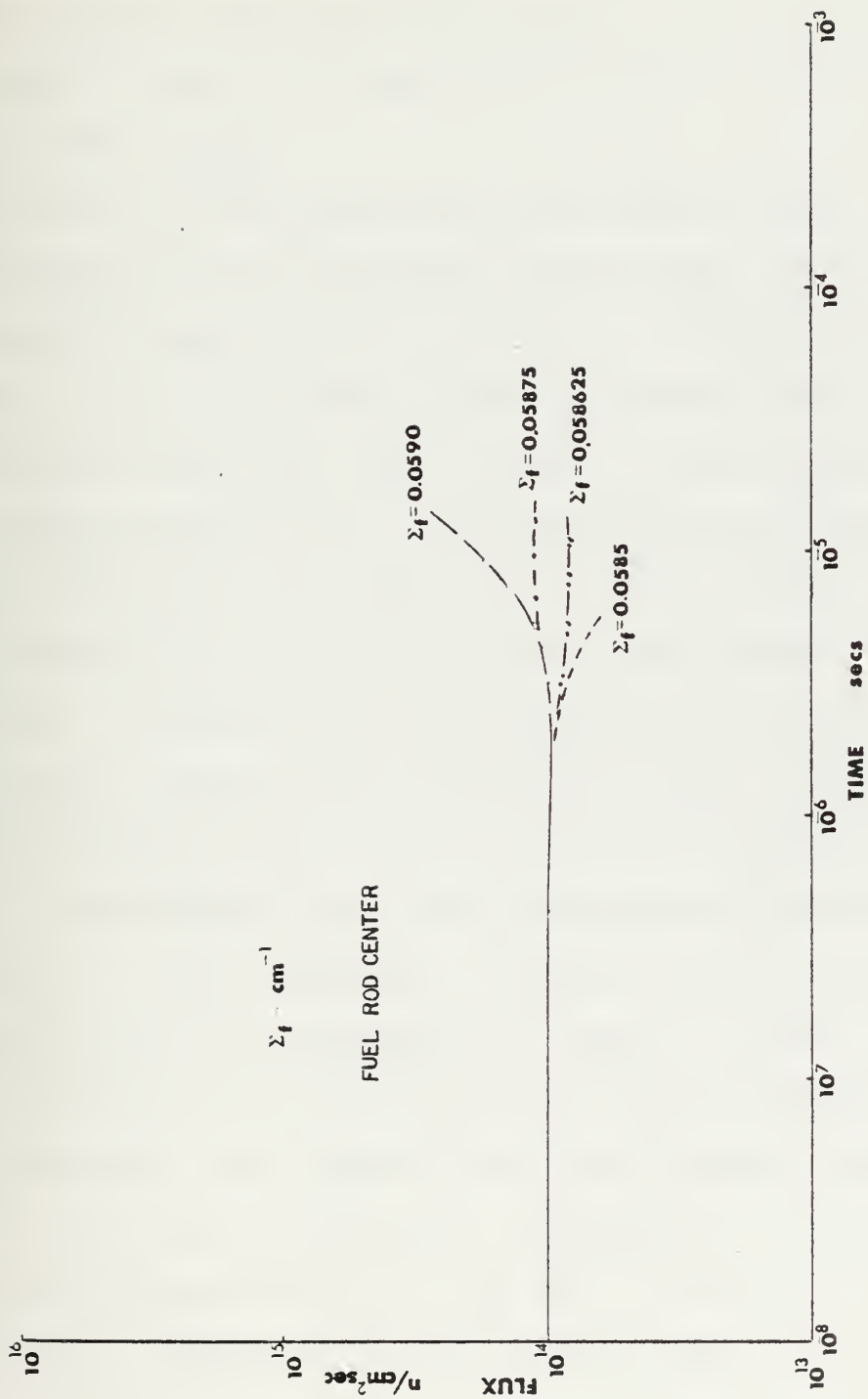


Figure 9. Determination of Critical Fission Cross Section

Even if this value is in error, which it most probably is, the net effect would only be a small decrease or increase in the proposed reactivity insertion. The reactivity insertion would, then, be an approximation of the actual reactivity of the problem.

The first test problem considered was a step increase in reactivity of approximately ten dollars. For the uranium dioxide fuel, one dollar of reactivity was taken to be 0.0064. Figure 10 shows the time history of the flux at the center of the fuel rod. Figure 11 gives the corresponding temperature profile. The temperatures were taken at the hottest point of each region at the axial center (i.e., the fuel centerline, clad inside surface, and coolant inside surface). As seen on the fuel temperature time history, the fuel rapidly reaches the fuel melting point. The model developed does not take into consideration melting of the fuel. This melting would tend to decrease the effect of the transient. The problem was allowed to continue despite this inconsistency in the mathematical model. A short time after fuel melting, the inside surface of the clad reaches its melting point. The temperature in the coolant experienced what is felt to be a numerical phenomenon. The coolant temperature decreased prior to the small rise at the end of the transient. Intuitively, this decrease does not seem to be realistic. A similar occurrence was observed while conducting sample tests on the developed computer program. Reference [20] reported the same phenomenon. It is felt that this

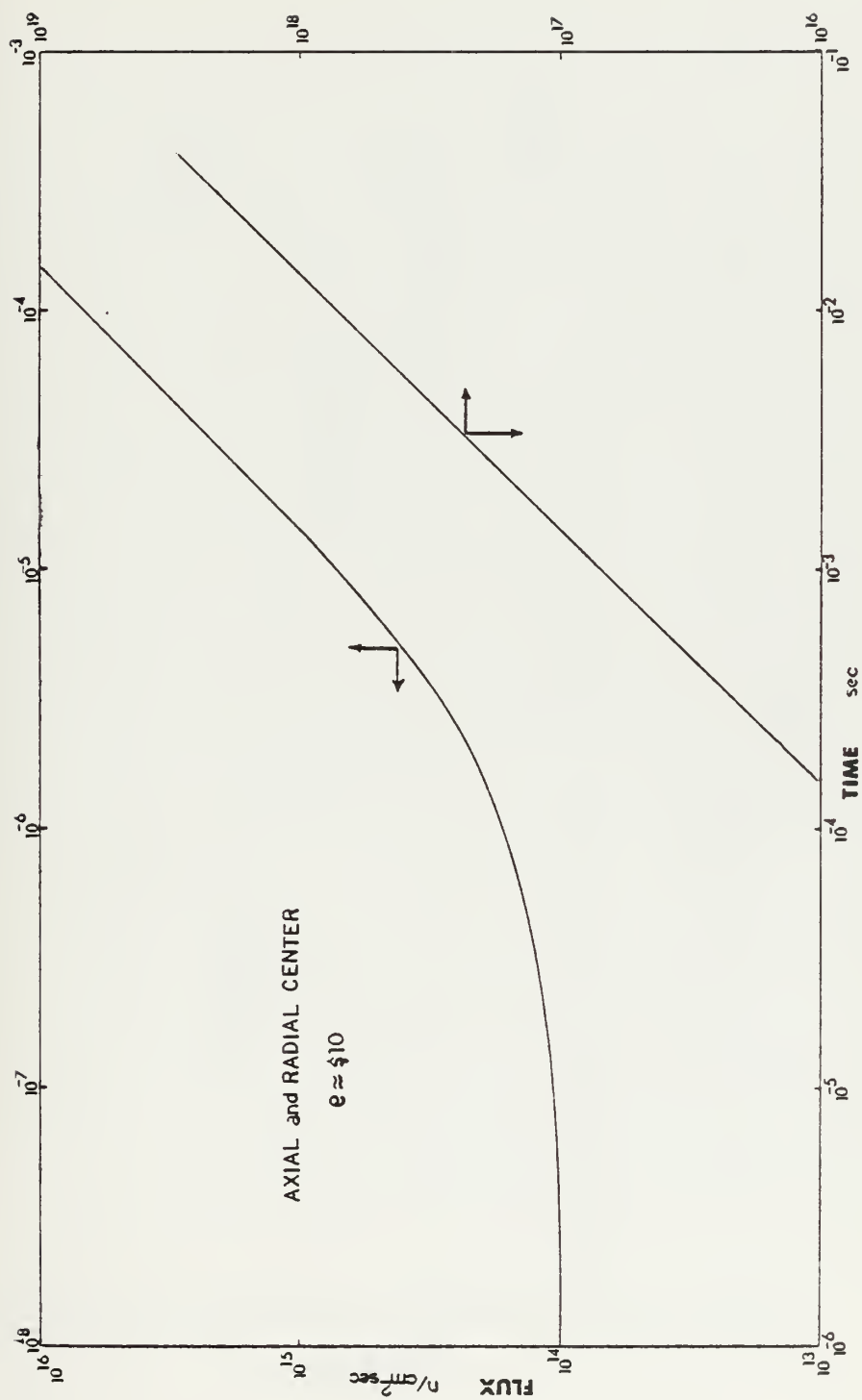


Figure 10. Flux Profile

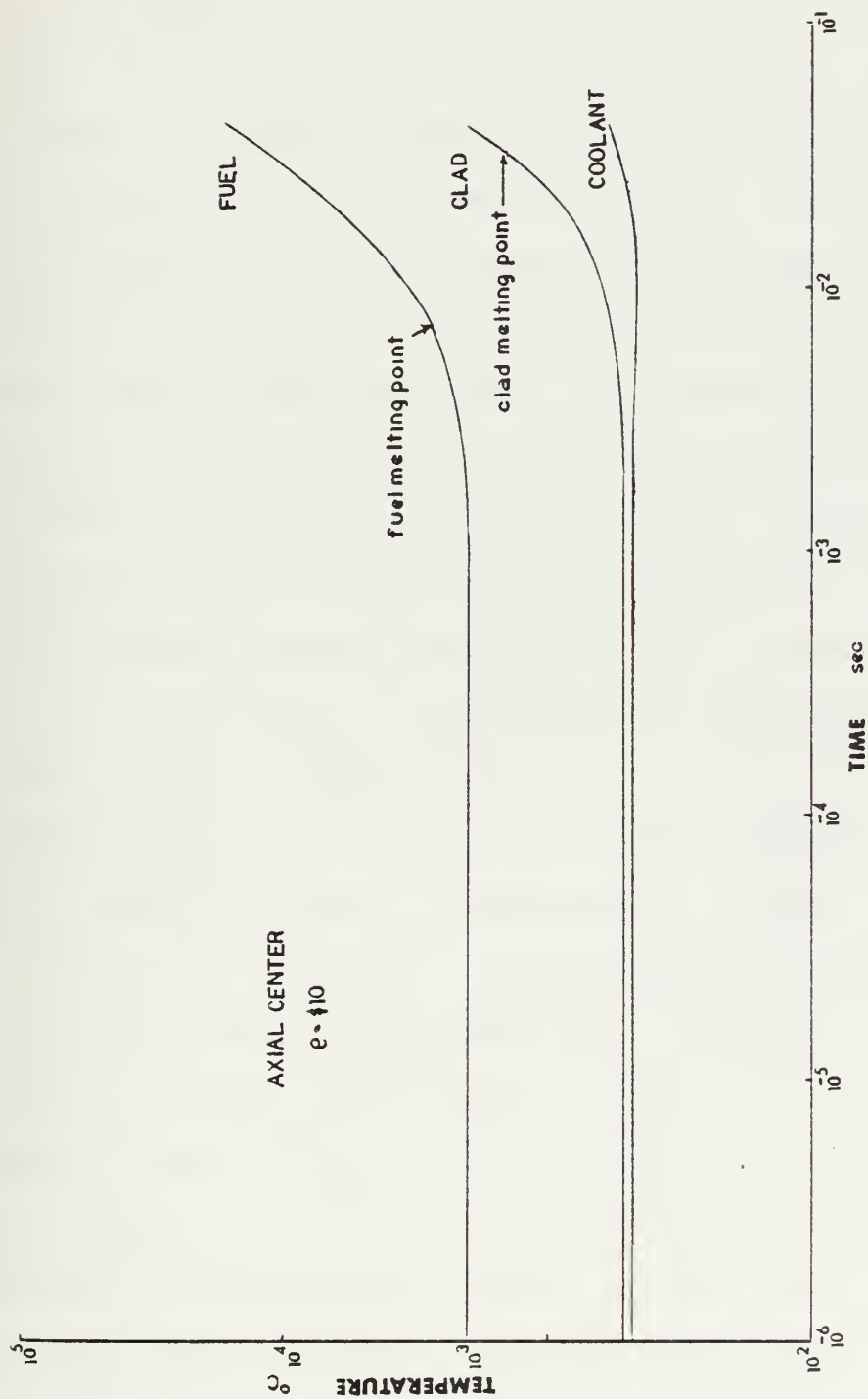


Figure 11. Temperature Profile

phenomenon is a quirk of the finite element method. The radial and axial distributions of the neutron flux are presented in figures 12 and 13 for time equal to 7.39×10^{-3} seconds. The distributions are, basically, as anticipated. The neutron flux peaks slightly before the axial center. It was expected to peak at the axial center. The radial and axial temperature distributions for the same time are presented in figures 14 and 15. As with the flux, the temperature profiles were, basically, as expected. The fuel temperature peaks slightly below the expected location, most likely in response to the peak in axial fluxes. The coolant unexpectedly drops near the outlet of the fuel rod. The finite element method characteristically has some problems on the boundaries of the domain; this may account for the drop in coolant temperature.

The temperature of the fuel and cladding do not appear to be as closely coupled as anticipated. As seen in figure 14, a significant increase in fuel temperature has resulted in a relatively small clad temperature increase. As noted in figure 11, there appears to be a time lag in temperature response for each region which may account for part of the apparent temperature disagreement. It is felt that the temperatures should be more closely related, which indicates a higher gap heat transfer coefficient should be utilized. Since values of the gap heat transfer coefficient were assumed, it is not unreasonable to believe the values used are too low.

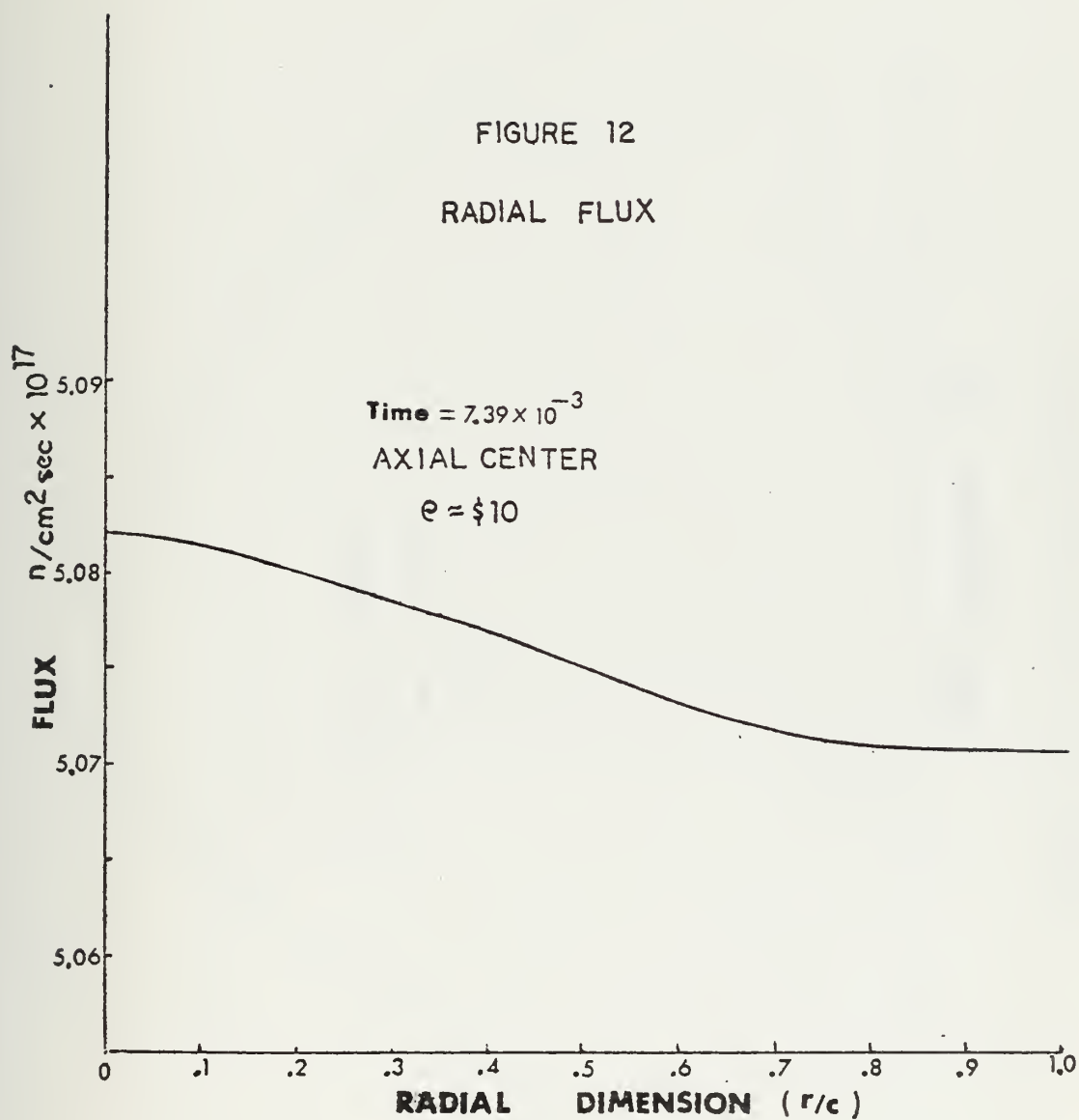


Figure 12. Radial Flux

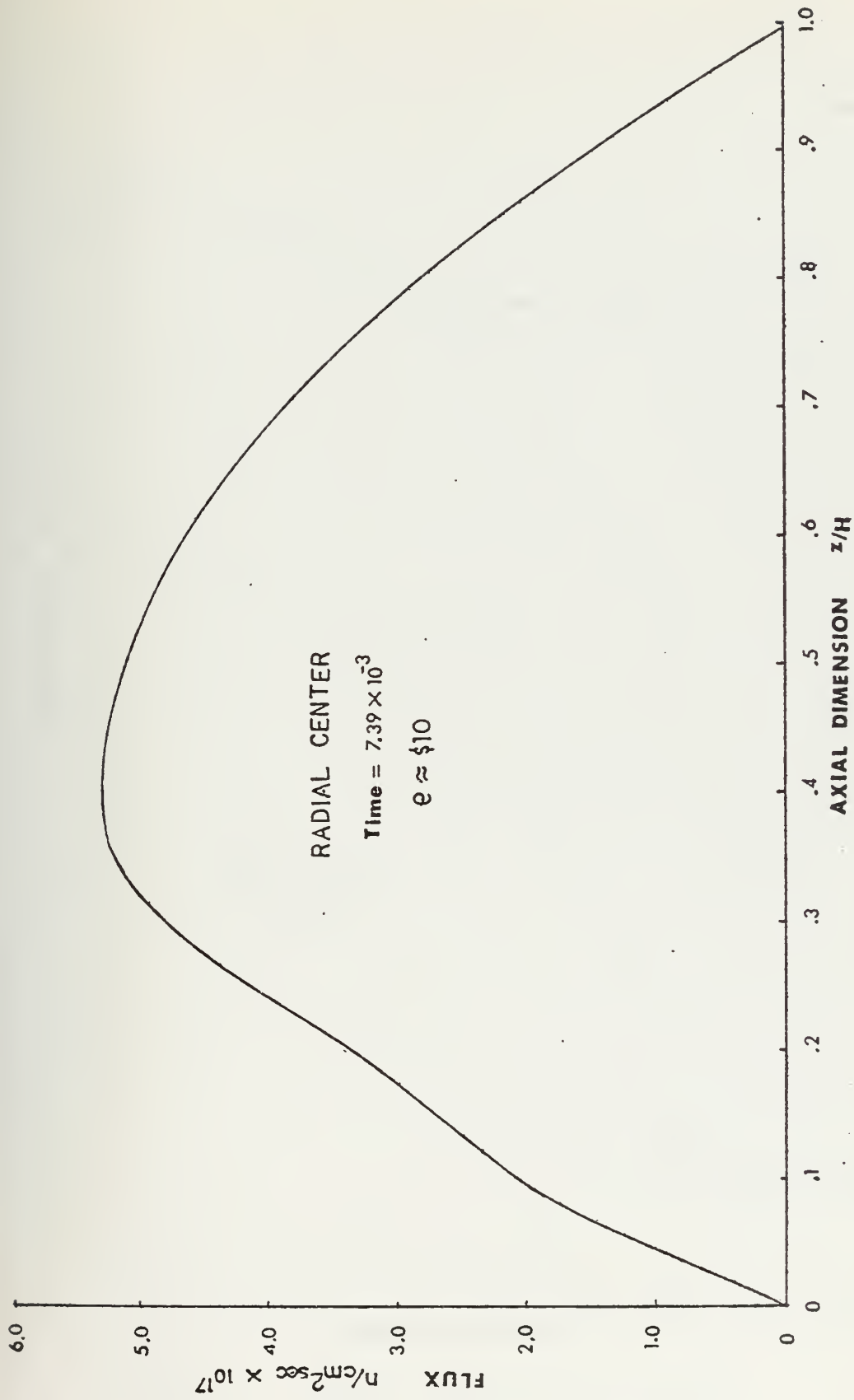


Figure 13. Axial Flux Profile

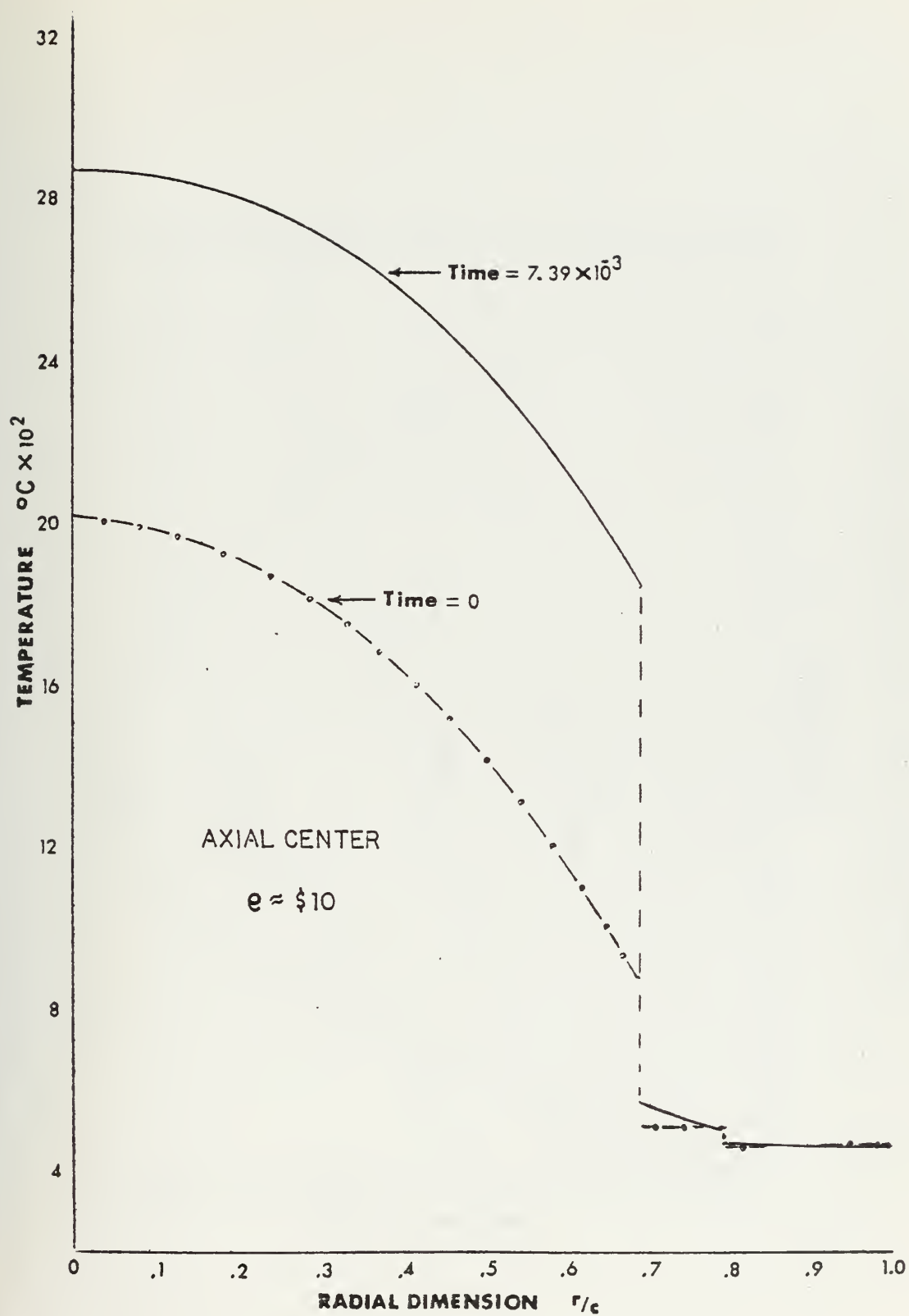


Figure 14. Radial Temperature Profile

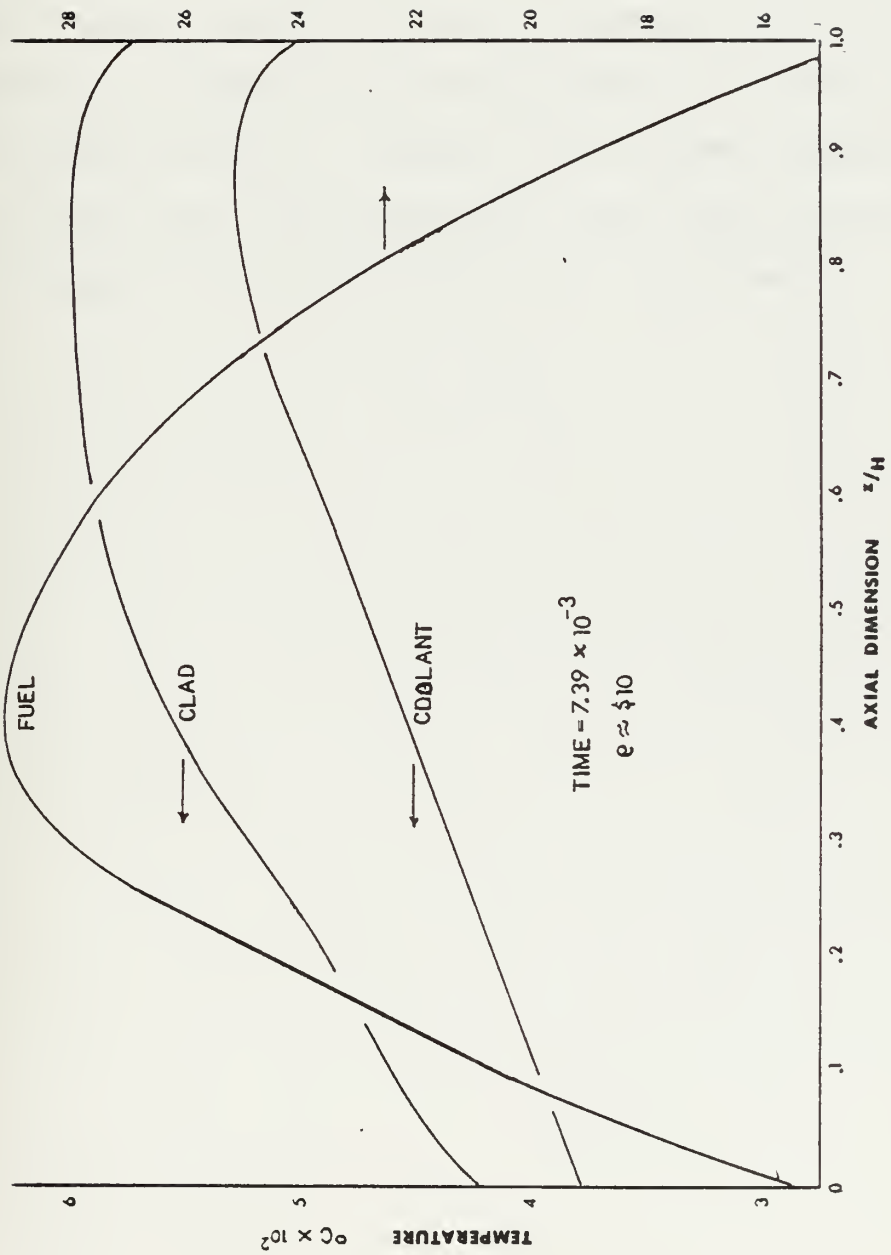


Figure 15. Axial Temperature Profile

Time did not permit investigation of other reactivity insertions. Other reactivity inputs may be investigated by students in the future.

This work does not represent a solution to the very complicated nuclear reactor problem. It does represent an application of a numerical technique which is relatively new to nuclear applications. Methods for implementing the finite element method have been discussed, and a computer code has been developed for the simplistic model considered.

IX. RECOMMENDATIONS

For the model developed, perhaps the most important item to pursue is the critical fission cross section. A better determination of this value is necessary so that the reactivity insertion is more accurately known. Different test cases for the prompt critical and prompt subcritical reactor could then be conducted.

In further developing the model, more consideration should be given the gap heat transfer coefficient. As noted in the results, the value used appears to be too small. Sample problems for different gap heat transfer coefficients would give a better indication of the values to use.

Melting of the fuel during the transient would probably be the next major improvement on the model. With relatively few changes, the model could be adapted to allow melting element by element. This, too, would be an approximation but, still, an improvement to the model. Perhaps at the same time, a simplified model to take into consideration the fuel restructuring could be implemented.

Another improvement would be to consider reactivity feedbacks in addition to the Doppler feedback. Sodium voiding and fuel rod expansion are two of the more important feedback effects to consider.

On the numerical side, probably the most important thing to do would be to run the computer program on the

"H-compiler", which optimizes the program. However, on several runs using the H-compiler, erroneous results were obtained. With sufficient time, this could be corrected to allow use of the H-compiler. The use of the H-compiler results in a savings in computer time. The present program runs on a "G-compiler" and takes excessive amounts of computer time (two to four hours per run).

In addition to this, the optimum over-relaxation factor in the implicit Gear's method could be determined by trial-and-error.

Implementation of these recommendations should enhance the analysis and lead to a more efficient computer code.

APPENDIX A
DEVELOPMENT OF TRANSFORMATIONS

The Jacobian matrix $[J]$ may be written (8) for two dimensions as

$$[J] = \begin{bmatrix} \sum_{i=1}^N N_{i,\xi} r_i & \sum_{i=1}^N N_{i,\xi} z_i \\ \sum_{i=1}^N N_{i,\eta} r_i & \sum_{i=1}^N N_{i,\eta} z_i \end{bmatrix} \quad (A1)$$

For a simple 2x2 matrix $[A]$ the inverse is

$$[A] = \begin{bmatrix} a & b \\ c & d \end{bmatrix}$$

$$[A]^{-1} = \frac{1}{\det[A]} \begin{bmatrix} d & -b \\ -c & a \end{bmatrix}$$

Applying this fact to equation (A1) gives

$$[J]^{-1} = \frac{1}{\det[J]} \begin{bmatrix} \sum_{i=1}^N N_{i,\eta} z_i & - \sum_{i=1}^N N_{i,\xi} z_i \\ \sum_{i=1}^N N_{i,\eta} r_i & \sum_{i=1}^N N_{i,\xi} r_i \end{bmatrix} \quad (A2)$$

$$= \begin{bmatrix} J_{11}^* & J_{12}^* \\ J_{21}^* & J_{22}^* \end{bmatrix}$$

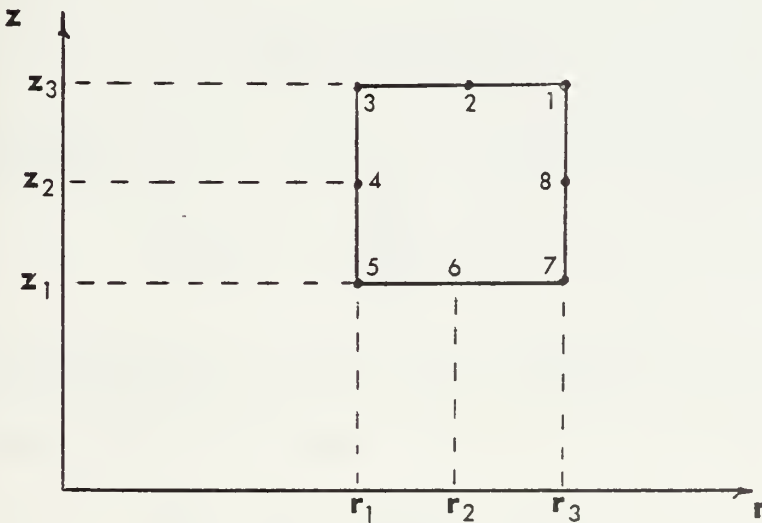
From matrix algebra

$$\det[J] = \sum_{i=1}^N N_{i,\xi} r_i \sum_{i=1}^N N_{i,\eta} z_i - \sum_{i=1}^N N_{i,\eta} r_i \sum_{i=1}^N N_{i,\xi} z_i \quad (A3)$$

The derivatives of the shape functions may be found from equations (38).

$$\begin{aligned} N_{1,\xi} &= \frac{1}{4}(1+\eta)(2\xi+\eta) & N_{1,\eta} &= \frac{1}{4}(1+\xi)(2\eta+\xi) \\ N_{2,\xi} &= -\xi(1+\eta) & N_{2,\eta} &= \frac{1}{2}(1-\xi^2) \\ N_{3,\xi} &= \frac{1}{4}(1+\eta)(2\xi-\eta) & N_{3,\eta} &= \frac{1}{4}(1-\xi)(2\eta-\xi) \\ N_{4,\xi} &= -\frac{1}{2}(1-\eta^2) & N_{4,\eta} &= -\eta(1-\xi) \\ N_{5,\xi} &= \frac{1}{4}(1-\eta)(2\xi+\eta) & N_{5,\eta} &= \frac{1}{4}(1-\xi)(2\eta+\xi) \\ N_{6,\xi} &= -\xi(1-\eta) & N_{6,\eta} &= -\frac{1}{2}(1-\xi^2) \\ N_{7,\xi} &= \frac{1}{4}(1-\eta)(2\xi-\eta) & N_{7,\eta} &= \frac{1}{4}(1+\xi)(2\eta-\xi) \\ N_{8,\xi} &= \frac{1}{2}(1-\eta^2) & N_{8,\eta} &= -\eta(1+\xi) \end{aligned} \quad (A4)$$

If one now considers an arbitrary element



with the midside nodes exactly at the midpoint (not a necessary criteria for the FEM) and substitutes into equation (A3),

the result is

$$\det[J] = \frac{(z_3 - z_1)(r_3 - r_1)}{4} = \frac{A^e}{4} \quad (A5)$$

Substituting into equation (A2) and using (A5) will yield

$$J_{11}^* = \frac{4}{A^e} \left[\frac{1}{2}(z_3 - z_1) \right] = 2/r_3 - r_1, \quad (A6)$$

$$J_{12}^* = 0, \quad (A7)$$

$$J_{21}^* = 0, \quad (A8)$$

and

$$J_{22}^* = \frac{4}{A^e} \left[\frac{1}{2}(r_3 - r_1) \right] = 2/z_3 - z_1. \quad (A9)$$

The inverse of the Jacobian matrix now becomes

$$[J]^{-1} = \begin{bmatrix} 2/r_3 - r_1 & 0 \\ 0 & 2/z_3 - z_1 \end{bmatrix} \quad (A10)$$

For integration along a line, the transformation used is

$$dz = \det[J'] d\eta \quad (A11)$$

In this case

$$\det[J'] = \sum_{i=1}^N N_{i,\eta} z_i \quad (A12)$$

Again considering the arbitrary element and substituting (A4) into (A12) will result in

$$\det[J'] = \frac{z_3 - z_1}{2} = \frac{L^e}{2} \quad (A13)$$

APPENDIX B

REDUCTION OF SECOND ORDER TERM

The second order term of the governing field equations may be reduced to first order by integration by parts.

Consider, for example,

$$\iint_{r,z} N_i \left[\frac{1}{r} \frac{\partial}{\partial r} (rD \frac{\partial \psi}{\partial r}) + \frac{\partial}{\partial z} (D \frac{\partial \psi}{\partial z}) \right] r dr dz \quad (B1)$$

which may be expanded as

$$\iint_{r,z} [N_i r D \frac{\partial^2 \psi}{\partial r^2} + N_i r \frac{\partial D}{\partial r} \frac{\partial \psi}{\partial r} + N_i D \frac{\partial \psi}{\partial r} + N_i r D \frac{\partial^2 \psi}{\partial z^2} + N_i r \frac{\partial D}{\partial z} \frac{\partial \psi}{\partial z}] dr dz \quad (B2)$$

Integrating just the second order terms by parts will yield

$$\begin{aligned} \iint_{r,z} N_i D \frac{\partial^2 \psi}{\partial r^2} r dr dz &= \int_z [N_i r D \frac{\partial \psi}{\partial r}]_r dz \\ &\quad - \iint_{r,z} \frac{\partial \psi}{\partial r} [D N_i + r N_i \frac{\partial D}{\partial r} + r D \frac{\partial N_i}{\partial r}] dr dz \quad (B3) \end{aligned}$$

and

$$\iint_{r,z} N_i D \frac{\partial^2 \psi}{\partial z^2} r dr dz = \int_r [N_i D \frac{\partial \psi}{\partial z}]_z r dr - \iint_{r,z} \frac{\partial \psi}{\partial z} [\frac{\partial N_i}{\partial z} D + N_i \frac{\partial D}{\partial z}] r dr dz \quad (B4)$$

Substituting the results of (B3) and (B4) into equation (B2) will give

$$\int_z [N_i r D \frac{\partial \psi}{\partial r}]_r dz + \int_r [N_i D \frac{\partial \psi}{\partial z}]_z r dr - \iint_{r,z} D [\frac{\partial N_i}{\partial r} \frac{\partial \psi}{\partial r} + \frac{\partial N_i}{\partial z} \frac{\partial \psi}{\partial z}] r dr dz \quad (B5)$$

APPENDIX C

LIST OF RELATIONS FOR MATERIAL THERMAL PROPERTIES

A. FUEL (UO_2)

1. Specific Heat, Ref. [19]

$$C_{pF} = [18.45 + 2.431 \times 10^{-3}T - 2.272 \times 10^{-5}T^2] / 270.07$$

[cal/gm °C]

T - °C

2. Thermal Conductivity, Ref. [5]

$$Tk_F = [1 - 2.5(1 - \rho_{TD})] \times \left[\frac{45.1}{135 + T} + 4.79 \times 10^{-13}T^3 \right] \times 0.239$$

[cal/cm sec °C]

ρ_{TD} - percent theoretical density

T - °K

B. CLAD (Stainless Steel)

Properties are assumed to be temperature independent, and average values from Ref. [5] were used for the clad properties.

C. COOLANT (Liquid Sodium)

1. Specific Heat, Ref. [5]

$$C_{p_{Co}} = 0.34574 - 0.79226 \times 10^{-4}T + 0.34086 \times 10^{-7}T^2$$

[cal/gm °C]

T - °F

2. Density, Ref. [5]

$$\rho_{Co} = [59.566 - 7.9504 \times 10^{-3}T - 0.2872 \times 10^{-6}T^2 + 0.06035 \times 10^{-9}T^3] \times 0.01601$$

[gm/cm³]

T - °F

3. Thermal Conductivity, Ref. [5]

$$Tk_{co} = [54.306 - 1.878 \times 10^{-2}T + 2.0914 \times 10^{-6}T^2] \\ \times 4.134 \times 10^{-3} \quad [cal/cm \text{ sec } ^\circ C]$$

$$T - ^\circ F$$

D. SURFACE HEAT-TRANSFER COEFFICIENT

$$h_{surf} = \frac{Tk_{co}}{De} [7.0 + 0.025 \left(\frac{De V_{co} \rho_{co} C_{p_{co}}}{Tk_{co}} \right)^{0.8}] \\ [BTU/hr \text{ ft}^2 \text{ } ^\circ F]$$

$$Tk_{co} - [BTU/hr \text{ ft } ^\circ F]$$

$$\rho_{co} - [lbm/ft^3]$$

$$V_{co} - [ft/sec]$$

$$C_{p_{co}} - [BTU/lbm \text{ } ^\circ F]$$

$$De - \text{equivalent diameter [ft]}$$


```

5C CCNTINLE (NELCCN)
CALL MATK1 (NELCCN)
CC 100 MM= 1,NEL
CALL JACOB (MM,NELCON,R,Z)
CALL MATH12 (MM,NELCON,R,Z)
CALL SYSH12 (MM,BIGH12,BIGH3,BIGH5,BIGF,JA,JB,NAME,NELCCN)
IF(NELCCN(11,MM).EQ. C) GO TO 100
CALL SYSK12 (MM,JA,JB,NAME,NELCON,R,Z,BIGK)
CCNTINLE
1CC *** INPUT FUEL PHYSICAL CONSTANTS
REAC(5,15) DCF,B,E,BETA,BETAI,DCLAMI,FFLUX0,ANU,VEL,SIGAF,SIGFF,
SIGAB,SIGFB,RHCA,RHCB,DENF
1 AM INF = ANU*SIGFF/SIGAF
C *** INPUT CLAD PHYSICAL CONSTANTS
REAC(5,15) DCC,CPC,DENC,TKC,SIGAC
C *** INPUT COLANT PHYSICAL CONSTANTS
REAC(5,15) DCCO,SIGACC,VCO,HSURF
WRITE(6,40) DCF,B,E,BETA,BETAI,DCLAMI,ANU,AKINF
WRITE(6,41) DCF,B,E,BETA,BETAI,DCLAMI,ANU,AKINF
WRITE(6,42) FFLUX0,VEL,SIGAF,SIGFF,SIGAB,SIGFB,SIGAC,SIGACC
WRITE(6,43) DCC,CPC,DENC,TKC,DCCC,VCO,HSURF,DENF
WRITE(6,44) RHOA, RHOB
JSKF = C
NY = NP*2
NL = 0
M = NY
EPS = 0.01
T,TEND,F,HMIN,HMAX
REAC(5,15) T,TEND,F,HMIN,HMAX
200 CALL SDESOL (Y,YL,T,TEND,NY,NL,M,JSKF,6,1,F,HMIN,HMAX,EFS,W,
1 BIGH12,BIGH3,BIGH5,BIGK,BIGF,IBP,ITYPE,JA,JB,NAME,NELCCN,F,Z,YIT,
2 BIGH4)
WRITE(6,11) JSKF
STCP
ENC

```



```

SUBROUTINE GINNY (ITYPE,NELCCN,NSTART,R,Z)
  SUBROUTINE GINNY DEVELOPES THE SYSTEM MESH
  IT ALLOWS FOR ONE OR TWO RADIAL ELEMENTS IN THE FUEL
  AND ONE ELEMENT IN THE CLAD AND COCLANT
  AND SETS THE NUMBER OF RADIAL FUEL ELEMENTS
  THE GENERATOR ALLOWS FOR ANY NUMBER OF AXIAL ELEMENTS DESIRED
  NUMBER OF AXIAL ELEMENTS SET BY NEZ
  THE GENERATOR WILL GIVE THE R AND Z DIMENSION FOR EACH NCCE
  IT ALSO DEVELOPES THE CONNECTIVE MATRIX

  ITYPE -- AN ARRAY USED TO INDICATE THE TYPE OF NCCE
           0 -- FUEL NODE NOT IN AN INTERFACE ELEMENT
           1 -- FUEL NCCE IN AN INTERFACE ELEMENT
           2 -- CLADDING NODE
           3 -- COCLANT NCCE
  NELCON -- THE CONNECTIVE MATRIX
  R -- RADIAL DIMENSION OF THE ITH NODE
  Z -- AXIAL DIMENSION OF THE ITH NCCE

  IMPLICIT INTEGER*2(I-N)
  DIMENSION DR(13)
  DIMENSION ITYPE(1)
  DIMENSION NELCON(11,1)
  DIMENSION NSTART(1)
  DIMENSION R(1), Z(1)
  COMMON/CCNN/ NP,NEL,JC
  COMMON/INP/DR,HEIGHT,NPR,NEZ,NFE
  LL = 0
  NF = 0
  NK = 1
  MN = 2*NFE + 1
  MM = M + 4
  MN = M + 2
  NEL = NFE + 2
  NPZ = NEZ*2
  NACC = 3*NFE + 14
  EZ = NEZ
  ZI = HEIGHT/(2.0*EZ)
  CC 100 I = 1,NPZ
  T = 1
  IF (KK.EQ.1) GO TO 80
  LL = LL + 1
  NSTART(LL) = NP + 1
  CC 70 J = 1,NPR
  NP = NP + 1
  Z(NP) = ZI*T
  R(NP) = DR(J)

```

6C


```

70 KK = 1
   CCNT INUE
   CC TO 100
   EC 50 JJ = 1, NPR, 2
   IF (JJ.EQ. MMM) GO TO 50
   NF = NP + 1
   Z(NP) = Z1*1
   R(NP) = DR(JJ)
   IF ((JJ.EQ. M) .CR. (JJ.EQ. MM)) GO TO 85
   GC TO 9C
   NF = NP + 1
   Z(NP) = Z1*1
   R(NP) = DR(JJ)
   KK = 0
   CCNT INUE
   5C CCNT INUE
   1CC CC 125 J = 1, NPR
   NP = NP + 1
   Z(NP) = 0
   R(NP) = DR(J)
   125 CCNT INUE
   NNEZ = NEZ - 1
   IF (NNEZ.EQ.0) GO TO 151
   CC 150 LL = 1, NNEZ
   NST = NSTART(LL)
   NCUT = NOUT + M + 1
   NCHA = NOUT - 2
   NENDC = NST + NPR - 3
   CC 150 AN = NST, NENDC, 2
   IF (NK.EG. NCHA) GC TO 150
   NNA = NN
   NEL = NEL + 1
   NCCNT = NCCNT + 1
   IF (NN.EQ. NOUT) NNA = NN - 1
   NELCON(1, NEL) = NNN + NADD
   NELCCN(1, NEL) = NELCCN(1, NEL) - 1
   NELCON(2, NEL) = NELCON(2, NEL) - 1
   NELCON(3, NEL) = NNN + NPR + 1 - ACCOUNT
   NELCON(4, NEL) = NNN
   NELCON(5, NEL) = NNN + 1
   NELCON(6, NEL) = NNN + 2
   NELCON(7, NEL) = NELCCN(4, NEL) + 1
   CCNT INUE
   15C NST = NSTART(1)
   151 NENDC = NST + NPR - 3
   NCUT = NCUT + M + 1
   NCHA = NCUT - 2

```



```

N = 0
CC 153 I = AST,NEND,2
II = 0
IF (I .EQ. NOUT) II=1
IF (I .EQ. NCTA) GC TO 153
N = N + 1
NELCCN(1,N) = I + 2 - II
NELCCN(2,N) = I + 1 - II
NELCCN(3,N) = I - II
NELCCN(4,N) = I - NFE - 4 - N - II
IF (I .EQ. NEND) II=2
NELCCN(5,N) = NP - NPR + 2*N - 1 + II
NELCCN(6,N) = NELCCN(5,N) + 1
NELCCN(7,N) = NELCCN(5,N) + 2
NELCCN(8,N) = NELCCN(4,N) + 1
CCNTINUE + 2
153 CC 155 I = 1,NFE
IF (I .EQ. NFE) GO TO 156
CC 155 J = I,NEL,M
NELCCN(1,J) = 0
CCNTINUE
GC TO 159
155 CC 158 J = I,NEL,M
NELCCN(1,J) = 1
NELCCN(11,J+1) = 2
NELCCN(11,J+2) = 3
CCNTINUE
CC 180 J = 1,NEL
J = NELCCN(11,J)
CC 180 I = 1,8
I = NELCCN(1,J)
I = JJ
CCNTINUE
158 CC 180 WRITE(6,15)
CC 160 I = 1,NP,3
II = I + 1
IF (II .GT. NP) GC TO 161
I2 = I + 2
IF (I2 .GT. NP) GO TO 162
WRITE(6,20) I,R(1),Z(1),I1,R(11),Z(11),I2,F(12),Z(12)
GC TO 165
160 CC 160 WRITE(6,20) I,R(1),Z(1)
GC TO 165
161 CC 160 WRITE(6,20) I,R(1),Z(1),I1,R(11),Z(11)
162 CC 160 WRITE(6,25)
165 CC 170 J = 1,NEL

```



```

17C WRITE (6,30) (J, (NELCCN(I,J), I = 1,8))
15 FCFRMT (10X, 'NODAL COORDINATES', //, 10X, 'NOCDE', 4X, 'R VALUE', 6X, 'Z V
16 VALUE', 10X, 'NODE', 4X, 'R VALUE', 6X, 'Z VALUE', 4X, 'R VALUE',
20 VALUE', //)
25 FCFRMT (10X, 'CONNECTIVITY MATRIX', //, 10X, 'ELEM', 3X, 'NCD1', 3X, 'NCD2
30 VALUE', 10X, 'NCD3', 3X, 'NCD4', 3X, 'NCD5', 3X, 'NCD6', 3X, 'NCD7', 3X, 'NCD8', //)
3C FCFRMT (7X, 9(3X, I4))
RETURN
END

```



```

SLROUTINE UPCOMP (IBP,JA,JB,NAME,NAME,NELCCN,NSTART)
SLROUTINE CFCOMP CALCULATES THE NAME ARRAY,
THE JA ARRAY, AND THE JB ARRAY WHICH
ARE USED IN THE OPTIMUM COMPACTING SCHEME

IBP -- AN ARRAY USED TO STORE THE FUEL-CLAC INTERFACE NODES
JA -- AN ARRAY WHICH INDICATES THE NUMBER OF NODES CONTRIBUTING
TO THE ITH NODE
JB -- THE POINTER ARRAY WHICH INDICATES WHERE THE ITH
EQUATION BEGINS IN NAME
NAME -- TWC-DIMENSIONAL ARRAY USED TO DEVELOP NAME
NAME -- THE ARRAY USED FOR THE OPTIMUM COMPACTING SCHEME
NELCCN -- CONNECTIVE MATRIX
NSTART -- AN ARRAY USED TO STORE CENTERLINE NCCAL FCINTS

IMPLICIT INTEGER*2(I-N)
DIMENSION DR(13)
DIMENSION IBP(1)
DIMENSION JA(1), JB(1), NAME(1), NAME(26,1)
DIMENSION NELCON(11,1)
DIMENSION NSTART(1)
COMMON/BCND/ NCCOUNT
COMMON/CCNN/ NP,NEL,JC
COMMON/INP/DR,HEIGHT,NPR,NEZ,NFE
NELCOF = 8
WRITE (6,10) NP,NEL,NELCOF
FCRMAT(/2X,'NUMNP=',I5,5X,'NUMEL=',I5,5X,'NELDOF=',I5)
1C DC 40 I = 1,NP
JA(I)=1
CCNTINUE
CC 6C I = 1,NP
CC 50 J = 1,26
NAME(J,I)=0
CCNTINUE
CC 6C I=1,NP
NAME(1,I)=1
CCNTINUE
CC 100 I = 1,NEL
CC 50 J=1,NELDOF
J = NELCCN(J,I)
CC 80 K=1,NELDOF
IF(K.EQ.J) GC TO 80
KK = NELCCN(K,I)
JA(JJ)=JA(JJ)+1
JJA=JA(JJ)

```

CCCCCCCCCCCCCCCC

C


```

DC 7C L=2,JAA
JL=MAME(L,JJ)
IF(JJL.EQ.KK) JA(JJ)=JA(JJ)-1
IF(JJL.EQ.KK) GO TO 80
IF(JJL.EQ.0) MAME(JAA,JJ)=KK
CCCONTINUE
7C
EC
SO
1CC
C ***
C ***
TC ACCUNT FOR THE INTERFACE CONDITIONS
ACCUNT = 3
IEP(1) = NF - 6
IEP(2) = NFE + 1
NN = NFE + 7
NLM = 2*NFE
WRITE (6,1000) (NSTART(I),I=1,NEZ)
FCRMT (10X,1115)
1000 DC 300 I=1,NEZ
IF(NCONT) = NSTART(I) + NUM
IF(I.EQ. NEZ) GC TO 300
IEP(NCONT + 1) = IEP(NCCUNT) + NN
ACCUNT = NCONT + 2
CCCONTINUE
3CC WRITE (6,1000) (IBP(I),I=1,NCCUNT)
KK = 1
CC 400 I = 1, ACCUNT
JJ = IBP(I)
IF (KK.EQ. 0) GO TO 350
IF (I.EQ. NCCUNT) GO TO 350
IF (I.EQ. 1) GO TO 350
CC 325 J = 1,13
MAME (J+13,JJ) = MAME (J,JJ+1)
MAME (J+13,JJ+1) = MAME (J,JJ)
MAME (J+13,JJ+3) = MAME (J,JJ+4)
MAME (J+13,JJ+4) = MAME (J,JJ+3)
CCCONTINUE
325 JA(JJ) = JA(JJ) + 13
JA (JJ+1) = JA(JJ+1) + 13
JA (JJ+3) = JA(JJ+3) + 13
JA (JJ+4) = JA(JJ+4) + 13
KK = 0
GC TO 400
35C KK = 1
CC 375 J = 1,8
MAME (J+8,JJ) = MAME (J,JJ+1)
MAME (J+8,JJ+1) = MAME (J,JJ)

```



```

375 IF ((I .EQ. 1) .OR. (I .EQ. NEZ1)) JJ = JJ + 1
NAME (J+8, JJ+2) = NAME (J, JJ+3)
NAME (J+8, JJ+3) = NAME (J, JJ+2)
IF ((I .EQ. 1) .OR. (I .EQ. NEZ1)) JJ = JJ - 1
CCNTINUE
JA (JJ) = JA (JJ) + 8
JA (JJ+1) = JA (JJ+1) + 8
IF ((I .EQ. 1) .OR. (I .EQ. NEZ1)) JJ = JJ + 1
JA (JJ+2) = JA (JJ+2) + 8
JA (JJ+3) = JA (JJ+3) + 8
IF (I .EQ. 1) KK = C
CCNTINUE

400 C ***
C ***

JE (1) = 1
JC = C
CC2CO I = 1, NP
JN = JA(I)
JE (I+1) = JB(I) + JA(I)
JC = JC + JA(I)
CCNTINUE
WRITE(7,5) (JA(I), I=1, NP)
WRITE(7,5) (JB(I), I=1, NP), JC
FCRMAT(16I5)
FCRMAT(29I4)
WRITE(6,2I5) JC
FCRMAT(/2X, 'JC=', I10)
DC250 I = 1, NP
JA = JA(I)
JEL = JB(I)
CC 240 J = 1, JAA
J = JBL + J - 1
NAME(JJ) = NAME(J, I)
CCNTINUE
WRITE(7,5) (NAME(I), I=1, JC)
RETURN
ENC

20C C
C
205 C
215 C
240 C
250 C

```



```

A2 = R(NCDES)**2
R2 = R(NODE)**2
Y(1,NODE+NP) = Y(1,NCCEM+NP)*(1.0-R2/A2) + Y(1,NODES+NP)*R2/A2
GC TO 250
23C IF (K.EQ. 4).OR.(K.EQ. 8)) GC TC 240
NCES = NELCCN(7,M+1)
NCDEM = NELCCN(5,M)
A2 = R(NCDES)**2
R2 = R(NCDE)**2
Y(1,NODE+NP) = Y(1,NODEM+NP)*(1.0-R2/A2) + Y(1,NCDES+NP)*R2/A2
GC TO 250
24C NCES = NELCCN(8,M+1)
NCDEM = NELCCN(4,M)
A2 = R(NCDES)**2
R2 = R(NCDE)**2
Y(1,NODE+NP) = Y(1,NODEM+NP)*(1.0-R2/A2) + Y(1,NCDES+NP)*R2/A2
GC TO 250
25C CCNTINUE
30C KK = 1
IF(NFE.EQ. 1) KK = 0
NCES = NELCCN(2,M)
NCDEM = NELCCN(1,M)
A2 = R(NCDES)**2
R2 = R(NCDE)**2
Y(1,NODE+NP) = Y(1,NODEM+NP)*(1.0-R2/A2) + Y(1,NCDES+NP)*R2/A2
GC TO 600
40C NCDE1 = NELCCN(1,M) + NP
NCDE2 = NELCCN(2,M) + NP
NCDE3 = NELCCN(3,M) + NP
NCDE4 = NELCCN(4,M) + NP
NCDE5 = NELCCN(5,M) + NP
NCDE6 = NELCCN(6,M) + NP
NCDE7 = NELCCN(7,M) + NP
NCDE8 = NELCCN(8,M) + NP
Y(1,NODE1) = Y(1,NCDE3)
Y(1,NODE2) = Y(1,NCDE3)
Y(1,NODE6) = Y(1,NCDE5)
Y(1,NODE7) = Y(1,NCDE5)
Y(1,NODE8) = Y(1,NCDE4)
CCNTINUE
60C
C**
C**
C**
INPUT THE MAXIMUM FLUX (I.E. THE FLUX
AT THE AXIAL CENTER) RADIAL FLUX WILL BE ASSUMED

```



```

C**      TC REMAIN CONSTANT (FLAT FLUX ASSUMPTION)
C**      AXIALLY THE FLUX ASSUMED TC VARY AS THE
C**      SHAPE OF THE SINE FUNCTION
      REAC(5,1001) FMAX
      FCRMAT (F10.5)
1001      CC 700 M = 1,NP
      X = Z(M)*3.1415926/HEIGHT
      S = SIN(X)
      Y(1,M) = FMAX*S
700      CCNTINUE
      WRITE (6,1050)
1050      FCRMAT (1,9X,'NODAL',5X,'INITIAL',3X,'INITIAL',/,1CX,'POINT',5X,
1      ,FLUX',6X,'TEMP',//)
      CC 800 I = 1,NP
      WRITE (6,1100) I, Y(1,I),Y(1,I+NP)
800      CCNTINUE
1100      FCRMAT (9X,I4,E13.3,F10.3)
      RETURN
      END

```



```

SLERCUTINE SHAPE
SHAPE EVALUATES THE SHAPE FUNCTION AND
ITS DERIVATIVE WITH RESPECT TO XI AND
WITH RESPECT TO ETA AT EACH OF THE GAUSS POINTS
SLERCUTINE SHAPE ALSO SETS THE WEIGHTS AND LCCATCN
CF THE GAUSS POINTS DEPENDING UPON THE ORDER
CF GAUSSIAN QUADRATURE USED

IMPLICIT INTEGER*2(I-N)
REAL*8 GP,ETA,SN,DESN,DXSN,XI,ETA1,ETA2,XI1,XI2,ETASC,XISC
REAL*8 XW,WT
DIMENSION SN(8,25), DESN(8,25), CXSN(8,25), WT(25), GP(5), XW(5)
COMMON/SHAFUN/ GP,NORD
COMMON/WEIGHT/ WT, DESN, DXSN
NC = NORD-2
N = 0
4 GC TO (4,6,8),NO
GP(1) = 0.7745966652414834
GP(2) = 0.0
GP(3) = -GP(1)
XW(1) = 5.0/9.0
XW(2) = 8.0/9.0
XW(3) = XW(1)
6 GC TO 10
GP(1) = 0.861136311594053
GP(2) = 0.339981043584856
GP(3) = -GP(2)
GP(4) = -GP(1)
XW(1) = 0.347854845137454
XW(2) = 0.652145154862546
XW(3) = XW(2)
XW(4) = XW(1)
8 GC TO 10
GP(1) = 0.906179845938664
GP(2) = 0.538469310105683
GP(3) = 0.0
GP(4) = -GP(2)
GP(5) = -GP(1)
XW(1) = 0.236926885056189
XW(2) = 0.478628670455366
XW(3) = 0.5688888888888889
XW(4) = XW(2)
XW(5) = XW(1)
10 CC 13 I J = 1,NORD
N = 1

```

CCCCCCCC

109


```

C
C
C
C
C
SUBROUTINE MATK1 (NELCCN)
SLBFCUTINE MATK1 CALCULATES THE 8X8 ELEMENT K1
AND K2 MATRICES FOR THE ELEMENTS BECARDERIN
CNE OF THE TWO INTERFACES
IAPLICIT INTEGER*2 (I-N)
REAL*8 WT, GP, ETA, XSN{X,Z,XW,Y
DIMENSION NELCON(5), XSN(5,5), WT(25)
DIMENSION NELCON(11,1)
DIMENSION XK1(8,8), XK2(8,8)
COMMON/CCCN/ NP, NEL, JC
COMMON/ GP/ GP, NORD
COMMON/ KIMAT/ XK1, XK2
COMMON/ WEIGHT/ WT
X1 = WT(1)**0.5
CC 15 I = 1,8
CC 15 J = 1,8
XK1(I,J) = 0.0
XK2(I,J) = 0.0
CCNTINUE
15 CC 30 K = 1, NORD
ETA = GP(K)
XSN(1,K) = 0.5*(ETA+1.0)*ETA
XSN(2,K) = 0.5*(ETA-1.0)*ETA
XSN(3,K) = 1.0 - ETA*ETA
CCNTINUE
30 CC 100 I = 1,3
CC 80 J = 1,3
Z = 0.0
CC 60 K = 1, NORD
X = XSN(I,K)
Y = XSN(J,K)
Z = Z + X*Y*WT(K)/XW
CCNTINUE
60 XK1(I,J) = Z
CCNTINUE
80 CCNTINUE
100 CCNTINUE
XK1(1,7) = XK1(1,2)
XK1(1,8) = XK1(1,3)
XK1(7,7) = XK1(2,2)
XK1(7,8) = XK1(2,3)
XK1(8,8) = XK1(3,3)
XK2(3,3) = XK1(1,1)
XK2(3,4) = XK1(1,3)
XK2(3,5) = XK1(1,2)
XK2(4,4) = XK1(3,2)
XK2(4,5) = XK1(2,3)
120

```



```

14C      XK2(5,5) = XK1(2,2)
      CC 160 I J = 1,3
      CC 160 I J = 2,3
      XK1(I,J) = 0.0
16C      CCATINUE = 1,8
      CC 180 I J = 1,8
      CC 180 I J = XK1(I,J)
      XK1(J,I) = XK2(I,J)
      XK2(J,I) =
      CCATINUE
18C      RETURN
      ENC

```



```

SLERCUTINE MATH12 (M,NELCON,R,Z)
SLERCUTINE MATH12 CALCULATES THE 8X8 ELEMENT11 H12,
F3 AND H5 MATRICES AND CALCULATES THE 8X1 VECTOR F
F3 IS STORED IN THE H4 ARRAY TO SAVE CORE STORAGE

IMPLICIT INTEGER*2(I-N)
REAL*8 SN,DESN,DXSN,STJ11,STJ22,CETJ,WT,ZX,ZY,ZZ,X,Y,XX,YY,RR
REAL*8 XXX,YYY,FF,RC,GP
DIMENSION GP(5)
DIMENSION H12(8,8),H3(8,8),F(8)
DIMENSION F4(8,8)
DIMENSION NELCON(11,1)
DIMENSION R(1),Z(1)
DIMENSION SN(8,25), DESN(8,25), CXSN(8,25), WT(25)
DIMENSION STJ1(25), STJ2(25)
COMMON/CGN/ NP,NEL,JC
COMMON/CP/ GP,NORD
COMMON/JACINV/ STJ11,STJ22,CETJ
COMMON/MATRIX/ H12,F3,F
COMMON/MTRXH4/ H4
COMMON/SHAFUN/SN, DESN, DXSN
COMMON/WEIGHT/ WT
NC = NCRD*NORD
LE = NELCON(11,M)
CC 100 I = 1,8
FF = 0.0
CC 80 J = 1,8
ZY = 0.0
ZZ = 0.0
ZZZ = 0.0
CC 60 K = 1,NO
X = DXSN(I,K)*STJ11(K)
XX = DXSN(J,K)*STJ11(K)
YY = DESN(J,K)*STJ22(K)
XXX = SN(I,K)
YYY = SN(J,K)
RR = 0.0
CC 40 L = 1,8
LL = NELCON(L,M)
RC = R(LL)
RR = RR + SN(L,K)*RD
CCATINUE X*Y*RR*WT(K)
ZZ = ZX + XX*Y*RR*WT(K)
ZZ = ZZ + XX*Y*RR*WT(K)

```

40


```

ZZZ = ZZZ + XXX*YYY*RR*WT(K)
IF(J.NE.1) GO TO 60
IF((LE.EQ.2).OR.(LE.EQ.3)) GO TO 60
FF = FF + XXX*RR*WT(K)
6C CCNTINUE = (ZX + ZZ)*DETJ
F12(I,J) = ZZZ*DETJ
F3(I,J) = ZY*DETJ
F4(I,J) = FF*DETJ
F(I) = FF*DETJ
8C CCNTINUE
1CC CCNTINUE
      RETURN
      ENC

```



```

C
C
C
SLERCUTINE MATH4 (M,NELCON,R,Z,YIT,Y)
SLERCUTINE MATH4 CALCULATES THE 8X8, TEMPERATURE
DEPENDENT ELEMENT H4 MATRIX
IAFLICIT INTEGER*2(I-N)
REAL*8 SN,DESN,DXSN,WT,XXX,YYY,RR,FH,TEMP,TAUO
REAL*8XX,YY,TT,RD,GP
DIMENSION H4(8,8)
DIMENSION SNELCCN(11,1)
DIMENSION R(1),Z(1)
DIMENSION SN(8,25), DESN(8,25), CXSN(8,25), WT(25), GP(5)
DIMENSION Y(7,1)
DIMENSION YIT(1)
COMMON/CCAN/ NP,NEL,JC
COMMON/CP/ GP,NORD
COMMON/MTRXH4/ H4
COMMON/SHAFUN/SN, DESN, DXSN
COMMON/WEIGHT/ WT
A = NORC*NCRD
I = NELCCN(1,M)
J = NELCCN(3,M)
K = NELCCN(5,M)
LET J = C.25*(R(I)-R(J))*(Z(J)-Z(K))
DO 100 I = 1,8
DO 60 J = 1,8
FH = 0.C
CC 60 K = 1,N
XXX = SN(I,K)
YYY = SN(J,K)
RR = 0.C
TEMP = C.0
TALC = C.0
CC 40 L = 1,8
LL = NELCCN(L,M)
RL = R(LL)
XX = Y(I,LL+NP)
YY = Y(J,LL)
RR = RR + SN(L,K)*RD
TEMP = TEMP + SN(L,K)*XX
TALC = TALC + SN(L,K)*YY
CCNT INUE
40 IF (TEMP .LE. 1.) TEMP = 1.0
IF (TALC .LE. 1.) TALC = 1.0
TAL1 = TEMP/TAUO
IF (TAL1 .GE. 0.) GC TC 50
WRITE(6,999) I,J,M,TEMP,TAUO,RR
FCFMAT(5X,3110,5X,3G18.4)
999

```



```

CC 45 II = 1,8
LL = NELCCN(II,M)
WRITE(6,888) SN(II,K),Y(1,LL+NP),YIT(LL)
888 FCFMAT(5X,3G18.4,/)
45 CCNTINUE
50 TAU NL = ALOG(TAU1)
6C TT = TAU NL
7C FF = HH + XXX*YYY*RR*WT(K)*TT
8C CCNTINUE = FF*DETJ
9C H4(I,J) =
1CC CCNTINUE
    RETURN
    END

```



```

2      JE = NELCCN(3,M)
      CC = 220 K=3,5
      KK = NELCCN(K,M)
      KKK = JA(KK)/2 - 1
      LLL = JB(KK) - 1
      CC = 200 I=3,5
      II = NELCCN(I,M)
      JJ = II + 1
      NN = 0
      NN2 = 0
      CC = 180 I = 1, KKK
      LL = LLL + L
      LS = LLL + KKK
      KKM = NAME(LL)
      IF (II .NE. KKM) GC TO 160
      BIGK(LL) = BIGK(LL) + XK2(K,I)
      NN = 1
      IF (JJ .NE. KKS) GG TO 170
      BIGK(LS) = BIGK(LS) - XK2(K,I)
      NN2 = 1
      IF (NN1 .EQ. 1) .AND. (N2 .EQ. 1) GO TO 200
      CC CONTINUE
      160 CC CONTINUE
      170 CC CONTINUE
      200 CC CONTINUE
      220 4 RETURN
      END

```



```
C
C SLEROUT INE DIFFUN(Y,YL,T,HINV,DY,EIGH12,BIGF3,BIGF4,BIGF5,BIGK
C   I EIGF,IBP,I TYPE,JA,JB,NAME,NELCCN,R,Z,YIT)
C
C SLEROUT INE DIFFUN FORMS THE NODAL EQUATIONCS.
C IT SETS THE ITH EQUATION FOR FLUX EQUAL TO CYI
C AND FCR TEMPERATURE TO DYII
C
C IMPLICIT INTEGER*2(I-N)
C REAL*8 CYI,DYII
C DIMENSION BIGH12(1), BIGH3(1), BIGF5(1)
C DIMENSION BIGH4(1)
C DIMENSION BIGK(1)
C DIMENSION BIGF(1)
C DIMENSION YIT(1)
C DIMENSION DR(13)
C DIMENSION IBP(1), ITYPE(1)
C DIMENSION JA(1), JB(1), NAME(1)
C DIMENSION NELCON(11,1)
C DIMENSION R(1), Z(1)
C DIMENSION Y(7,1), CY(1)
C COMMON/BCUNC/ NCCUNT
C COMMON/CLAD/ DCC,CPC,DENC,TKC,SIGAC,HGAP
C COMMON/CNN/ NP,NEL,JC
C COMMON/CCCC/DCCC,SIGACC,VCO,F SURF
C COMMON/FUEL/DCFC,B,E,BETA,BETAI,DCLAMI,FFLUXC,AKINF,VEL,
C     SIGAFI,SIGFFFI,SIGAB,SIGFB,DENF
C COMMON/INPD/DR,HEIGHT,NPR,NEZ,NFE
C COMMON/TIME/ F,G,RHC
C DATA TCLED/-1.0/ TIME DEPENDENT TERMS
C CALCULATE THE TOLD GO TO 75
C IF(T.EQ.TOLD)GO TO 75
C CALL FUNC(T,HINV)
C CALCULATE THE SYSTEM H4 MATRIX
C WFLCH IS TEMPERATURE DEPENDENT
C C30 I = 1,JC
C BIGF4(I) = 0.0
C C50 I = 1,NEL
C IF(NELCCN(11,I).GT.1)GO TO 50
C CALL MATF4(1,NELCCN,R,Z,YIT,Y)
C CALL SYSTF4(1,JA,JB,NAME,NELCON,BIGH4)
C CONTINUE
C TCLED=1
C VINV=1.0/VEL
C Z1=Z(IBP(2))-Z(IBP(1))
C E1=1.0-BETA
C SIGCF=SIGFFI
C SIGNST1=AKINF*B1-1.0
```



```

CCAST2 = SIGAF*B1*RH
TC = 0.98
FMCC = 1.0/(0.55*HEIGHT)
CC 510 I = 1,NP
JAA = JA(I)
JEB = JB(I)
JAB = JAA + JBB - 1
II = ITYPE(I) + 1
II = NP + I
CYI = 0.0
CYII = 0.0
CC TO (100,200,300,400),IT
CC 150 J = JBB,JAB
NN = NAME(J)
NN = NN + NP
DVI = DVI + (DCF*DBLE(BIGH12(J)) - ((CONST1+F)*SIGAF+CCNST2))*
1 CLE(BIGH3(J))+B*SIGAF*B1*DBLE(BIGH4(J))*Y(1,NN)-G*DBLE(BIGF(NN))
2 + VINV*HINV*DBLE(BIGH3(J))*Y(2,NN)
TKK = Y(1,NNN) + 273.
TKF = ((1.0-2.5*(1.0-TD))*(45.1/(135.+TDK) + 4.79E-13*TDK**2))*C.235
CPFF = (18.45 + 2.431E-3*TDK - 2.272E5/TDK**2)/270.07
CYII = DVI + TKF*DBLE(BIGH12(J))*Y(1,NNN) + FINV*DENF*CPF*
1 CLE(BIGH3(J))*Y(2,NNN)-E*SIGFF*CELE(BIGH3(J))*Y(1,NN)
CC CONTINUE
CC TC 500

C DETERMINE THE TYPE CF NCDE BEING CCNSIDERED
K = 0 NCDE NOT CN THE FUEL-CLAC INTERFACE
K = 1 NCDE CN THE FUEL-CLAC INTERFACE

200 K = 0
DC 210 IB = 1,NCCUNT
IF(1BP(IB) .EQ. 1) GO TO 215
210 CC CONTINUE
CC TC 220
215 K = 1
220 CC 250 J = JBB,JAB
NN = NAME(J)
NN = NN + NP
DVI = CYI + (DCF*DBLE(BIGH12(J)) - ((CONST1+F)*SIGAF+CCNST2))*
1 CLE(BIGH3(J))+B*SIGAF*B1*DBLE(BIGH4(J))*Y(1,NN)-G*CLE(BIGF(NN))
2 + VINV*HINV*DBLE(BIGH3(J))*Y(2,NN)
RA = R(NN)
TKK = Y(1,NNN) + 273.
TKF = ((1.0-2.5*(1.0-TD))*(45.1/(135.+TDK) + 4.79E-13*TDK**2))*C.235
CPFF = (18.45 + 2.431E-3*TDK - 2.272E5/TDK**2)/270.07
FMCC = (1000.0 + 247.0*COS(3.14159*(Z(NN)*FMCD-0.81818)))*1.356E-4
CCAST3 = FMCC*Z1

```


[illegible]

C**

```
NFF = NF + 1
NN = NPR*2
NN = NPF - NN
LC 600 I = NN,NP
CY(I) = 0.0
CCCONTINUE
```

6CC

C**

C**

C

C

C**

C**

TC ACCOUNT FOR THE TEMP BOUNDARY CONDITIONS
(CCCONSTANT COOLANT INLET TEMPERATURES)

```
NN = NP - 2
LC 700 I = NN,NP
CY(I+NP) = 0.0
CCCONTINUE
RETURN
ENC
```

7CC

126


```

1  + B*SIGAF*B1*BIGH4(J)
TCK = Y(1,NNN) + 273.
TKF = ((1.0-2.5*(1.0-ID))*(45.1/(135.+TDK) + 4.79E-12*(TK**2)))*C.235
CFF = (18.45 + 2.431E-3*TOK - 2.272E5/TOK**2)/270.07
PW(J+JC) = TKF*BIGH12(J)+DENF*CPF*AH*BIGH3(J)
CCCONTINUE
15C GC TC 500

C DETERMINE THE TYPE CF NODE BEING CONSIDERED
K = 0 NCDE NOT ON THE FUEL-CLAD INTERFACE
K = 1 NCDE CN THE FUEL-CLAD INTERFACE
20C K = 2
CC 210 IB = 1,NCOUNT
IF(IBP(IB) .EQ. I) GO TC 215
21C CCCONTINUE
GC TC 220
215 K = 1
22C CC 250 J = JBB,JAB
PW(J) = DCF*BIGH12(J)-((CONST1+F)*SIGAF+CCNST2-AH*VINV)*BIGH3(J)
1  + B*SIGAF*B1*BIGH4(J)
NN = NAME(J) + NP
RA = R(NN)
FGAP = (1000.0 + 247.0*COS(3.14159*(Z(NN)*MCD-0.81818)))*1.356E-4
CCNST3 = HGAP*Z1
TCK = Y(1,NNN) + 273.
TKF = ((1.0-2.5*(1.0-ID))*(45.1/(135.+TDK) + 4.79E-12*(TK**2)))*C.235
CFF = (18.45 + 2.431E-3*TOK - 2.272E5/TOK**2)/270.07
PW(J+JC) = TKF*BIGH12(J)+DENF*CPF*AH*BIGH3(J)+RA*CCNST3*BIGHK(J)
IF(K .EQ. 0) GO TC 250
IF(J .LT. JAB) GO TO 250
J = JAB + J - JAB + 1
PW(J) = DCF*BIGH12(JJ) + (SIGAC + AH*VINV)*BIGH3(JJ)
IF(J .NE. JAB) GO TO 25C
PW(JBB) = PW(J)
PW(J) = 0.0
CCCONTINUE
25C GC TC 500

C DETERMINE THE TYPE CF NODE BEING CONSIDERED
K = 1 NCDE NOT CN AN INTERFACE
K = 2 NCDE CN THE FUEL-CLAD INTERFACE
K = 3 NCDE ON THE CLAD-COOLANT INTERFACE
30C K = 1
NCCCL = 0
CC 320 IB = 1,NCOUNT
LEF = IBP(IB) + 1

```



```

IF(LBP, EQ, I) GC TC 325
NBP = LBP + 2
IF (NCCD, EQ, 0) GC TO 305
NBP = NBP - 1
NCCD = C
GC TO 310
NCCD = 1
305 NCCD = 1
310 IF(NBP, EQ, I) GO TO 330
320 CCNTINUE
GC TC 340
325 K = 2
GC TO 340
330 K = 3
CC 380 J = JBB, JAB
NN = NAME(J)
NNN = NAME(J) + NP
RA = R(NN)
PW(J) = 0.0
IF(K, EQ, 2) GO TO 360
PW(J) = DCC*BIGH12(J) + (SIGAC + AH*VINV)*BIGH3(J)
360 F-GAP = (1000.0 + 247.0*CCS(3.14159*(Z(NN)*FMCC-C.81818))) * 1.356E-4
F = HGAP
IF(K, EQ, 3) H = HSURF
PW(J+JC) = TKC*BIGH12(J)+RA*H*Z1*BIGK(J)+AH*ENC*CPC*BIGH3(J)
IF(K, NE, 3) GO TO 380
IF(J, NE, JA3) GO TC 380
J = JAB + J - JA3 + 1
PW(J) = DCC*BIGH12(JJ) + (SIGACC + AH*VINV)*BIGH3(JJ)
IF(J, NE, JA3) GC TC 380
PW(JBB) = PW(JBB) + PW(J)
PW(J) = 0.0
CCNTINUE
380 IF(K, NE, 2) GO TO 500
IF(JBB) = 1.0
PW(JA3) = -1.0
GC TO 500

CC
CC
CC
CC
Determine the type of NCDE being considered
K = 1 Node not on CLAD-COOLANT INTERFACE
K = 2 NCDE on the CLAD-COOLANT INTERFACE

400 K = 1
NCCD = C
DC 420 IB = 1, NCCLNT
LEF = LBP(IB) + 4
IF(NCCD, EQ, 0) GC TO 410
LEF = LBP - 1
NCCD = C

```



```

GC TO 415
410 ACCC = 1
415 IF(LBP.EC. 1) GO TO 425
420 CCNTINUE
GC TC 420
425 K = 2
430 CC 480 J = JBB, JAB
435 NN = NAME(J)
440 NN = NAME(J) + NF
445 PW(J) = 0.0
450 IF(K.EC. 2) GO TO 450
455 PW(J) = DCCO*BIGH12(J) + (SIGACC + AH*VINV)*BIGH3(J)
460 RE = R(NN)
465 TCF = 1.8*Y(1,NNN) + 32.0
470 TKCO = (54.306 - 1.878E-2*TCF + 2.0914E-6*TCF**2)*4.134E-3
475 CPCC = C.34574 - 0.79226E-4*TDF + C.34086E-7*TCF**2
480 DENC = (59.566 - 7.9504E-3*TDF - 0.2872E-6*TCF**2 +
1 C.06C35E-9*TDF**3)*0.01601
485 A = DENC0*CPCC
490 PW(J+JC) = TKCO*BIGH12(J) + VCO*A*BIGH5(J) + AT*A*BIGH3(J)
495 1 + RB*HSURF*Z1*BIGH(J)
48C CCNTINUE
490 IF(K.NE. 2) GO TC 500
495 PW(JBB) = 1.0
500 PW(JA3) = -1.0
505 CCNTINUE
510 C***
515 C***
520 C***
525 C***
TC ACCCNT FOR THE FLUX AXIAL BCUNARY CCNDITICNS
530 C***
535 C***
540 C***
545 C***
NPF = NP + 1
550 NN = NPF*2
555 NN = NPF - NN, NP
560 CC 600 I = NN, NP
565 PW(JB(I)) = 1.0
570 JS = JB(I) + 1
575 JE = JB(I) + JA(I) - 1
580 CC 600 J = JS, JE
585 PW(J) = 0.0
590 CCNTINUE
595 C***
600 C***
605 C***
610 C***
TC ACCCNT FOR THE TEMP BOUNDARY CCNDITICNS
615 C***
620 C***
625 C***
630 C***
635 C***
640 C***
NN = NP - 2

```



```

CC 700 I = NN, NP
PW(JB(I)+JC) = 1.0
JS = JB(I) + 1 + JC
JE = JB(I) + JA(I) - 1 + JC
CC 700 J = JS, JE
PW(J) = 0.0
CCCONTINUE
700 RETURN
END

```



```

C
C
C
C
C
SUBROUTINE NUTISL (PW,DY,FI,N,NY,EPS,YMAX,NEWPW,KRET,BIGH3,
1 ITYPE,JA,JB,NAME,NELCCN)
SUBROUTINE NUTISL CALCULATES THE NEWTON-ITERATES
USING A 'SOR' TECHNIQUE WITH AN CVER-RELAXATION
FACTOR EQUAL TO OMEGA1
IMPLICIT INTEGER*2(I-N)
INTEGER*4 N,NY,NEWPW,KRET
REAL*8 PN,TN,BIGH3(1)
DIMENSION ITYPE(1)
DIMENSION JAL(1),JB(1),NAME(1)
DIMENSION JALCON(1,1),DY(1),FI(1),YMAX(1)
DIMENSION PW(1),CCNN/ NP,NEL,JC
CCMCMCN/FUEL/CCF,B,E,BETA,BETAI,DCLAMI,FFLUXC,AKINF,VEL,
1 SIGAF,SIGFF,SIGAB,SIGFB,DENF
DATA CMCG,OMEGM1/1.02,0.02/
TEST = 1.E35
KRET = C
EPSS = EPS*2
EPSA2 = EPSS*0.0001
NCIT = N
EI = E*SIGFF
DO 100 I = 1,NP
FI(I) = DY(I)/PW(JB(I))
FI(I+NP) = DY(I+NP)/PW(JB(I)+JC)
100 CC 300 ITT = 1,NGIT
RCF = 0.
RCF1 = 0.
RCF2 = C.
CCF1 = 0.
CCF2 = 0.
CC 200 I = 1,NP
JS = JB(I) + 1
JE = JB(I+1) - 1
IT = ITYPE(I)
IFN = DY(I)
TN = DY(I+NP)
CC 150 J = JS,JE
PN = PN - DBLE(PW(J))*FI(NAME(J))
TN = TN - DBLE(PW(J+JC))*FI(NAME(J)+NP)
IF(IT.GT.1) GO TC 150
TN = TN + EI*DBLE(BIGH3(J))*FI(NAME(J))
150 CC  CONTINUE
FN = PN/PW(JS-1)

```



```

TN = TN/PW(JS+JC-1)
FN = FN*OMEG - F1(I)*OMEGM1
TN = TN*OMEG - F1(I+NP)*OMEGM1
PCH = F1(I) - PN
TCH = F1(I+NP) - TN
IF((ABS(PCH).GT.TEST).OR.(ABS(TCH).GT.TEST))GC TC 18C
CF1 = CFI + (PCH/YMAX(I))*2
CF2 = CH2 + (TCH/YMAX(I+NP))*2
RCH1 = RCH1 + (PCH/AMAX1(ABS(SNGL(PN)),EPS))*2
RCH2 = RCH2 + (TCH/AMAX1(ABS(SNGL(TN)),EPS))*2
GC TO 190
18C CF1 = 1.
    CF2 = 1.
    RCH1 = 1.
    RCH2 = 1.
    F1(I) = 1.
    F1(I+NP) = 1.
    CH1 = CH1 + PN
    CH2 = CH2 + RCH1
    RCH = RCH1 + RCH2
    IF(RCH.LT.EPSS) RETURN
    IF(CH.LT.EPSA2) RETURN
    CCNTINUE
    KRET = 1
    RETURN
END
2CC
3CC

```



```

SLEROUTINE LDASUB (Y,YL,T,TEND,N,NY,M,J,START,KFLAG,MAXOR,IPRT,F,
1 FMIN,HMAX,RMSEPS,SAVE,YLSV,YMAX,ER,ESV,FI,CY,PW,BIGF12,BIGF3,
2 BIGF4,BIGH5,BIGK,BIGF,IBP,ITYPE,JA,JB,NAME,NELCCN,RR,Z,YIT)
LCA 10

SLEROUTINE LDASUB IS A MODIFICATION OF SLEROUTINE CFASUB
WHICH IS DUE TO R. L. BROWN AND C. W. GEAR. CFASUB IS DOCUMENTED
IN THE REPERT
LCA 30
LCA 40
LCA 50
LCA 60
LCA 70
LCA 80
LCA 90
LCA 100
LCA 110
LCA 120
LCA 130
LCA 140
LCA 150
LCA 160
LCA 170
LCA 180
LCA 190
LCA 200
LCA 210
LCA 220
LCA 230
LCA 240
LCA 250
LCA 260
LCA 270
LCA 280
LCA 290
LCA 300
LCA 310
LCA 320
LCA 330
LCA 340
LCA 350
LCA 360
LCA 370
LCA 380
LCA 390
LCA 400
LCA 410
LCA 420
LCA 430
LCA 440
LCA 450
LCA 460
LCA 470

CO DOCUMENTATION FOR CFASUB--
BY R. L. BROWN AND C. W. GEAR
REPERT UIUCDCS-R-73-575 JULY 1973
UNIVERSITY OF ILLINOIS AT URBANA-CHAMPAIGN
URBANA, ILLINOIS 61801
THIS REPORT IS AVAILABLE FROM THE NATIONAL TECHNICAL INFORMATION
SERVICE OF THE U. S. DEPARTMENT OF COMMERCE UNDER ACCESSION NUMBER
CCC-1465-225.

THE MODIFICATION HERE IS DOCUMENTED IN THE REPORT SPARSE SYSTEMS OF
ALGEBRAIC AND IMPLICITLY DEFINED STIFF DIFFERENTIAL EQUATIONS
BY RICHARD FRANK
REPORT NPS53FE76C51, MAY 1976
NAVAL POSTGRADUATE SCHOOL
MCNTREY, CALIFORNIA 93940

-----

THE CALLING SEQUENCE FOR LDASUB IS

CALL LDASUB(Y,YL,T,TEND,N,NY,M,J,START,KFLAG,MAXOR,IPRT,F,FMIN,
FMAX,RMSEPS,SAVE,YLSV,YMAX,ER,ESV,FI,DY,PA)

WHERE THE PARAMETERS ARE DEFINED AS FOLLOWS:
Y - ARRAY DIMENSIONED (7,NY). THIS ARRAY CONTAINS THE
DEPENDENT VARIABLES AND THEIR DERIVATIVES. VAR
Y(J+1,I) CONTAINS THE J-TH DERIVATIVE OF THE I-TH VAR
TABLE TIMES H**J/J-FACTORIAL, WHERE H IS THE CURRENT
STEP SIZE. ON FIRST ENTRY THE CALLER SUPPLIES THE
INITIAL VALUES OF EACH VARIABLE CF IN Y(1,I) AND ANS
IN Y(2,I). ON SUBSEQUENT VALUES CF IT IS ASSUMED THAT
THE ARRAY HAS NOT BEEN CHANGED. TC INTERPOLATE TC
NON-MESH POINTS, THESE VALUES CAN BE USED AS FOLLOWS.
IF H IS THE CURRENT STEP SIZE AND VALUES AT TIME T+E
NEEDED, LET S = E/H AND THEN

```

```

I-TH VARIABLE AT T+E IS SUM Y(J+1,I)*S**J
NG SUM
J=0

```

CC

48C LDA
490 LDA
500 LDA
510 LDA
520 LDA
530 LDA
540 LDA
550 LDA
560 LDA
570 LDA
580 LDA
590 LDA
600 LDA
610 LDA
620 LDA
630 LDA
640 LDA
650 LDA
660 LDA
670 LDA
680 LDA
690 LDA
700 LDA
710 LDA
720 LDA
730 LDA
740 LDA
750 LDA
760 LDA
770 LDA
780 LDA
790 LDA
800 LDA
810 LDA
820 LDA
830 LDA
840 LDA
850 LDA
860 LDA
870 LDA
880 LDA
890 LDA
900 LDA
910 LDA
920 LDA
930 LDA
940 LDA
950 LDA

```

THE VALUE OF NQ IS OBTAINED IN THE CALLING PROGRAM
BY NQ = JSTART.

- ARRAY OF NL = N - NY VARIABLES WHICH APPEAR LINEARLY.
- THE USER SUPPLIES INITIAL VALUES FOR THESE VARIABLES.
- CURRENT VALUE OF THE INDEPENDENT VARIABLE (TIME)
- END TIME
- TOTAL NUMBER OF VARIABLES
- NUMBER OF DIFFERENTIAL EQUATIONS AND NONLINEAR
  VARIABLES.
- NUMBER OF VARIABLES INCLUDED IN THE ERROR TEST.
- THIS NUMBER CAN BE NC GREATER THAN NY. IF IT IS
  GREATER THAN NY, NY VARIABLES ARE USED IN THE ERROR
  TEST.
- INPUT AND OUTPUT INDICATOR.
  JSTART HAS THE FOLLOWING MEANINGS.
  THIS INDICATES A RE-START FROM A PREVIOUS
  POINT FOLLOWING A TERMINATION OF THE RUN OR
  SOLUTION OF ANOTHER PROBLEM DURING THE SAME
  RUN. PARAMETERS IN THE CALLING SEQUENCE
  MUST HAVE BEEN PREVIOUSLY RESERVED FOR THE PREVIOUS
  USE, PARTIALLY THE ARRAYS AND PW.
  THESE ARRAYS MUST BE SAVED AFTER A CALL
  TO SUBROUTINE LDASAV, WHICH ALSO SAVES
  NECESSARY PARAMETERS, INTERNAL TO LCASUB.
  INDICATES AN INITIAL CALL TO LCASUB. THE
  ROUTINE INITIALIZES ITSELF, SCALES THE
  DERIVATIVES INITIALLY(2, I) AND THEN PERFORMS THE
  INTEGRATION UNTIL I > TEND. IT BE CONTINUED.
  INDICATES THE SOLUTIONARY IT IS NEITHER
  AFTER THE INITIAL ENTRY TO RE-ENTER WITH
  DESIRABLE NCR SINCE THIS RE-INITIALIZES
  JSTART = 0, BEGINNING WITH A FIRST ORDER
  METHOD AGAIN.
  ON OUTPUT, JSTART IS SET TO THE VALUE OF NC, THE
  ORDER OF THE FORMULA CURRENTLY BEING USED.
- THE COMPLETION CODE INDICATOR, WITH THE FOLLOWING
  MEANINGS
  +1 THE INTEGRATION WAS SUCCESSFUL
  -1 ERROR TEST FAILED FOR F > HMIN
  -3 CORRECTOR FAILED TO CONVERGE FOR F > HMIN
  -4 CORRECTOR FAILED TO CONVERGE FOR FIRST
    ORDER METHOD
  -5 ERROR RETURN FROM SUBROUTINE NLITSL
- MAXIMUM ORDER DERIVATIVE THAT SHOULD BE USED IN THE

```

CC


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DIMENSION Y(7,1), YL(1), SAVE(7,1), YMAX(1), ER(1), YLSV(1), F1(1) LDA
1 PERT(6,3), COF(21), ES(1), DY(1), PW(1), SAV(1), A(25) LDA
EQUIVALENCE (A(8),END), (A(9),ER), (A(10),E), (A(11),EDWN), LDA
1(A(12),ENQ1), (A(13),ENQ2), (A(14),ENC3), (A(15),EFS), (A(16),EUP) LDA
2,(A(17),HNEW), (A(18),PEPSH), (A(19),IDCUB), (A(20),IWEAL), LDA
3(A(21),K), (A(22),LCOPYL), (A(23),LCOPYY), (A(24),MAXCER), LDA
4(A(25),N1), (A(26),N1), (A(27),NC), (A(28),NS), (A(29),Nw) LDA
----- LDA
THE CCEFFICIENTS IN THE PERT ARRAY ARE USED FOR ERROR TESTING AND
CHANGING STEP SIZE AND NEED TO BE ACCURATE TO ONLY A FEW DIGITS. LDA
----- LDA
DATA PERT/4.,9.,16.,25.,36.,49.,5.,16.,25.,36.,49.,64.,1.,1.,.25, LDA
12.7885E-2,1.70569E-3,6.83925E-5/ LDA
----- LDA
THE ENTRIES IN THE COF ARRAY ARE THE COEFFICIENTS FOR THE STIFFLY
STABLE METHODS USED IN THIS PROGRAM AND ARE TO BE THE MACHINE
PRECISION EQUIVALENTS OF THE FOLLOWING CONSTANTS. LDA
----- LDA
-1/2, -1/2 -1/6 -1/6 -1/24, -1/12, -1/24, -1/120 LDA
-11/6, -1, -35/24, -5/12, -17/24, -1/8, -35/144, -7/24, -1/72C LDA
-25/12, -15/8, -203/90, -49/48, -35/144, -7/24, -1/72C LDA
-127/60, -15/8, -203/90, -49/48, -35/144, -7/24, -1/72C LDA
-147/60, -203/90, -49/48, -35/144, -7/24, -1/72C LDA
----- LDA
DATA COF/-1.,-1.5,-.5,-1.833333,-1.,-1.666667,-2.083333,-1.458333, LDA
1-.416667,-.04166667,-2.283333,-1.875,-.708333,-.125,-.08333333, LDA
2-.2.45,-2.255556,-1.020833,-.2430556,-.02516667,-.00138889/ LDA
IF (JSTART) 100,110,150 LDA
----- LDA
IF THIS IS A RESTART ENTRY, RESTORE Y AND YL FROM THE SAVE AND
YLSV ARRAYS, WHERE THEY WERE SAVED BY A PREVIOUS CALL TO LCASAV. LDA
----- LDA
100 CALL COPYZ (Y,SAVE,LCOPYY) LDA
CALL CCPYZ (YL,YLSV,LCOPYL) LDA
GC TC 150 LDA
----- LDA
IF THIS IS THE FIRST CALL, INITIALIZE YMAX, SCALE DERIVATIVES, AND
INITIALIZE INDICATORS AND SET ORDER TO ONE. LDA
FOR DOUBLE PRECISION, SET LCCPYL = 14*NY AND LCCPYL = 2*NL IF
ROUTINE COPYZ IS IN SINGLE PRECISION. LDA
----- LDA
110 NL = N-NY LDA

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CCCCCCCC

CCCCCCCCCCCCCCCC

CCCC

CCCCCCC


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LCCPYY = 7*NY
LCCPYL = NL
M1 = MINO(M,NY)
EPCS = SQRT(FLOAT(M1))*RMSEPS
MAXCER = MINO(MAXOR,6)
IF (IPRT.LE.0) GC TC 120
PRINT 3, N,NL,RMSEPS,TEND,H
12C NS = C
    NW = 0
C
CC 130 J=1,NY
YMAX(J) = AMAX1(1.,ABS(Y(1,J)))
13C Y(2,J) = Y(2,J)*H
C
NG = 1
BR = 1.
ASSIGN 190 TO IRET
C
SET COEFFICIENTS FOR THE CRDER CURRENTLY BEING USED.
EUP IS A TEST FOR ERRORS OF THE CURRENT ORDER NG
EUP IS TO TEST FOR INCREASING THE ORDER, EDWN FOR DECREASING THE
CRDER.
14C K = NG*(NG-1)/2
CALL COPYZ (A(2),COF(K+1),NG)
K = NG+1
ICCUB = NG
ENG1 = .5/NG
ENG2 = .5/K
ENG3 = .5/(NG+2)
PEPST = EPS*#2
EUP = PERT(NG,1)*PEPST
ELP = PERT(NG,2)*PEPSTH
EDWN = PERT(NG,3)*PEPSTH
BNC = (EPS*ENG3)**2
IWEVAL = 1
GC TO IRET, (190,200,45C,570)
15C IF (H.EC.HNEW) GO TO 190
C
IF CALLER HAS CHANGED P, RESCALE DERIVATIVES TO REFLECT THAT PNEW
WAS USED ON THE LAST CALL.
C
R = H/HNEW
ASSIGN 190 TO IRET
GC TO 610
C
SET JSTART TO NQ, THE CURRENT ORDER OF THE METHCC, BEFCRE EXIT,

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C      AND SAVE THE CURRENT STEPSIZE IN TNEW.
C 160 JSTART = NC
      FNEW = F
      RETURN
C 170 NC = NS+1
      IF (IPRT.LE.0) GO TC 180
C      PRINT DATA IF DESIRED BY USER
C      PRINT I, NS,NW,NQ,F,T,(Y(1,I),I=1,NY)
C 180 CONTINUE
      IF (KFLAG.LT.0) GO TO 160
      IF (T.GE.TEND) GO TO 160
C      TAKE ANOTHER STEP IF T < TEND
C      JSTART = 1
C      SAVE DATA FOR TRIAL WITH A SMALLER TIMESTEP IF THIS STEP FAILS
C 190 CALL CCOPYZ (SAVE,Y,LCCPPY)
      CALL CCOPYZ (YLSV,YL,LCCPYL)
      RACUM = 1.
      KFLAG = 1
      FCLD = F
      NCCLC = NQ
      TCCLC = T
      T = T+H
      F INV = 1./F
C      COMPUTE PREDICTED VALUES BY EFFECTIVELY MULTIPLYING DERIVATIVE
      VECTOR BY PASCAL TRIANGLE MATRIX
C 200 J=2,K
      J3 = K+J-1
C 210 J1=J,K
      J2 = J3-J1
C 210 Y(J2,I) = Y(J2,I)+Y(J2+1,I)
C 220 I=1,NY
      DC 220 I=1,NY
      ER(I) = 0.

```



```

C CC LP TC THREE CORRECTR ITERATIONS. CONVERGENCE IS OBTAINED WHEN
C CHANGES ARE LESS THAN BND WHICH IS DEPENDENT ON THE ERROR TEST
C CONSTANT. THE SUM OF CORRECTIONS IS ACCUMULATED IN ER(I). IT IS
C EQUAL TO THE K-TH DERIVATIVE OF Y TIMES F**K/(K-FACTCRIAL*(K)),
C AND THUS IS PROPORTIONAL TO THE ACTUAL ERRORS TO THE LOWEST POWER
C OF F PRESENT, WHICH IS F**K.
C
C CC 270 L=1,3
C CALL DIFFUN (Y,YL,T,HINV,DY,BIGH12,BIGH3,BIGH4,BIGH5,BIGK,BIGF,
C 1 IEP,ITYPE,JA,JB,NAME,NELCON,RR,Z,YIT)
C IF (IWEVAL.LT.1) GC TC 230
C
C IF THERE HAS BEEN A CHANGE OF ORDER OR THERE HAS BEEN TROUBLE
C WITH CONVERGENCE, PW IS RE-EVALUATED PRIOR TO STARTING THE
C CORRECTR ITERATION. IWEVAL IS THEN SET TO -1 AS AN INDICATOR
C THAT IT HAS BEEN DONE. NEWPW IS SET NONZERO TO INDICATE TO
C SUBROUTINE NUTSL THAT A NEW PW HAS BEEN PROVIDED.
C
C CALL JACMAT (Y,YL,T,HINV,A(2),N,NY,EPS,DY,F1,PW,BIGH12,BIGH3,
C 1 BIGH4,BIGH5,BIGK,IBF,ITYPE,JA,JB,NAME,NELCON,RR,Z)
C KFLAG = 1
C IWEVAL = -1
C NAME = NW+1
C NEWPW = 1
C CALL NUTSL (PW,DY,F1,N,NY,EPS,YMAX,NEWPW,KFRET,EIGH3,ITYPE,
C 1 JA,JB,NAME,NELCON)
C IF (KFRET.NE.0) GC TO 600
C IF (NL.LE.0) GO TO 250
C
C CC 240 I=1,NL
C 24C YL(I) = YL(I)-F1(I+NY)
C
C 25C CCNTINUE
C DEL = 0.
C
C CC 260 I=1,NY
C Y(1,I) = Y(1,I)-F1(I)
C Y(2,I) = Y(2,I)+A(2)*F1(I)
C ER(I) = ER(I)+F1(I)
C DEL = DEL+(F1(I)/AMAX1(YMAX(I),ABS(Y(1,I))))**2
C CCNTINUE
C
C IF (L.GE.2) BR = AMAX1(.9*BR,DEL/CELL)
C CELL = DEL
C IF (AMIN1(DEL,BR*DEL*2.).LE.BND) GO TO 330
C CCNTINUE
C
C 270 CCNTINUE

```



```

C      THE CORRECTOR ITERATION FAILED TO CONVERGE IN 3 TRIES.  VARIUS
C      POSSIBILITIES ARE CHECKED FOR.  IF H IS ALREADY HAIN AND PW FAS
C      ALREADY BEEN RE-EVALUATED, A NO CONVERGENCE EXIT IS TAKEN.
C      OTHERWISE THE MATRIX PW IS RE-EVALUATED AND/CR (IN THAT CR CER) THE
C      STEP IS REDUCED TO TRY AND GET CONVERGENCE.
C      T = TOLD
C      IF (IWEVAL) 280,300,290
C      IF (H.LE.HMIN*.1,00001) GO TO 310
C      RACUM = RACUM*.25
C      CCNTINUE
C      GC TO 560
C      KFLAG = -3
C      RESTORE Y AND YL AFTER CONVERGENCE FAILURE
C      CALL COPYZ (Y,SAVE,LCCPPY)
C      CALL COPYZ (YL,YLSV,LCCPYL)
C      F = HOLL
C      NC = NQCCLC
C      GC TO 170
C      THE CORRECTOR CONVERGED, SC NOW THE ERROR TEST IS MADE.
C      C = 0.
C      CC 340 I=1,M1
C      YM = AMAX1 (ABS(Y(1,I)),YMAX(I))
C      C = D+(ER(I)/YM)*2
C      IWEVAL = 0
C      IF (D.GT.E) GO TO 380
C      THE ERRCR TEST IS CKAY, SO THE STEP IS ACCEPTED.  IF IDOUE
C      NCW BECOMES NEGATIVE, A TEST IS MADE TO SEE IF THE STEP SIZE
C      CAN BE INCREASED AT THIS ORDER OR CNE HIGHER CRNE LOWER.
C      THE CHANGE IS MADE ONLY IF THE STEP CAN BE INCREASED BY AT
C      LEAST 10%.  IDOUB IS SET TO NO TO PREVENT FURTHER TESTING
C      FOR A WHILE.  IF NC CHANGE IS MADE, IDOUB IS SET TO 5.
C      IF (K.LT.3) GO TO 360
C      CC 350 J=3,K
C      CC 350 I=1,NY
C      CC 350 Y(J,I) = Y(J,I)+A(J)*ER(I)

```



```

C 36C KFLAG = 1 IDCUB-1
      ICCUB = IDCUB-1
      IF (IDCUB) 410,37C,510
      CALL CCPYZ (ESV,ER,M1)
      GC TO 510
      LDA 3720
      LDA 3730
      LDA 3740
      LDA 3750
      LDA 3760
      LDA 3770
      LDA 3780
      LDA 3790
      LDA 3800
      LDA 3810
      LDA 3820
      LDA 3830
      LDA 3840
      LDA 3850
      LDA 3860
      LDA 3870
      LDA 3880
      LDA 3890
      LDA 3900
      LDA 3910
      LDA 3920
      LDA 3930
      LDA 3940
      LDA 3950
      LDA 3960
      LDA 3970
      LDA 3980
      LDA 3990
      LDA 4000
      LDA 4010
      LDA 4020
      LDA 4030
      LDA 4040
      LDA 4050
      LDA 4060
      LDA 4070
      LDA 4080
      LDA 4090
      LDA 4100
      LDA 4110
      LDA 4120
      LDA 4130
      LDA 4140
      LDA 4150
      LDA 4160
      LDA 4170
      LDA 4180
      LDA 4190

      -----
      THE ERROR TEST FAILED. IF JSTART = 0, THE DERIVATIVES IN THE
      SAVE ARRAY ARE UPDATED. TESTS ARE THEN MADE TO FIX THE STEPSIZE
      AND PERHAPS REDUCE THE ORDER. AFTER RESTRICTING AND SCALING THE
      Y VARIABLES, THE STEP IS RETRIED.
      -----
C 38C IF (JSTART.GT.0) GC TO 400
      CC 390 I=1,NY
      CC SAVE(2,I) = Y(2,I)
      LDA 3840
      LDA 3850
      LDA 3860
      LDA 3870
      LDA 3880
      LDA 3890
      LDA 3900
      LDA 3910
      LDA 3920
      LDA 3930
      LDA 3940
      LDA 3950
      LDA 3960
      LDA 3970
      LDA 3980
      LDA 3990
      LDA 4000
      LDA 4010
      LDA 4020
      LDA 4030
      LDA 4040
      LDA 4050
      LDA 4060
      LDA 4070
      LDA 4080
      LDA 4090
      LDA 4100
      LDA 4110
      LDA 4120
      LDA 4130
      LDA 4140
      LDA 4150
      LDA 4160
      LDA 4170
      LDA 4180
      LDA 4190

C 40C KFLAG = KFLAG-2
      IF (H.LE.FMIN) GC TO 550
      I = TOLD
      IF (KFLAG.LE.-5) GC TO 530
      PR2 = (C/E)**ENQ2*1.2
      L = 0
      IF (NQ.LE.1) GO TO 430
      D = 0.
      CC 420 J=1,M1
      YM = AMAX1(ABS(Y(1,J)),YMAX(J))
      C = C+(Y(K,J)/YM)**2
      LDA 3840
      LDA 3850
      LDA 3860
      LDA 3870
      LDA 3880
      LDA 3890
      LDA 3900
      LDA 3910
      LDA 3920
      LDA 3930
      LDA 3940
      LDA 3950
      LDA 3960
      LDA 3970
      LDA 3980
      LDA 3990
      LDA 4000
      LDA 4010
      LDA 4020
      LDA 4030
      LDA 4040
      LDA 4050
      LDA 4060
      LDA 4070
      LDA 4080
      LDA 4090
      LDA 4100
      LDA 4110
      LDA 4120
      LDA 4130
      LDA 4140
      LDA 4150
      LDA 4160
      LDA 4170
      LDA 4180
      LDA 4190

C 42C PR1 = (C/EDWN)**ENC1*1.3
      IF (PR1.GE.PR2) GO TO 430
      PR2 = PR1
      L = -1
      IF (KFLAG.LT.0.OR.NQ.GE.MAXCER) GC TO 450
      C = 0
      CC 440 J=1,M1
      YM = AMAX1(ABS(Y(1,J)),YMAX(J))
      C = C+((ER(J)-ESV(J))/YM)**2
      LDA 3840
      LDA 3850
      LDA 3860
      LDA 3870
      LDA 3880
      LDA 3890
      LDA 3900
      LDA 3910
      LDA 3920
      LDA 3930
      LDA 3940
      LDA 3950
      LDA 3960
      LDA 3970
      LDA 3980
      LDA 3990
      LDA 4000
      LDA 4010
      LDA 4020
      LDA 4030
      LDA 4040
      LDA 4050
      LDA 4060
      LDA 4070
      LDA 4080
      LDA 4090
      LDA 4100
      LDA 4110
      LDA 4120
      LDA 4130
      LDA 4140
      LDA 4150
      LDA 4160
      LDA 4170
      LDA 4180
      LDA 4190

C 44C PR1 = (C/EUP)**ENC3*1.4
      IF (PR1.GE.PR2) GO TO 450
      PR2 = PR1
      L = 1
      R = 1./AMAX1(PR2,1.E-5)
      IF (KFLAG.LT.0.OR.R.GE.1.1) GC TO 460
      ICCUB = 9
      LDA 3840
      LDA 3850
      LDA 3860
      LDA 3870
      LDA 3880
      LDA 3890
      LDA 3900
      LDA 3910
      LDA 3920
      LDA 3930
      LDA 3940
      LDA 3950
      LDA 3960
      LDA 3970
      LDA 3980
      LDA 3990
      LDA 4000
      LDA 4010
      LDA 4020
      LDA 4030
      LDA 4040
      LDA 4050
      LDA 4060
      LDA 4070
      LDA 4080
      LDA 4090
      LDA 4100
      LDA 4110
      LDA 4120
      LDA 4130
      LDA 4140
      LDA 4150
      LDA 4160
      LDA 4170
      LDA 4180
      LDA 4190

```



```

C 460 GC TC 510
      NEWQ = NQ+L
      K = NEWQ+1
      IF (NEWQ.LE.NQ) GC TO 480
      R1 = A(NEWQ)/FLOAT(NEWQ)
C 470 DC 470 J=1,NY
      Y(K,J) = ER(J)*R1
C 480 CC CONTINUE
      -----
      IF THE STEP WAS OKAY, SCALE THE Y VARIABLES IN ACCORDANCE
      WITH THE NEW VALUE OF F. IF KFLAG < 0, HOWEVER, USE THE
      SAVED VALUES (IN SAVE AND YLSV). IN EITHER CASE, IF THE
      HAS CHANGED IT IS NECESSARY TO FIX CERTAIN PARAMETERS BY CALLING
      THE PROGRAM SEGMENT AT STATEMENT NUMBER 140.
      -----
C 490 ICCLB = NQ
      IF (NEWQ.EQ.NQ) GC TO 450
      NC = NEWQ
      ASSIGN 490 TO IRET
      GC TO 140
C 500 IF (KFLAG.GT.0) GC TO 500
      RACUM = RACUM*R
      GC TO 560
C 510 R = AMAX1(AMIN1(HMAX/H,R),HMIN/H)
      F = H*R
      IREVAL = 1
      ASSIGN 510 TO IRET
      GC TO 610
C 520 DC 520 I=1,M1
      YMAX(I) = AMAX1(ABS(Y(1,I)),YMAX(I))
      GC TO 170
C 530 -----
      THE ERROR TEST HAS NOW FAILED THREE TIMES, SO THE DERIVATIVES ARE
      IN BAD SHAPE. RETURN TO FIRST ORDER METHOD AND TRY AGAIN. CF
      CCLRSE, IF NQ = 1 ALREADY, THEN THERE IS NO PCPE AND WE EXIT WITH
      KFLAG = -4.
      -----
C 540 IF (NQ.EQ.1) GO TO 540
      NC = 1
      ICCLB = 1
      ASSIGN 570 TO IRET
      GC TO 140
      NCCLD = 1
      KFLAG = -4

```



```

C C C C C C
TC RESTART THE USER FIRST CALLS LDCARST TO RESTORE THE VALUES SAVED
BY LDCASAV, THEN RE-ENTERS LDCASUB WITH JSTART<0, AND WITH THE
OTHER PARAMETERS THE SAME AS RETURNED FROM THE LAST ENTRY TO
LDCASUB, PARTICULARLY THOSE ARRAYS MENTIONED ABOVE.
-----
ENTRY LDCASAV(SAV)
LCCPY S = 25
CALL COPYZ (SAV,A,LCCPY S)
CALL COPYZ (SAVE,Y,LCCPY Y)
CALL COPYZ (YLSV,YL,LCCPY L)
RETURN
-----
ENTRY LDCARST(SAV)
LCCPY R = 25
CALL COPYZ (A,SAV,LCCPY R)
RETURN
-----
C C C C C C
1 FCRMAT (2I5,I2,1P2E10.2,7E14.6/(32X,7E14.6))
2 FCRMAT (32X,1P7E14.6)
3 FCRMAT (1I,N=,I3, NL =,I3, RMSEPS =,1PES.2, TENC =,
1 ,ES.2, H =,ES.2//) H',8X,'T ' ,8X,'Y(1,*) AND YL(*)'//)
4 FCRMAT (1 NS NW Q
END
-----
LDA 5170
LDA 5180
LDA 5190
LDA 5200
LDA 5210
LDA 5220
LDA 5230
LDA 5240
LDA 5250
LDA 5260
LDA 5270
LDA 5280
LDA 5290
LDA 5300
LDA 5310
LDA 5320
LDA 5330
LDA 5340
LDA 5350
LDA 5360
LDA 5370
LDA 5380
LDA 5390
LDA 5400
LDA 5410
LDA 5420
LDA 5430

```



```

SLROUTINE SDESOL (Y, YL, T, TEND, NY, NL, M, JSKF, MAXDER, IPRT, F, HMIN,
1 FMAX, RMSEPS, W, BIGH12, BIGH3, BIGH5, BIGK, BICF, IBP, ITYPE, JA, JB, NAME,
2 NELCON, R, Z, YIT, BIGH4)
-----
SLROUTINE SDESOL IS A DRIVER ROUTINE FOR SLROUTINE LOADSUB.
ITS PURPOSE IS TO SET UP THE NECESSARY REFERENCES TO A LARGE
BLOCK OF AUXILIARY STORAGE, AND OBTAIN INITIAL VALUES OF
DERIVATIVES.
THE CALLING SEQUENCE FOR SDESOL IS

CALL SDESOL(Y, YL, T, TEND, NY, NL, M, JSKF, MAXDER, IPRT, F, HMIN, FMAX, RMSEPS, W)
WHERE THE PARAMETERS ARE DEFINED AS FOLLOWS.

Y - ARRAY DIMENSIONED (7, NY). THIS ARRAY CONTAINS THE
DEPENDENT VARIABLES AND THEIR SCALED DERIVATIVES.
Y(J+1, I) CONTAINS THE J-TH DERIVATIVE OF THE I-TH VARIABLE.
IABLE TIMES H*J/J-FACTORIAL, WHERE H IS THE CURRENT
STEP SIZE. ON FIRST ENTRY THE CALLER SUPPLIES THE
INITIAL VALUES OF EACH VARIABLE IN Y(I, I). ON SUB-
SEQUENT ENTRIES IT IS ASSUMED THE ARRAY HAS NOT
BEEN CHANGED. TO INTERPOLATE TO NON-MESH POINTS,
THESE VALUES CAN BE USED AS FOLLOWS. IF H IS THE
CURRENT STEP SIZE AND VALUES AT TIME T+H ARE
NEEDED, LET S = E/H AND THEN

I-TH VARIABLE AT T+E IS SUM Y(J+1, I)*S**J
J=0

THE VALUE OF JS IS OBTAINED IN THE CALLING PROGRAM
BY JS = IABS(JSKF/10)
- ARRAY OF NL VARIABLES WHICH APPEAR LINEARLY.
- CURRENT VALUE OF THE INDEPENDENT VARIABLE (TIME)
- END TIME OF DIFFERENTIAL EQUATIONS AND NONLINEAR
VARIABLES.
- NUMBER OF LINEAR VARIABLES INCLUDED IN THE ERROR TEST
- NUMBER OF VARIABLES USED BOTH ON INPUT AND OUTPUT
- AN INDICATOR USED BOTH ON INPUT AND OUTPUT
ON INPUT, JSKF = -1 INDICATES A RESTART CALL TO
SDESOL. JSKF = 0 INDICATES AN INITIAL CALL TO
SDESOL. JSKF > 0 INDICATES A CONTINUATION OF THE
PREVIOUS CALL TO SDESOL. JSKF < -1 MAY HAVE RETURNS
FROM THE USER NEGLECTING TO TEST FOR ERROR, JSKF < -1
FROM SDESOL. BECAUSE OF THIS POSSIBILITY, JSKF < -1

```



```

RESULTS IN TERMINATION OF THE RUN WITH THE
APPROPRIATE COMMENT.
ON OUTPUT, JSKF CONSISTS OF TWO DIGITS AND SIGN,
+ OR - CF. Q IS THE ORDER OF THE FORMULA CURRENTLY
BEING USED. P INDICATES THE TYPE OF RETURN, AS
FOLLOWS.
JSKF > 0, P = 1 IS THE NORMAL RETURN
JSKF < 0, P IS AN ERROR RETURN, WITH THE FOLLOWING
MEANINGS.
P = 1 ERROR TEST FAILURE FOR F > HMIN
P = 3 CORRECTOR FAILED TO CONVERGE FOR F > HMIN
P = 4 CORRECTOR FAILED TO CONVERGE FOR FIRST
ORDER METHOD FROM SUBROUTINE NLITSL
P = 5 ERROR RETURN FROM SUBROUTINE DERRVAL
P = 6 ERROR DERIVATIVE THAT SHOULD BE USED IN
MAXCER - MAXIMUM ORDER OF DIFFERENTIAL IN
METHOD. IT MUST BE NO GREATER THAN SIX.
IPRT - INTERNAL PRINT CONTROL INDICATOR FOR LEASUB.
IPRT = 0 NO PRINT
IPRT > 0 PRINT COUNTERS, STEPSIZE, CURRENT TIMES
AND VALUES OF DEPENDENT VARIABLES AT
EACH STEP.
- CURRENT STEPSIZE. AN INITIAL VALUE MUST BE SUPPLIED
BUT NEED NOT BE THE ONE WHICH MUST BE USED, SINCE THE
SUBROUTINE WILL CHOOSE A SMALLER ONE IF NECESSARY TO
KEEP THE ERROR PER STEP SMALLER THAN THE SPECIFIED
VALUE. IT IS BETTER TO UNDERESTIMATE THE INITIAL
STEPSIZE THAN TO OVERESTIMATE IT. THE STEPSIZE IS
NORMALLY NOT CHANGED BY THE USER.
- MINIMUM STEPSIZE ALLOWED
- MAXIMUM STEPSIZE ALLOWED
- THE ERROR TEST CONSTANT ESTIMATES, ER(I), DIVIDED BY
YMAX(I) = (MAXIMUM ERROR TO CURRENT SIZE AND/OR THE ORDER
LESS THAN EPS. THE ABOVE THIS. MUST BE AT LEAST 13*NY + 5*NL
ARE VARIED TO ACHIEVE THIS. MUST BE AT LEAST 13*NY + 5*NL
SCRATCH STORAGE ARRAY. REQUIRED FOR STORAGE OF THE
LOCATIONS, PLUS THE DESCRIPTION OF SUBROUTINE JACMAT).
MATRIX PW (SEE DESCRIPTION OF SUBROUTINE JACMAT).
THE STORAGE OF PW WILL NORMALLY REQUIRE MORE THAN
N**2 + 2*N LOCATIONS, AND IF COMPACT STORAGE TECH-
NIQUES ARE USED, CAN BE MUCH FEWER.
-----
INTEGER*2 IBP, ITYPE, JA, JB, NAME, NELCCN
DIMENSION Y(7,1), YL(1), W(1)
DIMENSION BIGH12(1), BIGH3(1), BIGH5(1)
DIMENSION BIGH4(1)

```



```

C DIMENSION BIGK(1)
C DIMENSION BIGF(1)
C DIMENSION IBP(1), ITYPE(1), NAME(1)
C DIMENSION JA(1), JB(1), Z(1)
C DIMENSION NELCON(11,1)
C DIMENSION R(1), YIT(1)
C IF (JSKF.GT.0) GO TO 12C
C IF (JSKF.LT.-1) GO TO 140
C N = NY+NL
C IF (JSKF.LT.0) GO TO 110
C IF THIS IS THE FIRST ENTRY, OBTAIN VALUES OF THE DERIVATIVES.
C CALL Derval(Y,YL,T,N,NY,W,KRETR,BIGH12,BIGF3,BIGH4,EIGH5,EICK,
1 EIGF,IIP,ITYPE,JA,JB,NAME,NELCCN,R,Z,YIT)
C IF (KRETR.NE.0) GO TO 130
C NCH SET UP STORAGE BLOCKS IN THE W ARRAY. THIS NEEDS TO BE DONE
C ONLY INITIALLY AND CN RESTARTS.
C THE ARRAY SAVE STARTS AT LCCATION 1 NSVL W ARRAY
C THE ARRAY YLSV STARTS AT LCCATION NSVMX W ARRAY
C THE ARRAY YMAX STARTS AT LCCATION NYMAX W ARRAY
C THE ARRAY ERV STARTS AT LCCATION NESV W ARRAY
C THE ARRAY F1 STARTS AT LCCATION NF1 W ARRAY
C THE ARRAY DY STARTS AT LCCATION NCY W ARRAY
C THE MATRIX PW STARTS AT LOCATION NPW W ARRAY
110 NSVL = 7*NY+1
C NYMAX = NSVL+NL
C NER = NYMAX+NY
C NESV = NER+NY
C NF1 = NESV+NY
C NCY = NF1+N
C NPW = NCY+N
C JSKF = JSKF
C CALL LCASUB (Y,YL,T,TEND,N,NY,M,JS,KF,MAXDER,IPRT,F,H,FMIN,FMAX,
1 RMSEPS,W(NSVL),W(NYMAX),W(NER),W(NESV),W(NF1),W(NFY),W(NPW),
2 BIGH12,BIGH3,BIGH4,BIGH5,BIGK,BIGF,IBP,ITYPE,JA,JB,NAME,NELCCN,
3 R,Z,YIT)
120 CCDE JSKF ON RETURN FROM LDASUB
C C C C
C JSKF = ISIGN(JS*10+IABS(KF),KF)
C RETURN
C JSKF = -6
C RETURN
130

```


14C PRINT 1, JSKF
STOP

C
C

1 FCRMAT ('OIT IS AN ERRGR TO ENTER SDESOL WITH JSKF = ', I10//
1 ' RUN HAS BEEN TERMINATED.')
END

SDE 1310
SDE 1320
SDE 1330
SDE 1340
SDE 1350
SDE 1360
SDE 1370

C	SLRCUTINE COPYZ(S,Y,L)	COP	10
C	DIMENSION S(1),Y(1)	COP	20
C	-----	COP	30
C	THIS SUBROUTINE COPIES THE ARRAY Y, OF LENGTH L, INTO THE ARRAY S	COP	40
C	-----	COP	50
	IF(L.LE.0)RETURN	COP	60
	DC 100 J=1,L	COP	70
	S(J) = Y(J)	COP	80
	RETURN	COP	90
	END	COP	100
		COP	110
		COP	120


```

C      DIMENSION Y(7,1), YL(1), W(1)
C      DC 100 I=1,NY
C      W(2*N+1) = AMAX1(ABS(Y(1,1)),1.)
C      100 Y(3,1) = 0.
C      HINV = 16.**20
C      KERET = 0
C      EPS2 = NY/1.E5
C      EPS = SQRT(EPS2)
C      DC 140 IT=1,10
C      DC 110 I=1,NY
C      110 Y(2,I) = Y(3,I)/HINV
C      CALL DIFFUN (Y,YL,I,HINV,W,BIGH12,BIGH3,BIGH4,BIGH5,EIGK,EIGF,
C      1 IBP,ITYPE,JA,JB,NAME,NELCON,R,Z,YIT)
C      CALL JACMAT (Y,YL,T,HINV,-1.,NY,NY,EPS,W,W(N+1),W(3*N+1),EIGF12,
C      1 EIGF3,BIGH4,BIGH5,BIGK,IBP,ITYPE,JA,JB,NAME,NELCCN,R,Z)
C      NEWPW = 1
C      DC 120 I=1,NY
C      120 W(1) = W(I)*HINV
C      CALL NUTSL (W(3*N+1),W,W(N+1),NY,NY,EPS,W(2*N+1),NEWPW,KRET,
C      1 BIGH3,ITYPE,JA,JB,NAME,NELCCN)
C      IF (KRET.NE.0) GO TO 170
C      ER = 0.
C      DC 130 I=1,NY
C      Y(3,I) = Y(3,I)-W(N+1)
C      W(2*N+1) = AMAX1(ABS(Y(3,1)),1.)
C      130 ER = ER+(W(N+1)/AMAX1(ABS(Y(3,1)),1.))**2
C      IF (ER.LT.EPS2) GO TO 150
C      CCNTINUE
C      GC TO 170
C      DC 150 CC 160 I=1,NY
C      160 Y(2,I) = Y(3,I)
C      RETURN
C      170 KERET = 1
C      RETURN
C      END

```


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